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DESIGN AND FABRICATION OF A HIGH-PERFORMANCE
BRAYTON-CYCLE COMPRESSOR RESEARCH PACKAGE

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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FINAL REPORT

**DESIGN AND FABRICATION OF A HIGH-PERFORMANCE
BRAYTON-CYCLE COMPRESSOR RESEARCH PACKAGE**

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

May 1965

CONTRACT NAS 3-2778

Technical Management
NASA Lewis Research Center
Cleveland, Ohio
Space Power System Division
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**AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
Phoenix, Arizona**



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NASA CONTRACTOR REPORT

DESIGN AND DEVELOPMENT OF A HIGH-PERFORMANCE BRAYTON-CYCLE COMPRESSOR RESEARCH PACKAGE

AiResearch Manufacturing Company of Arizona

ABSTRACT

In this development program, advanced aerodynamic design procedures were used to design a high-efficiency radial compressor for operation on monatomic gases. The compressor research package consists of a 6-inch-diameter compressor wheel and shaft mounted on ball bearings and the associated mounting hardware. Following development testing, the final configuration was defined, and when the unit was tested on argon, efficiencies in excess of 0.80 total-to-total were obtained. Additional testing is to be accomplished by the NASA.



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SUMMARY

The NASA-Lewis Research Center is presently engaged in an investigation of the turbomachinery components of Brayton-cycle space power systems using solar or nuclear energy as the heat source and an inert gas as the working fluid. Under Contract NAS3-2778, three pieces of hardware are to be designed and developed. The hardware will be used by the NASA for the Brayton-cycle investigation and includes the following:

Compressor and Turbine Research Packages - The two research packages each include a cold gas model of a high-performance radial wheel and a suitable set of running gear with oil-lubricated bearings. Both research packages are to be used to evaluate component aerodynamic performance.

Gas Generator - This unit combines the turbine and compressor of the two research packages into a single hot unit with the running gear, including gas-lubricated bearings. The gas generator will be used to evaluate the Brayton-cycle turbomachinery in a complete system ground test loop.

This report describes the selection of the system design point and the design, fabrication, inspection, and testing of the compressor research package. The compressor design points for the research package is:

Working fluid	Argon
Mass flow rate, $\frac{\text{lb}}{\text{m} \cdot \text{sec}}$	0.621

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Compressor inlet temperature, T_1 - $^{\circ}\text{R}$ 520
Compressor inlet pressure, P_1 - psia 6.0
Compressor pressure ratio, r_c 2.30
Design operating corrected speed, $N/\sqrt{\theta}$ -rpm 37,900

$$\theta = \frac{T_1}{518.7}$$

The research package consists of a 6-inch-diameter compressor wheel and shaft mounted on ball bearings and the associated mounting hardware.

Development testing of the compressor consisted of running the uncut development impeller with three diffusers-- a nominal diffuser, a negative 3-degree diffuser, and a positive 3-degree diffuser. After mapping with the three diffusers, the impeller was cut back and run with the nominal and the positive 3-degree diffusers. After the final impeller configuration was determined, acceptance testing of the shipping units was accomplished. The first shipping unit was run for three speed lines and the second shipping unit was operated at design speed for 1.2 hours.

At the conclusion of the compressor research package development two conclusions are evident:

- (a) A high-efficiency radial compressor can be designed to operate on inert gas.
- (b) At the design point, the efficiency can be accurately predicted (predicted $\eta = 0.798$, test $\eta = 0.80$).



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DESIGN AND DEVELOPMENT OF A HIGH-PERFORMANCE BRAYTON-CYCLE COMPRESSOR RESEARCH PACKAGE

1.0 INTRODUCTION

This report describes the design, fabrication, inspection, and testing of a Brayton-cycle compressor research package that will be used to evaluate the aerodynamic performance of Brayton-cycle compressors. The compressor was designed for high-efficiency, low-Reynolds-number operation with an inert gas used as the working fluid. With the recent development of Brayton-cycle space-power systems, high-efficiency, low-Reynolds-number type turbomachinery has only recently been required and, therefore, no prior work on turbomachinery of this type has been accomplished.

The compressor research package consists of a 6-inch-diameter radial compressor wheel and shaft mounted on ball bearings with the associated mounting hardware. Notable features of the compressor research package include the advanced aerodynamic design procedures and the utilization of extremely thin blade sections. The same design goals have been used in a 6-inch-diameter turbine research package developed for NASA under the same contract and a 3.2-inch-diameter compressor developed for the Air Force [Contract AF33(657)-11721].

With the development of the compressor research package, the feasibility to design high-efficiency radial compressors for operation on inert gas has been proven. Test results on the compressor research package when tested on argon indicated efficiencies in excess of 0.81 total-to-total.



2.0 SELECTION OF DESIGN CONDITIONS

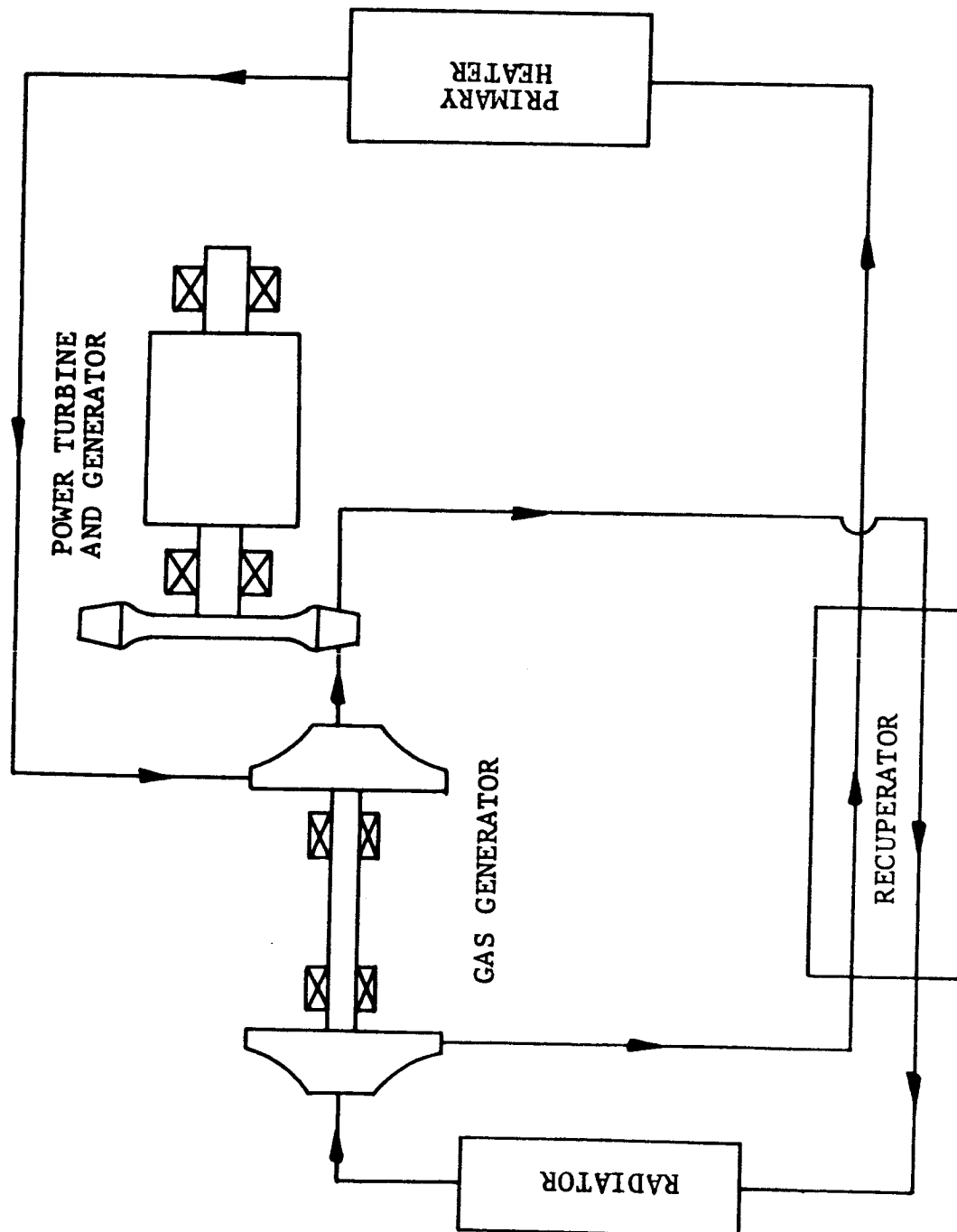
Figure 1 shows a schematic of the NASA Brayton-cycle space power system. Contract NAS 3-2778 calls for the development of the gas generator package, a turbine research test package, and a compressor research test package, with the same turbomachinery used in the test packages as in the gas generator. As specified by the contract, the gas generator and research test packages have identical design conditions when corrected mass-flow rates are compared. Table 1 presents a summary of the design conditions as specified by the contract. In addition to the conditions listed in Table 1, the most important remaining system variables include:

- (a) Recuperator effectiveness, E_R
- (b) Pressure loss parameter, r_t/r_c
- (c) Shaft speed, N
- (d) Compressor specific speed, N_{S_c}

As the recuperator effectiveness is increased, the cycle thermal efficiency and mass-flow rate increase and the optimum compressor pressure is reduced. The low compressor pressure ratio, in turn, leads to higher compressor efficiency. Moreover, at low power levels, the increased mass-flow rate is beneficial to the turbomachinery as a result of the higher attendant Reynolds number in the turbine and compressor. Since increased recuperator effectiveness lowers both the radiator inlet temperature and



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NASA BRAYTON-CYCLE-POWER SYSTEM

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FIGURE 1



TABLE 1

DESIGN PARAMETERS FIXED BY
CONTRACT NAS 3-2778

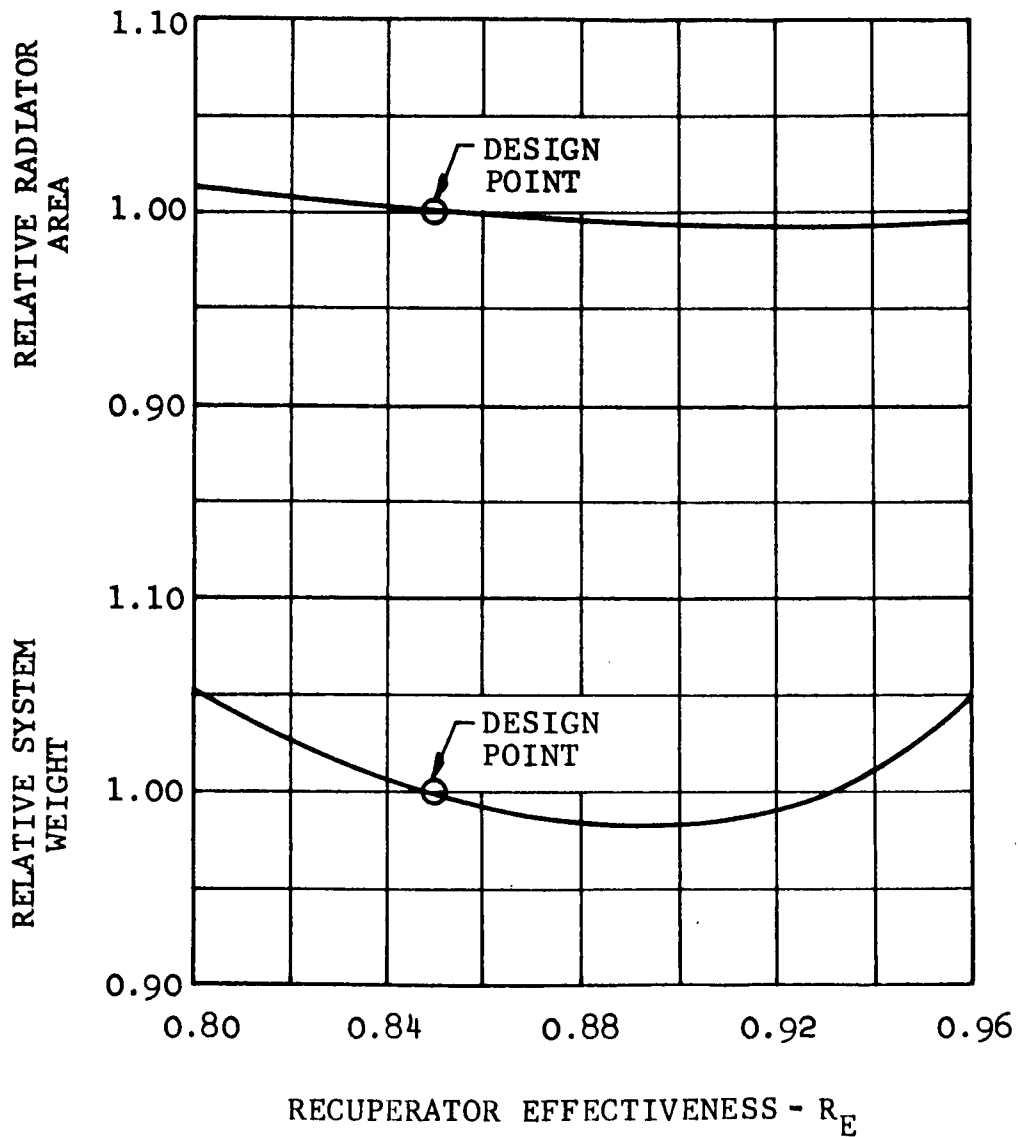
	<u>Turbine Package</u>	<u>Compressor Package</u>	<u>Gas Generator</u>
Working fluid	Argon	Argon	Argon
Mass flow rate, $\frac{\text{m}^{\circ}}{\text{sec.}}$ - lbs	1.184	0.621	0.611
Turbine inlet temperature, $T_3, ^{\circ}\text{R}$	520	-	1950
Turbine inlet pressure, P_3, psia	13.2	-	13.2
Turbine pressure ratio, r_{t_1}	1.56	-	1.56
Compressor inlet temperature, $T_1, ^{\circ}\text{R}$	-	520	536
Compressor inlet pressure, P_1, psia	-	6.0	6.0
Compressor pressure ratio, r_c	-	2.30	2.30
Corrected mass flow rate:			
$W\sqrt{\theta}/\delta$ turbine, lbs per sec.	1.3185	-	1.3185
$W\sqrt{\theta}/\delta$ compressor, lbs per sec.	-	1.5214	1.5217



the heat load, the radiator area is not strongly affected. Referring to Figure 2, the optimum recuperator effectiveness is established by a weight tradeoff between the recuperator and other components. Although a recuperator effectiveness of 0.9 appears to be optimum, a value of 0.85 has been chosen in view of the mass flow rate and compressor pressure ratio being specified.

The effect of the cycle pressure-loss parameter on system performance is shown in Figure 3. Although a value of 0.95 would be near optimum for the pressure-loss parameter, the value chosen was 0.90. This value allows increased flexibility, since the heat exchangers and manifolds would have to be designed for extremely low pressure drops if a pressure-loss parameter of 0.95 were utilized.

With the design point conditions listed in Table 1, a recuperator effectiveness of 0.85, and a pressure-loss parameter of 0.90, a design-point study was conducted to establish the gas generator thermodynamic and aerodynamic operating conditions. Figures 4 and 5 illustrate the variation of wheel diameters, component and cycle efficiencies, component specific speeds, and turbine pressure ratio over the range of shaft speeds with the turbine and compressor matched. (A list of the symbols used throughout this report can be found on page 10.) Wheel diameters of 6 inches occur for both the turbine and the compressor at a shaft rotational speed of 38,500 rpm with reasonable adiabatic efficiencies for the units. With this size wheels, manufacturing tolerances can be readily maintained to provide aerodynamic passages with surfaces that are hydraulically smooth.



VARIATION OF SYSTEM WEIGHT AND RADIATOR AREA
VERSUS RECUPERATOR EFFECTIVENESS

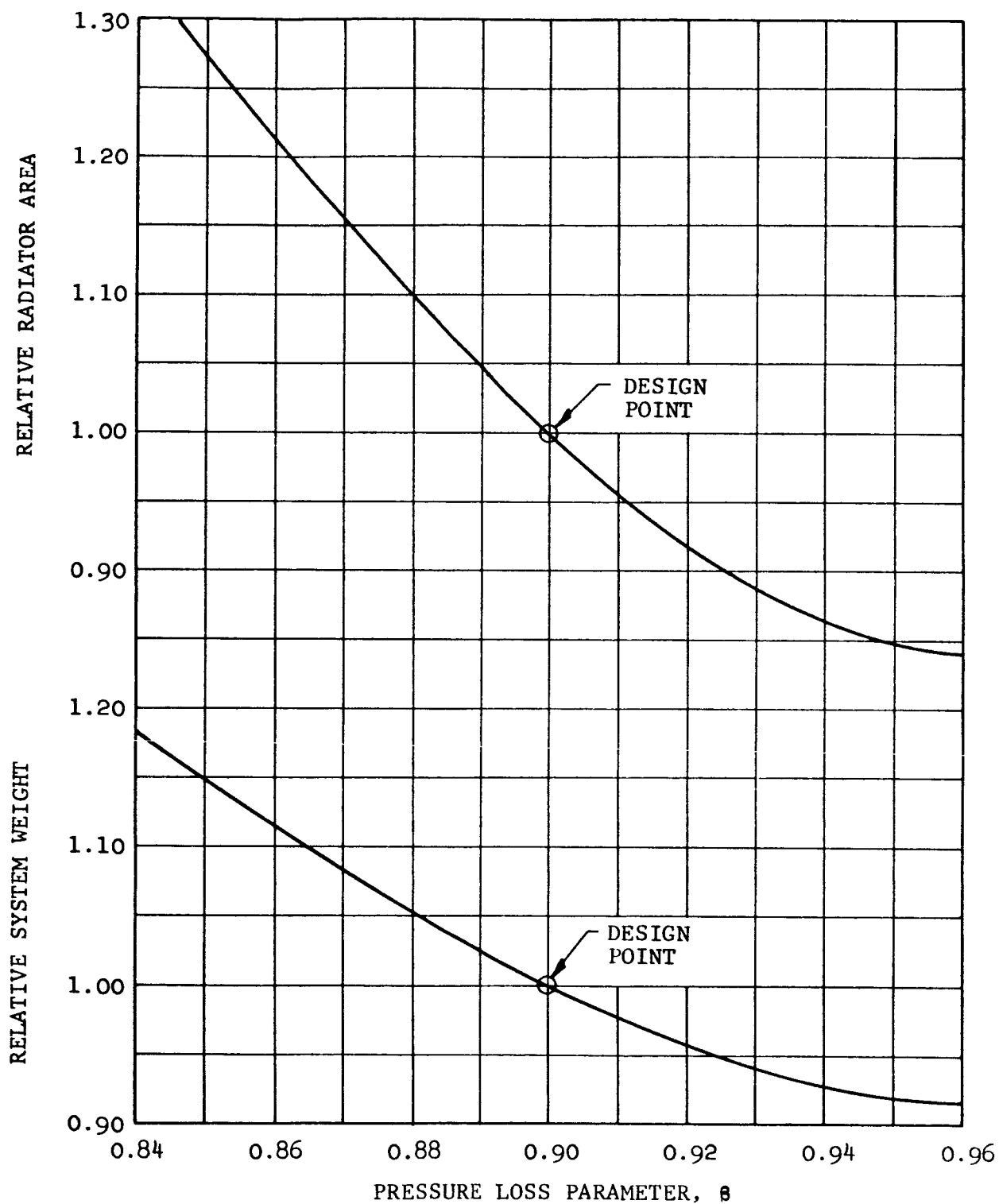
FIGURE 2



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VARIATION OF SYSTEM WEIGHT AND RADIATOR AREA
VERSUS PRESSURE LOSS PARAMETER

FIGURE 3

APS-5109-R

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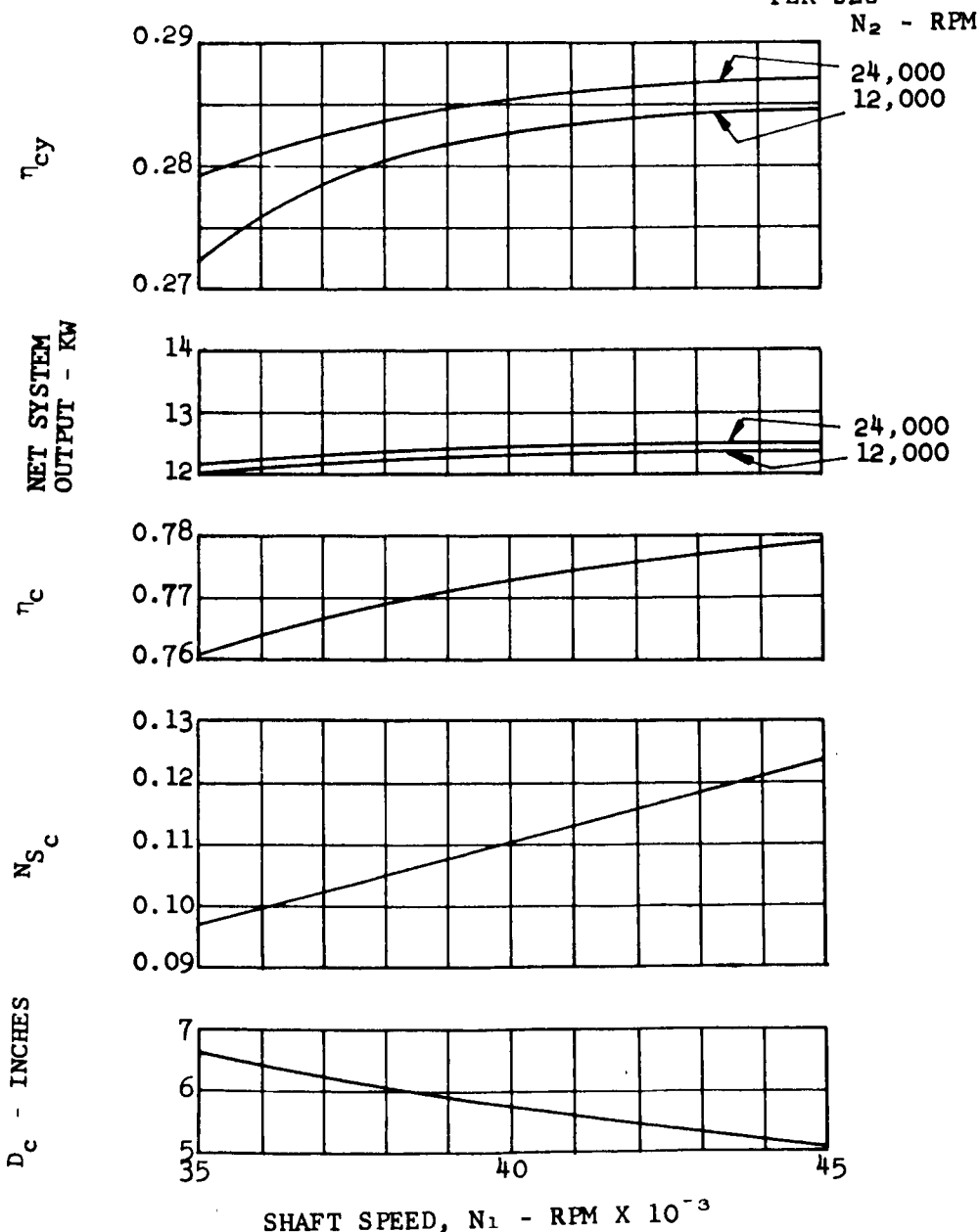
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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^{\circ}\text{R}$
TURBINE INLET TEMPERATURE, $T_3 = 1950^{\circ}\text{R}$
RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
WORKING FLUID $= \text{ARGON}$

$\frac{\text{TURBINE PRESSURE RATIO}}{\text{COMPRESSOR PRESSURE RATIO}}, \beta = 0.90$

COMPRESSOR PRESSURE RATIO, $r_c = 2.30$
COMPRESSOR INLET PRESSURE, $P_1 = 6.0 \text{ PSIA}$
MASS FLOW RATE, $\dot{m} = 0.611 \text{ LBS PER SEC}$

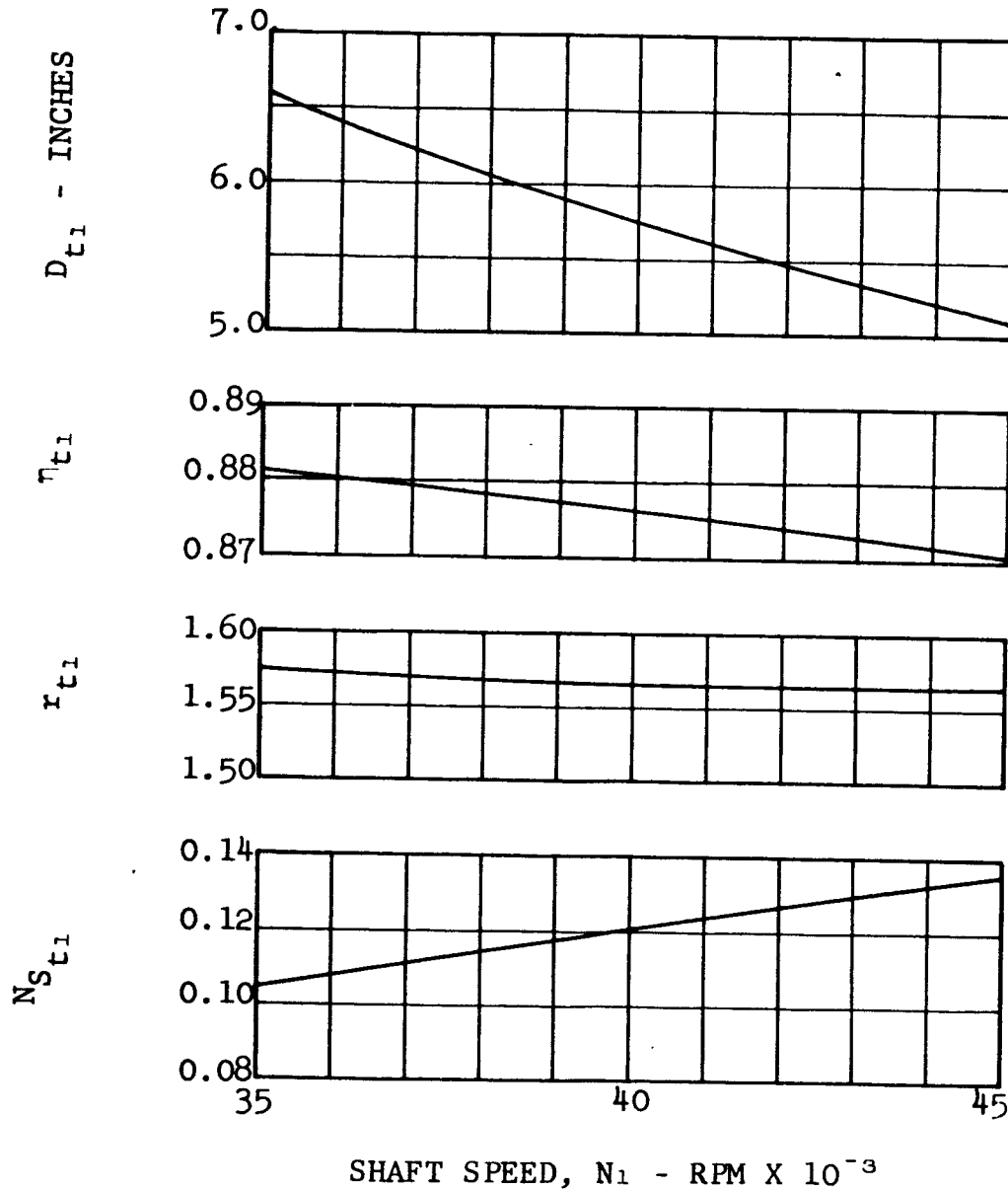


COMPRESSOR DESIGN
NASA DESIGN POINT
FIGURE 4



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COMPRESSOR INLET TEMPERATURE, T_1 = 536°R
TURBINE INLET TEMPERATURE, T_3 = 1950°R
RECUPERATOR EFFECTIVENESS, E_R = 0.85
WORKING FLUID = ARGON
TURBINE PRESSURE RATIO
COMPRESSOR PRESSURE RATIO, β = 0.90
COMPRESSOR PRESSURE RATIO, r_c = 2.30
COMPRESSOR INLET PRESSURE, P_1 = 6.0 PSIA
MASS FLOW RATE, \dot{m} = 0.611 LBS
PER SEC



FIRST STAGE TURBINE
NASA DESIGN POINT
FIGURE 5



LIST OF SYMBOLS

g	= conversion factor = 32.2 ft lb per lb sec. ²
\dot{m}	= molal gas flow rate, lbs mol per sec.
r_c	= compressor pressure ratio
r_{t1}	= gas-generator turbine pressure ratio
r_{t2}	= power-turbine pressure ratio
D_c	= compressor-wheel diameter, inches
D_{t1}	= gas-generator turbine-wheel diameter, inches
E_R	= recuperator effectiveness
M	= molecular weight, lbs per lb mol
N_1	= gas-generator shaft speed, rpm
N_2	= power-turbine shaft speed, rpm
N_{S_c}	= compressor shaft speed
$N_{S_{t1}}$	= gas-generator turbine specific speed
$N_{S_{t2}}$	= power-turbine specific speed
P_1	= compressor inlet pressure, lbs per sq ft
P_3	= gas-generator turbine inlet pressure, lbs per sq ft
R	= universal gas constant = 1545 ft-lbs per lb-mol °R
T_1	= compressor inlet temperature, °R
T_3	= gas-generator turbine-inlet temperature, °R
θ	= $\frac{\text{turbine pressure ratio}}{\text{compressor pressure ratio}} = \frac{r_{t1} \times r_{t2}}{r_c}$



LIST OF SYMBOLS (Contd.)

γ = ratio of gas specific heats = 1.667 for monatomic gases

θ = $(\gamma - 1)/\gamma = 0.4$ for monatomic gases

η_{cy} = $\frac{\text{power-turbine shaft power output}}{\text{gas-cycle input rate}}$

η_c = compressor adiabatic efficiency

η_{t1} = gas-generator turbine adiabatic efficiency

η_{t2} = power-turbine adiabatic efficiency

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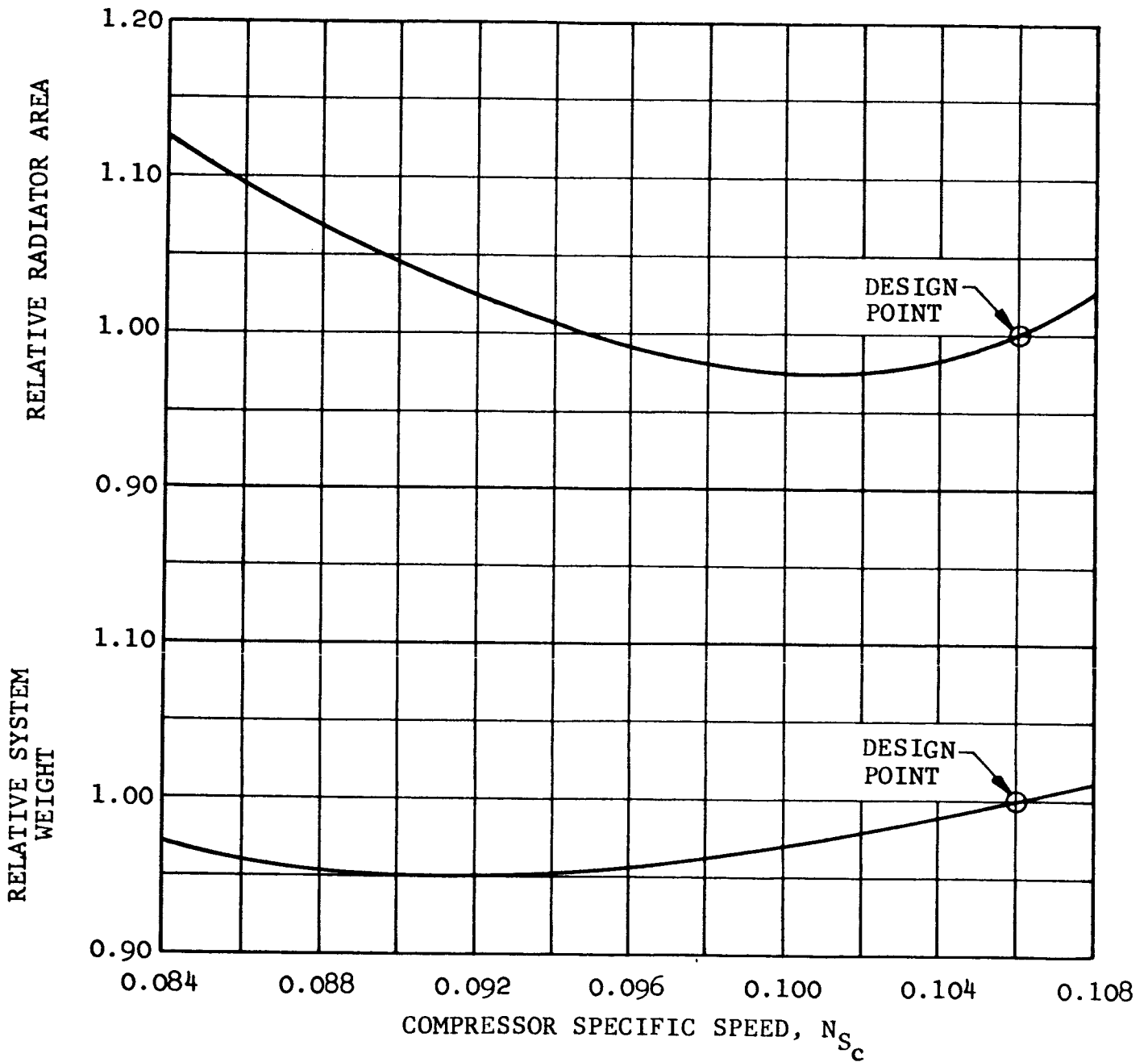
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A low compressor specific speed results at the shaft speed of 38,500 rpm, and the cycle efficiency is not seriously reduced from that which would be obtained at higher shaft speeds and smaller wheel diameters. The variation of system weight and radiator area versus compressor specific speed is shown in Figure 6. A compressor specific speed of 0.09 is near optimum based on the consideration of system weight; based on the consideration of both system weight and radiator area, the optimum specific speed is approximately 0.096. From the expression for compressor specific speed

$$N_{S_c} = \frac{N}{60} \left(\frac{1}{RT_1} \right)^{1/4} \left(\frac{\dot{m}}{P_1} \right)^{1/2} \left[\frac{(\gamma - 1)M}{\gamma g(r_c^{\frac{\gamma}{\gamma-1}} - 1)} \right]^{3/4}$$

it can be seen that the compressor specific speed is a function only of shaft speed, since the remaining variables are fixed by the contract. Therefore, the shaft speed selected is nearly optimum for the specified conditions. A lower shaft speed would result in a more desirable compressor specific speed, but the wheel sizes would be unnecessarily large and the slow speed could result in bearing problems.

Additional computer runs were made with variable cycle pressure level, mass flow rate, and compressor pressure ratio (variable compressor specific speed) over a gas-generator speed range of 35,000 to 65,000 rpm with a free-turbine shaft speed of 24,000 rpm. These calculations were made to supply added insight into the design-point examination and are summarized in Figures 7 through 11.



VARIATION OF SYSTEM WEIGHT AND RADIATOR AREA
VERSUS COMPRESSOR SPECIFIC SPEED

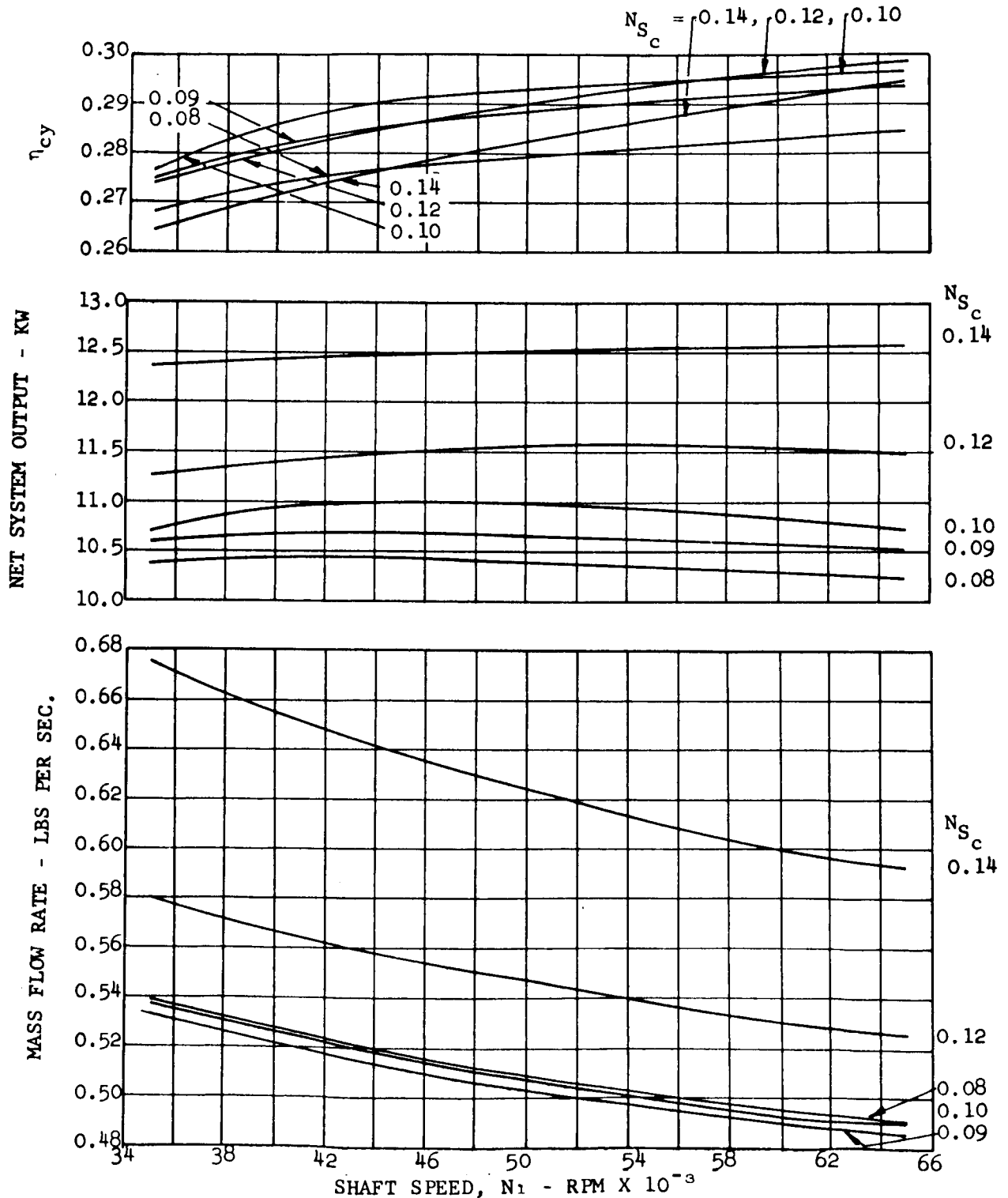
FIGURE 6



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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^{\circ}\text{R}$
 TURBINE INLET TEMPERATURE, $T_3 = 1950^{\circ}\text{R}$
 RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
 WORKING FLUID = ARGON
 TURBINE PRESSURE RATIO = 0.90
 COMPRESSOR PRESSURE RATIO, $\beta = 0.90$
 FREE TURBINE ROTOR SPEED, $N_2 = 24,000 \text{ RPM}$



OPTIMIZED NASA SYSTEM

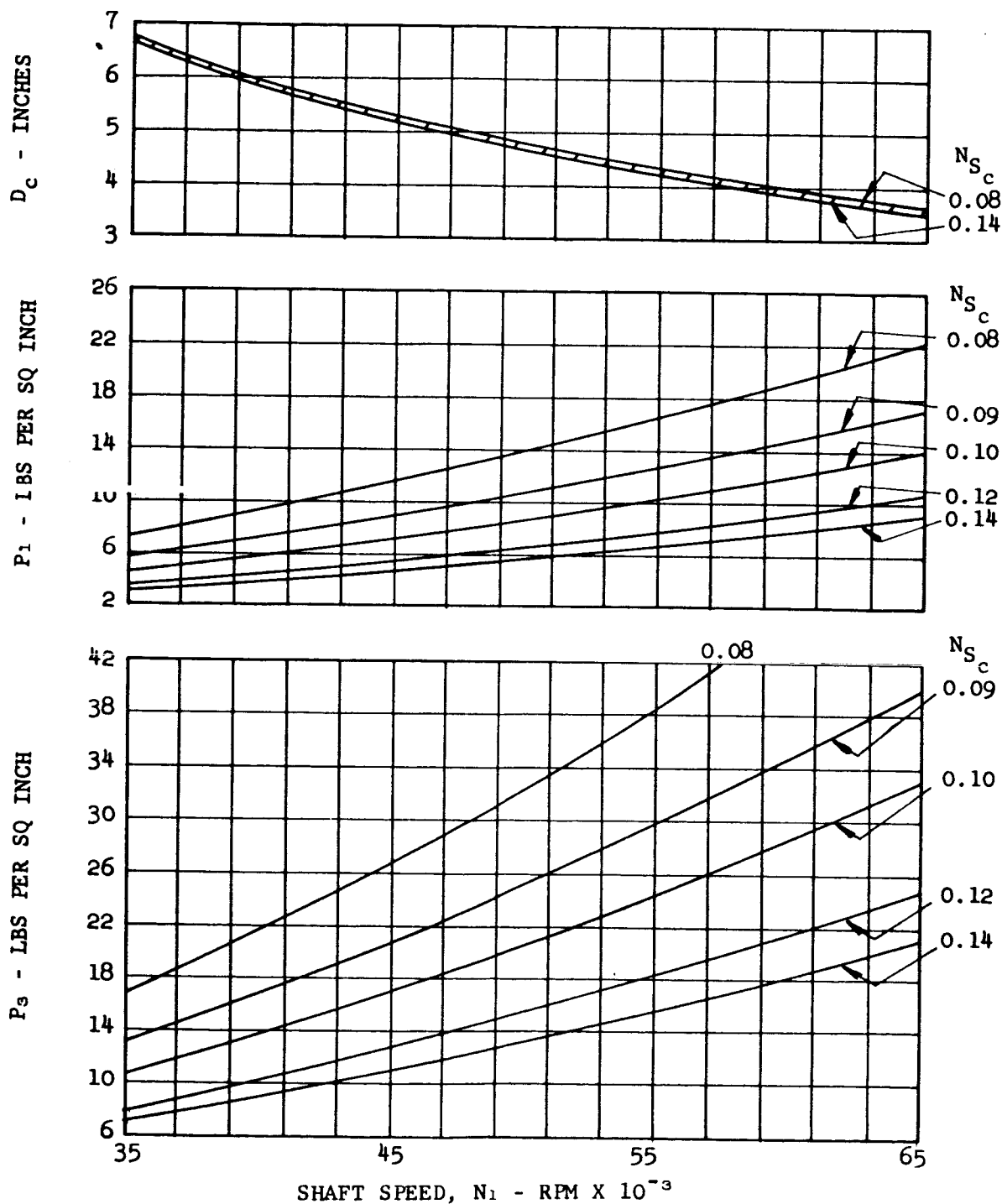


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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^\circ\text{R}$
TURBINE INLET TEMPERATURE, $T_3 = 1950^\circ\text{R}$
RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
WORKING FLUID = ARGON

TURBINE PRESSURE RATIO
COMPRESSOR PRESSURE RATIO, $\beta = 0.90$



COMPRESSOR DIAMETER, COMPRESSOR
INLET PRESSURE AND TURBINE INLET
PRESSURE FOR OPTIMIZED NASA SYSTEM

FIGURE 8

APS-5109-R

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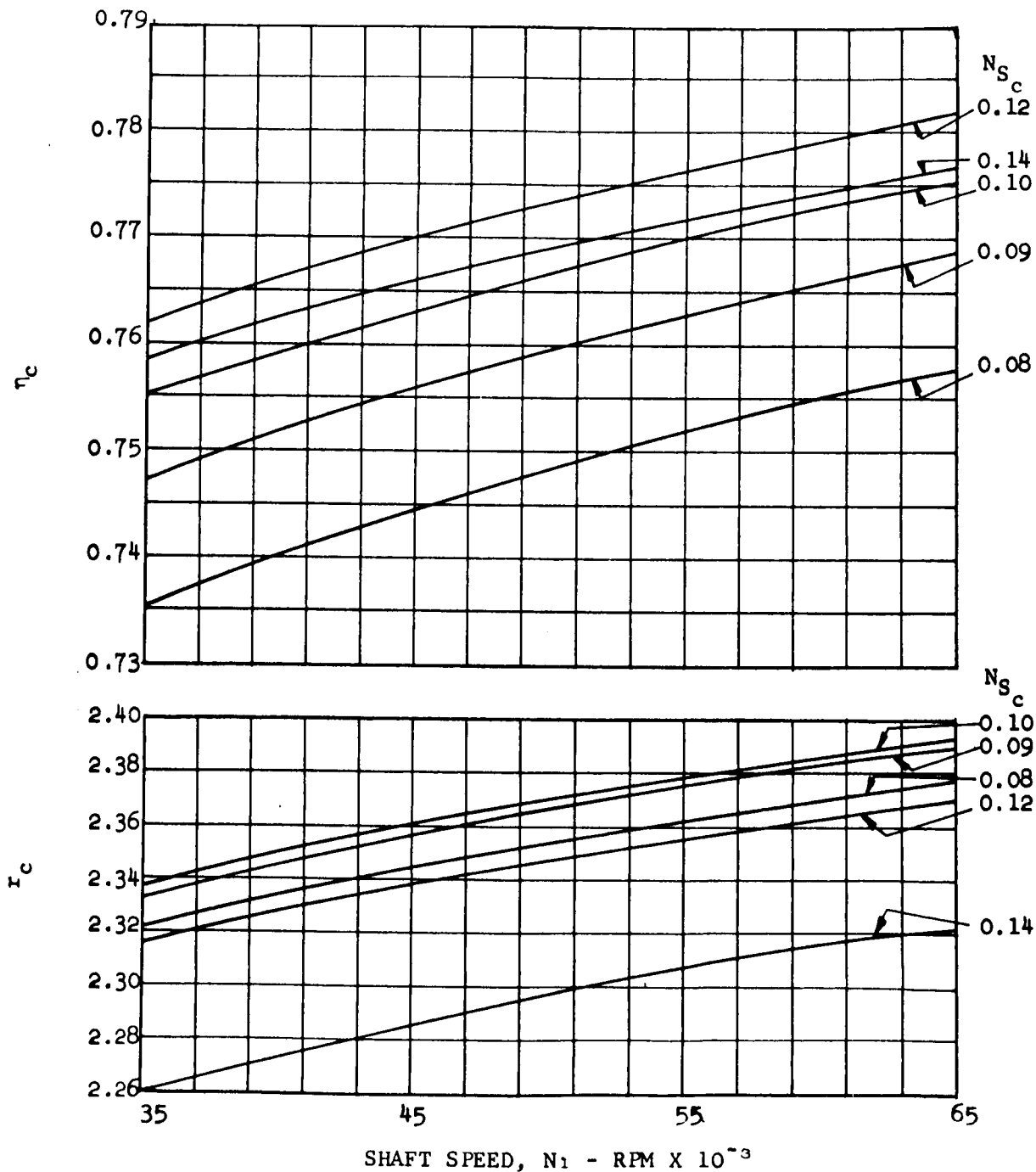
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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^\circ\text{R}$
TURBINE INLET TEMPERATURE, $T_3 = 1950^\circ\text{R}$
RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
WORKING FLUID = ARGON
TURBINE PRESSURE RATIO = 0.90
COMPRESSOR PRESSURE RATIO, $\beta = 0.90$



COMPRESSOR EFFICIENCY AND
COMPRESSOR PRESSURE RATIO FOR
OPTIMIZED NASA SYSTEM

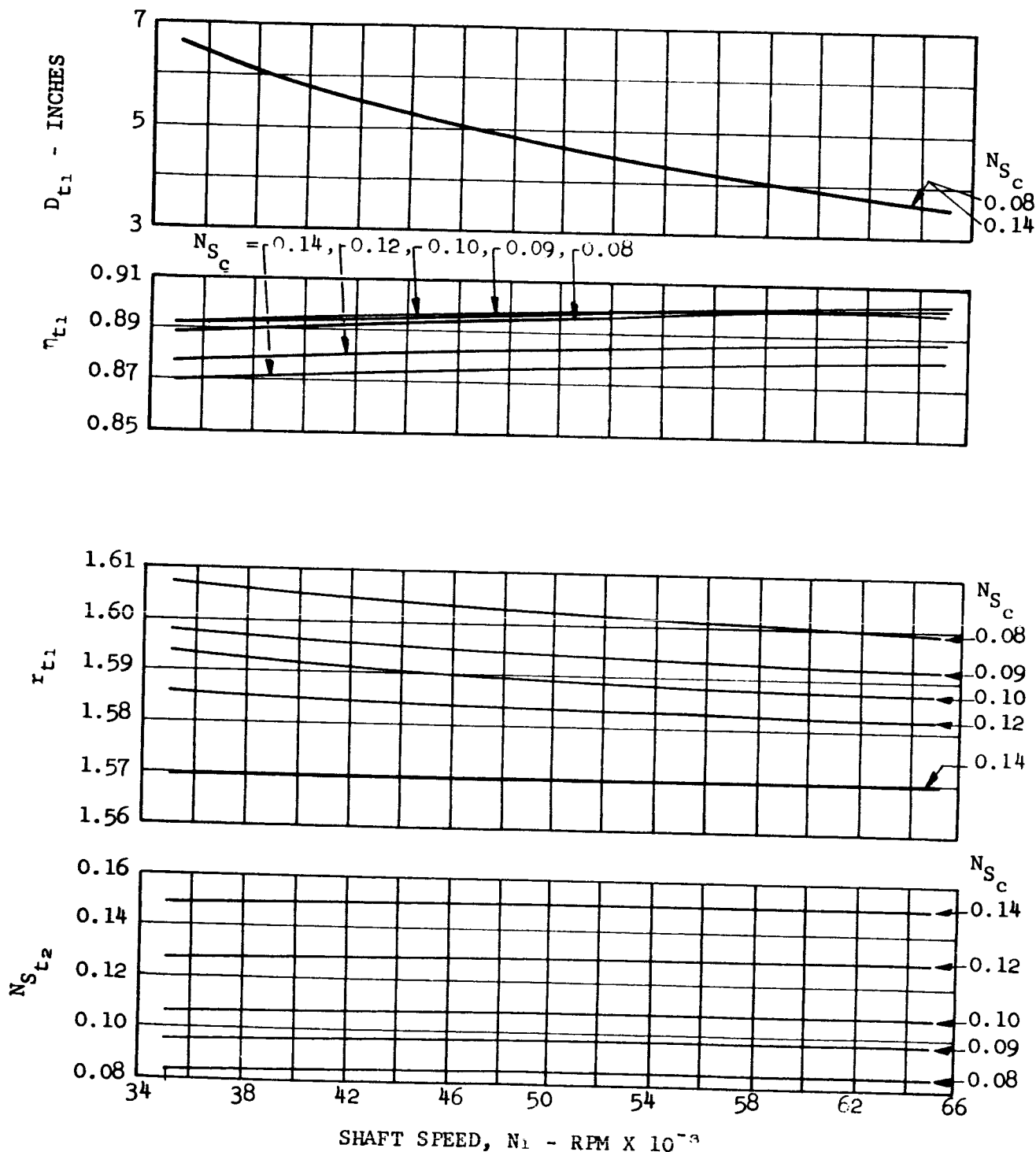
FIGURE 9

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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^{\circ}\text{R}$
TURBINE INLET TEMPERATURE, $T_3 = 1950^{\circ}\text{R}$
RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
WORKING FLUID = ARGON
TURBINE PRESSURE RATIO
COMPRESSOR PRESSURE RATIO, $\beta = 0.90$
FREE TURBINE ROTOR SPEED, $N_2 = 24,000 \text{ RPM}$



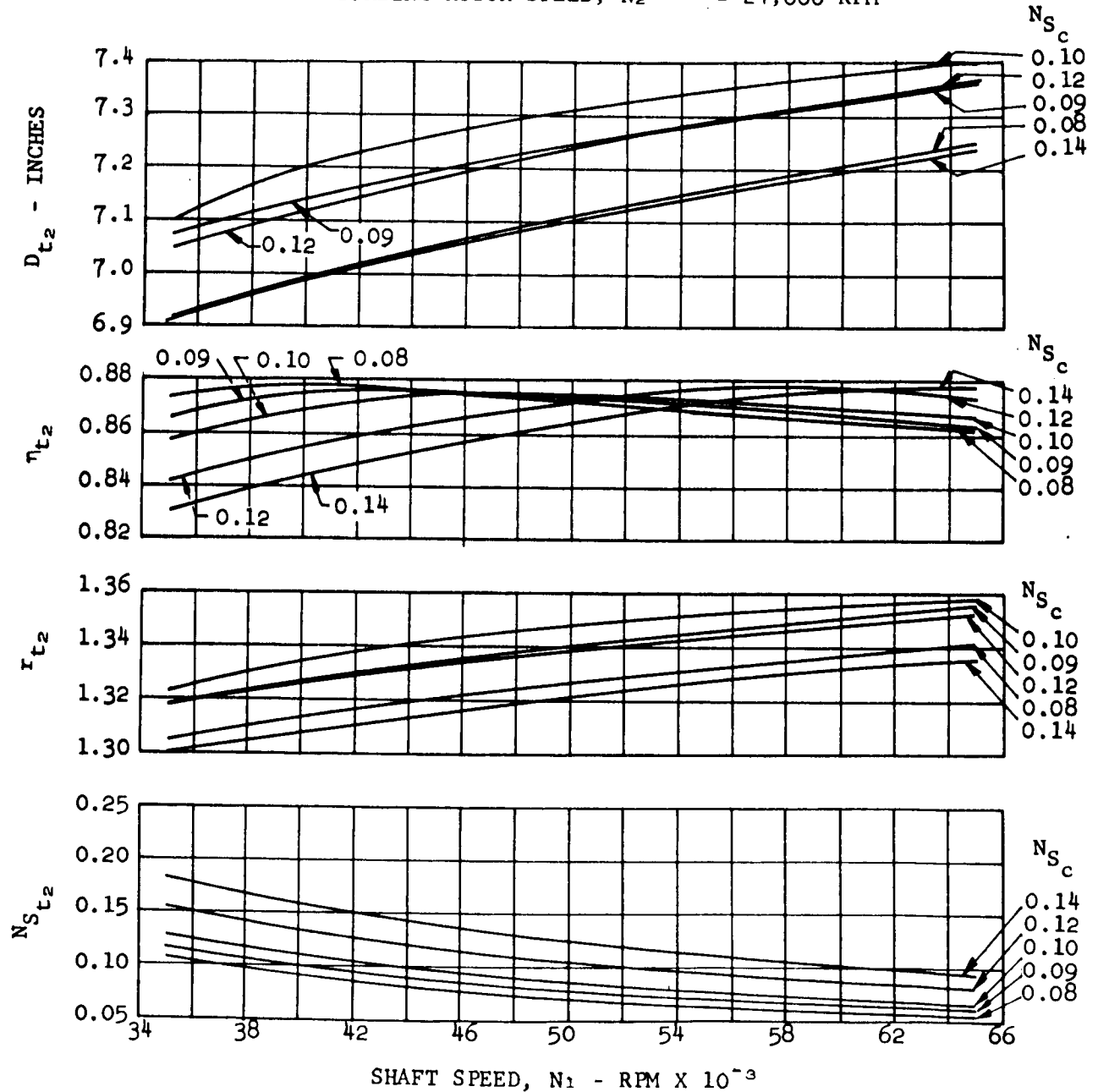
FIRST STAGE TURBINE
OPTIMIZED NASA SYSTEM
FIGURE 10



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COMPRESSOR INLET TEMPERATURE, $T_1 = 536^{\circ}\text{R}$
 TURBINE INLET TEMPERATURE, $T_3 = 1950^{\circ}\text{R}$
 RECUPERATOR EFFECTIVENESS, $E_R = 0.85$
 WORKING FLUID = ARGON
 TURBINE PRESSURE RATIO
 COMPRESSOR PRESSURE RATIO, $\beta = 0.90$
 FREE TURBINE ROTOR SPEED, $N_2 = 24,000 \text{ RPM}$



FREE TURBINE
OPTIMIZED NASA SYSTEM

FIGURE 11

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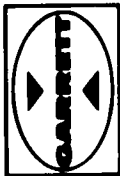


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A comparison was made between NASA design-point conditions at 38,500 rpm (6.0-inch wheels) and 45,000 rpm (5.1-inch wheels) and the "optimized" design-point conditions at 38,500 rpm (6.0-inch wheels), 45,000 rpm (5.25-inch wheels), and 55,800 rpm (4.3-inch wheels). Selected values of cycle parameters at these conditions appear in Table 2. In addition, the optimized conditions include both a high-cycle pressure cycle ($N_{sc} = 0.08$) and a case with compressor specific speed comparable to the NASA design conditions.

The above design-point condition studies revealed no major advantage to be gained through a change in design-point conditions from those proposed by NASA. Some advantage in system weight and radiator area would be realized with increased shaft speed and pressure level; however, decreased wheel diameters would result. Therefore, it was recommended that the mass flow rate, compressor pressure ratio, compressor inlet pressure, and working fluid, as recommended by NASA, be established as design values with the gas generator shaft speed of 38,500 rpm.



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TABLE 2
BRAYTON-CYCLE DESIGN PARAMETERS

	NASA		AiResearch Optimization						
	38,500	45,000	38,500	38,500	45,000	45,000	45,000	55,000	55,000
N_1	0.106	0.123	0.08	0.10	0.08	0.10	0.08	0.08	0.10
N_{Sc}	0.281 0.284	0.284 0.287	-- 0.272	-- 0.283	-- 0.277	-- 0.291	-- 0.281	-- 0.294	-- 0.294
η_{cy}	0.611	0.611	0.531	0.530	0.517	0.517	0.504	0.501	0.501
\dot{m}	12.25 12.40	12.35 12.50	-- 10.45	-- 10.88	-- 10.40	-- 11.00	-- 10.34	-- 10.82	-- 10.82
D_c	6.0	5.1	6.0	6.0	5.26	5.24	4.31	4.28	4.28
D_{t1}	6.0	5.1	6.0	6.0	5.24	5.18	4.27	4.25	4.25
D_{t2}	14.3 7.1	14.4 7.2	-- 6.97	-- 7.18	-- 7.05	-- 7.25	-- 7.15	-- 7.32	-- 7.32
η_c	0.770	0.779	0.739	0.758	0.745	0.763	0.752	0.770	0.770
η_{t1}	0.878	0.871	0.880	0.883	0.883	0.887	0.887	0.885	0.885
η_{t2}	0.854 0.864	0.853 0.864	-- 0.877	-- 0.865	-- 0.875	-- 0.876	-- 0.868	-- 0.873	-- 0.873
P_1	6.0	6.0	8.8	5.4	11.6	7.3	16.6	10.4	10.4
P_3	13.2	13.2	20.2	12.7	26.7	17.0	38.6	23.4	23.4
r_c	2.30	2.30	2.33	2.35	2.34	2.36	2.36	2.38	2.38
r_{t1}	1.567	1.564	1.606	1.592	1.604	1.588	1.601	1.591	1.591
r_{t2}	1.326	1.328	1.310	1.332	1.320	1.342	1.331	1.349	1.349
Vulnerable Area Factor	1.0	1.0	0.826	1.052	0.719	0.908	0.601	0.758	0.758



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Table 3 presents the final design conditions used for the compressor. Design conditions as applicable to the gas generator configuration are given in Section 3.2.

TABLE 3
DESIGN CONDITIONS
NASA COMPRESSOR RESEARCH PACKAGE

Working fluid	Argon
Flow rate, lbs per sec.	0.621
Inlet temperature, °R	520
inlet pressure, psia	6.0
Total pressure ratio	2.30
Operating speed, rpm	37,900
Operating life, hr.	100
Speed capability, percent design	120



3.0 COMPRESSOR DESIGN

3.1 Aerodynamic Design Approach

The impeller diameter was fixed and a preliminary estimate was made of compressor efficiency and slip factor. After an approximate design was established, the compressor efficiency was calculated from a detailed loss analysis and the impeller diameter was modified accordingly.

The impeller inlet was sized to give the minimum inlet relative Mach number at the inducer tip. The shroud and hub profile were designed to give an accelerating flow along the wall from the inlet pipe to the impeller eye as well as a minimum velocity variation from hub to shroud across the inducer inlet section. Initially, a geometric shape of the inlet was selected and the velocity gradients determined by an analog field plot. Using these velocity gradients the velocity distribution was calculated. The geometric shape was altered until a suitable velocity distribution was achieved.

The impeller exit width was established so that the diffusion of relative velocity along the shroud, expressed as a ratio of relative velocities at the impeller inlet and the impeller exit, was less than 0.30. Also, the direction of flow at the diffuser inlet was determined to be not much larger than 60 degrees from the radial direction. With the inlet and exit dimensions fixed, the shroud and hub profile were assumed. Using the electric analog field plot method, with approximate corrections for compressibility, blade blockage and boundary layer clogging, the meridional velocities



along several streamlines were calculated. The meridional shape was adjusted and the calculation repeated until a satisfactory distribution was obtained along the shroud and hub streamlines. Particular care was taken to insure that the shroud meridional velocity did not have too high a peak and that the hub velocity did not become so low as to cause negative velocities.

A slip factor of 0.869 based on temperature rise was used in the impeller design, however, the tangential component of velocity at the impeller exit was based on a value of 0.83, which is consistent with a windage loss of 3.9 percent. The value of slip factor used was based on experimental data and agrees closely with that obtained from Buseman's data for a 15 blade impeller.

Next, the blade loading diagram was estimated using a blade-to-blade calculation method. Utilized in these calculations were an assumed number of blades, an assumed blade angle variation, a calculated slip factor at the blade tip, and an estimated meridional velocity distribution. Since the blade elements were radial, the blade angles along the mean and hub streamlines were calculated from the value of the blade angles at the shroud.

Several combinations of blade angle distribution and numbers of vanes were checked until the following conditions were avoided:

- (1) Excessive deceleration of relative velocity at the shroud.
- (2) Very low or negative relative velocities at the hub.



- (3) Excessive decelerations on the suction surface of any streamline.

The above blade loading calculations were based on the preliminary estimate of meridional flow distribution. The variation of meridional flow distribution due to the above blade loading was calculated using a high-speed digital computer and the blade loading was recalculated based on the modified meridional flow distribution.

In the present design, the original field plot meridional velocity distribution was modified according to past experience, before the blade loading calculation was made. This assumed meridional velocity distribution was very close to the corrected distribution and no iteration was necessary.

The loadings on the radial trailing edge portion of the blade were recalculated based on potential flow analysis. The results compared favorably with the previously calculated loadings.

The losses in the impeller were estimated using two-dimensional boundary layer techniques. The losses considered were blade and wall friction losses, mixing losses due to boundary layer and blade wakes, impeller windage losses, and clearance losses. The Mach number was so low that there was no shock loss in the impeller.



In order to determine the diffuser geometry, equations were derived expressing the diffuser losses in terms of the variable parameters. The parameters considered were radius, diffuser exit vane angle, and number of vanes. The meridional width of the diffuser was allowed to increase from inlet to exit.

The exit core absolute velocity was held fixed at a value 0.6 times the inlet velocity. This was felt to be the maximum diffusion for which separation could be avoided. The required exit diffuser area was calculated to provide the specified diffusion, allowing for boundary layer growth and blade blockage, and the meridional width was adjusted to provide this area. The length of the diffuser vane, or the exit to inlet radius ratio of the diffuser was designed so that the equivalent conical or two-dimensional diffuser cone angle was somewhat less than the stall critical cone angle proposed by Kline.

The losses attributable to the diffuser are the friction losses and the loss of the kinetic energy associated with the radial component of velocity. The total loss was calculated for diffusers of various combinations of parameters and the optimum geometry was determined.

Finally, the blade core velocity distribution was calculated using a potential flow analysis performed on a high-speed digital computer. The overall loading and the suction and pressure velocities were found to be acceptable.



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Surface finish was specified for each component so that the surface was fluid dynamically smooth.

A family of scroll cross sections with different areas was selected for ease of manufacture and blending with the diffuser meridional shape. Using continuity and conservation of angular momentum and assuming inviscid flow, these cross sections were located circumferentially.

Next, the boundary layer thickness at each location was calculated. The area required at each circumferential section was increased by the boundary layer clogging and the original sections were relocated. The process was then repeated until reasonable convergence was obtained. The mean radius of each section was increased from the tang to the exit of the scroll so that considerable diffusion had taken place by the time the flow reached the exit of the scroll. The resulting exit mean Mach number is 0.16, low enough so that no further diffusion is deemed necessary before entering the heat exchanger upstream.



3.2 Aerodynamic Design

The following data and design information defines the impeller, the diffuser, and the scroll configurations resulting from the aerodynamic analysis of the compressor research package. The values are for the compressor design inlet conditions applicable to the gas generator configuration.

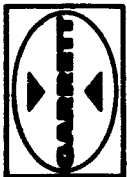
- A. Efficiencies: See Table 4.
- B. Pressure ratios:
 - Total-to-total pressure ratio = 2.38
 - Total-to-static pressure ratio = 2.575
- C. Speed = 38,500 rpm
- D. Specific work = 49.45 np per lb per sec.
- E. Weight flow = 0.611 lbs per sec.
- F. Specific speed = 0.1057
- G. Loss distribution: See Table 4.
- H. Total and static pressures: See Table 4.
- I. Total temperatures: See Table 4.
- J. Gas velocity distribution:
 - See Figure 12 for shroud velocity distribution.
 - See Figure 13 for 50 percent streamline velocity distribution.
 - See Figure 14 for hub velocity distribution.
 - See Figure 15 for diffuser velocity distribution.



TABLE 4

NASA BRAYTON CYCLE COMPRESSOR
DESIGN VALUES

LOCATION	P_{TOT} (PSIA)	P_{STATIC} (PSIA)	T_{TOTAL} (°R)	η (UP TO LOCATION)
Inlet (Outside blade)	6.00	5.55	536	--
Impeller Exit (After B.L. and blade wake mixing)	15.72	9.70	817.3	0.896
Diffuser Inlet (Inside blade based on core velocity)	15.57	10.21	817.3	0.886
Diffuser Exit (Inside blade based on core velocity)	15.57	13.47	817.3	--
Scroll Inlet (Based on mean velocity after B.L. and blade wake mixing)	14.58	13.14	817.3	0.813
Scroll Exit (Based on mean velocity at scroll exit)	14.29	13.97	817.3	0.798

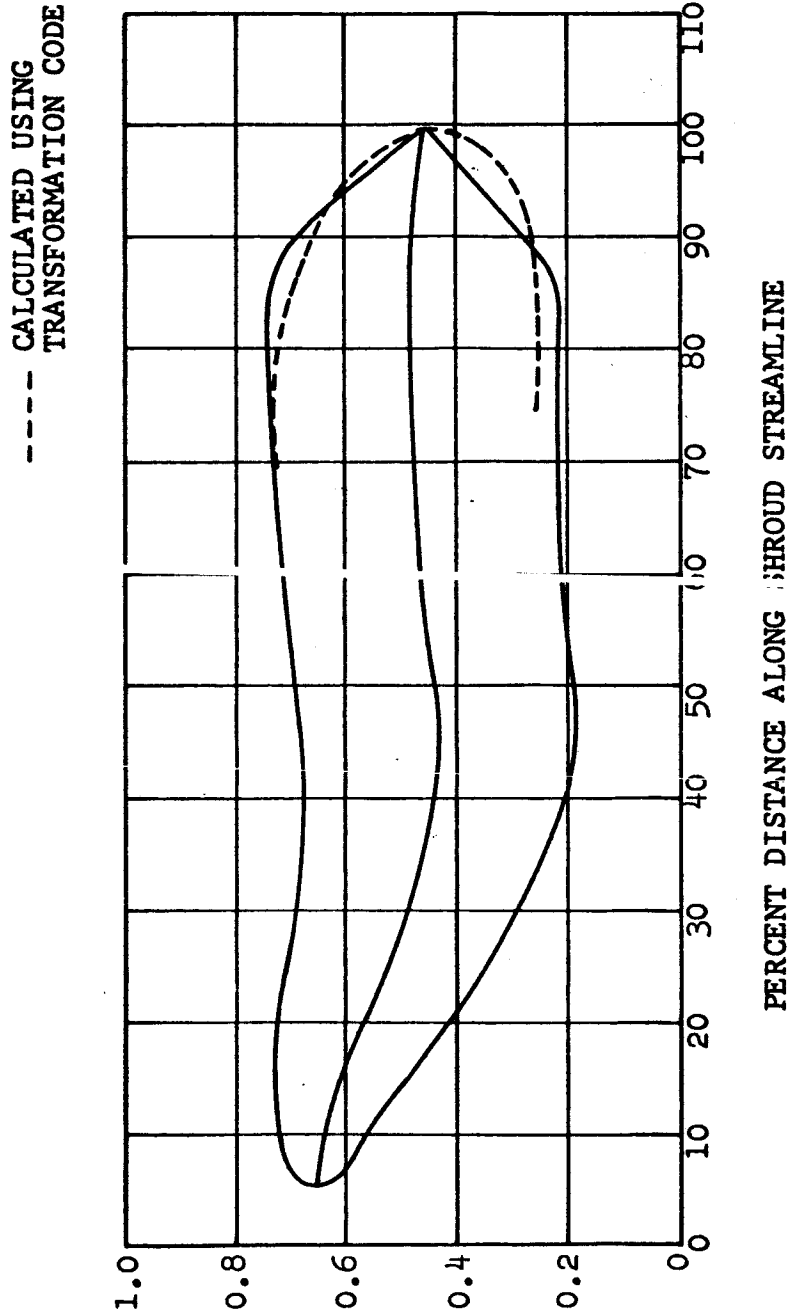


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BLADE RELATIVE VELOCITY
INLET STAGNATION - VELOCITY OF SOUND



COMPRESSOR IMPELLER VELOCITY DISTRIBUTION
SHROUD STREAMLINE

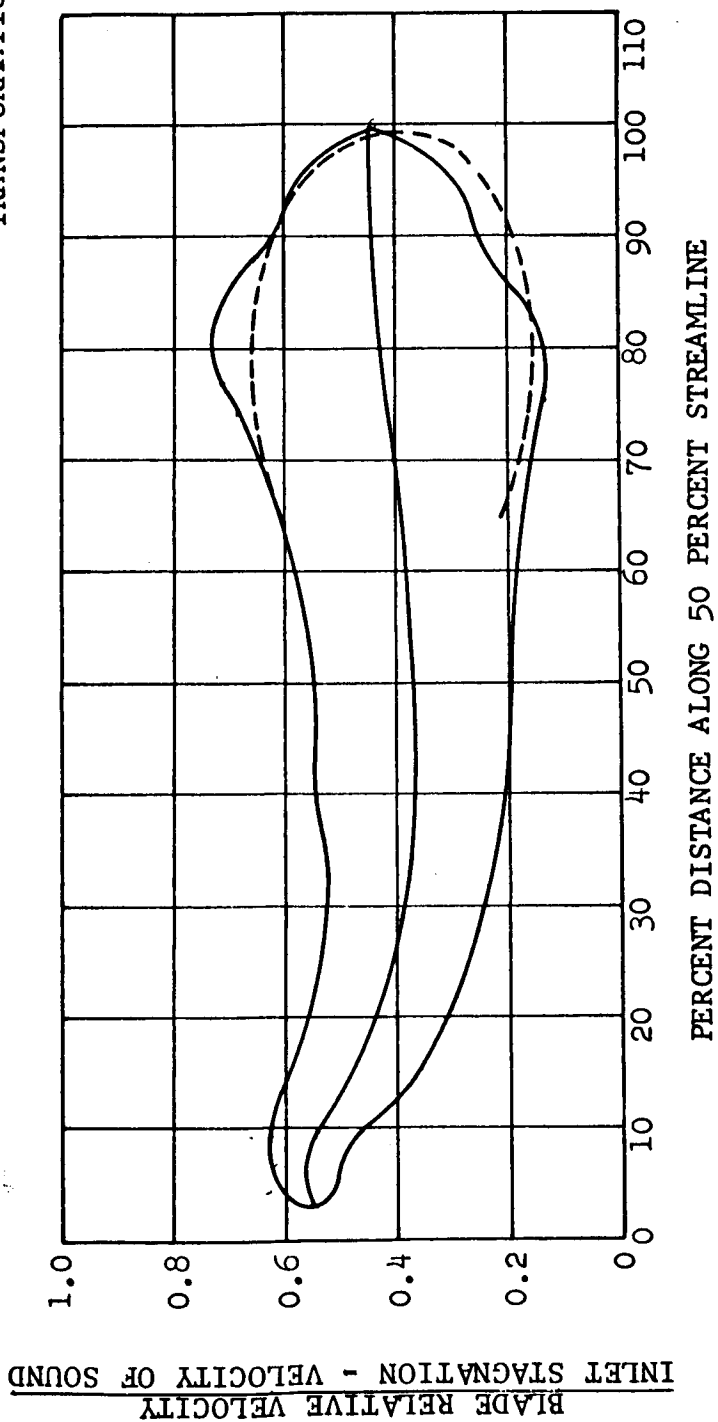
A31544

FIGURE 12



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----- CALCULATED USING
TRANSFORMATION CODE



COMPRESSOR IMPELLER VELOCITY DISTRIBUTION
50 PERCENT STREAMLINE

FIGURE 13

A31545

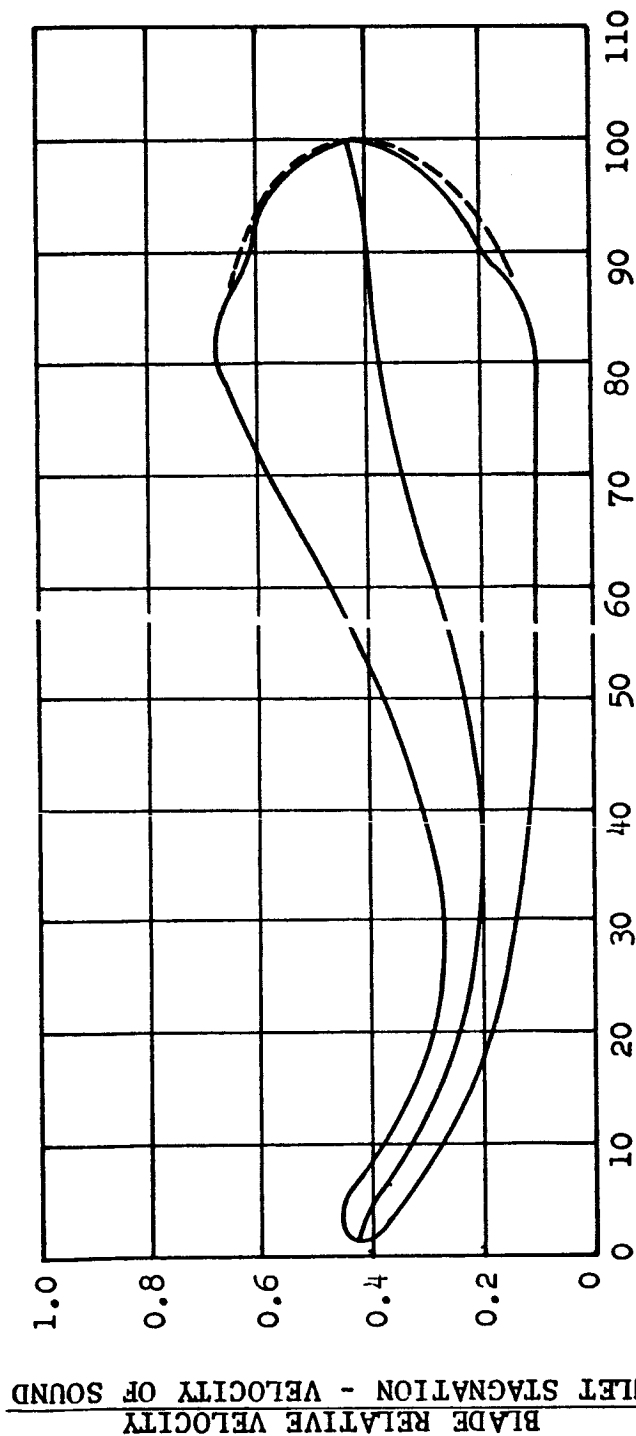


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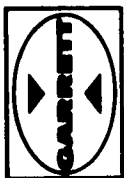
----- CALCULATED USING
TRANSFORMATION CODE



PERCENT DISTANCE ALONG HUB STREAMLINE
COMPRESSOR IMPELLER VELOCITY DISTRIBUTION
HUB STREAMLINE

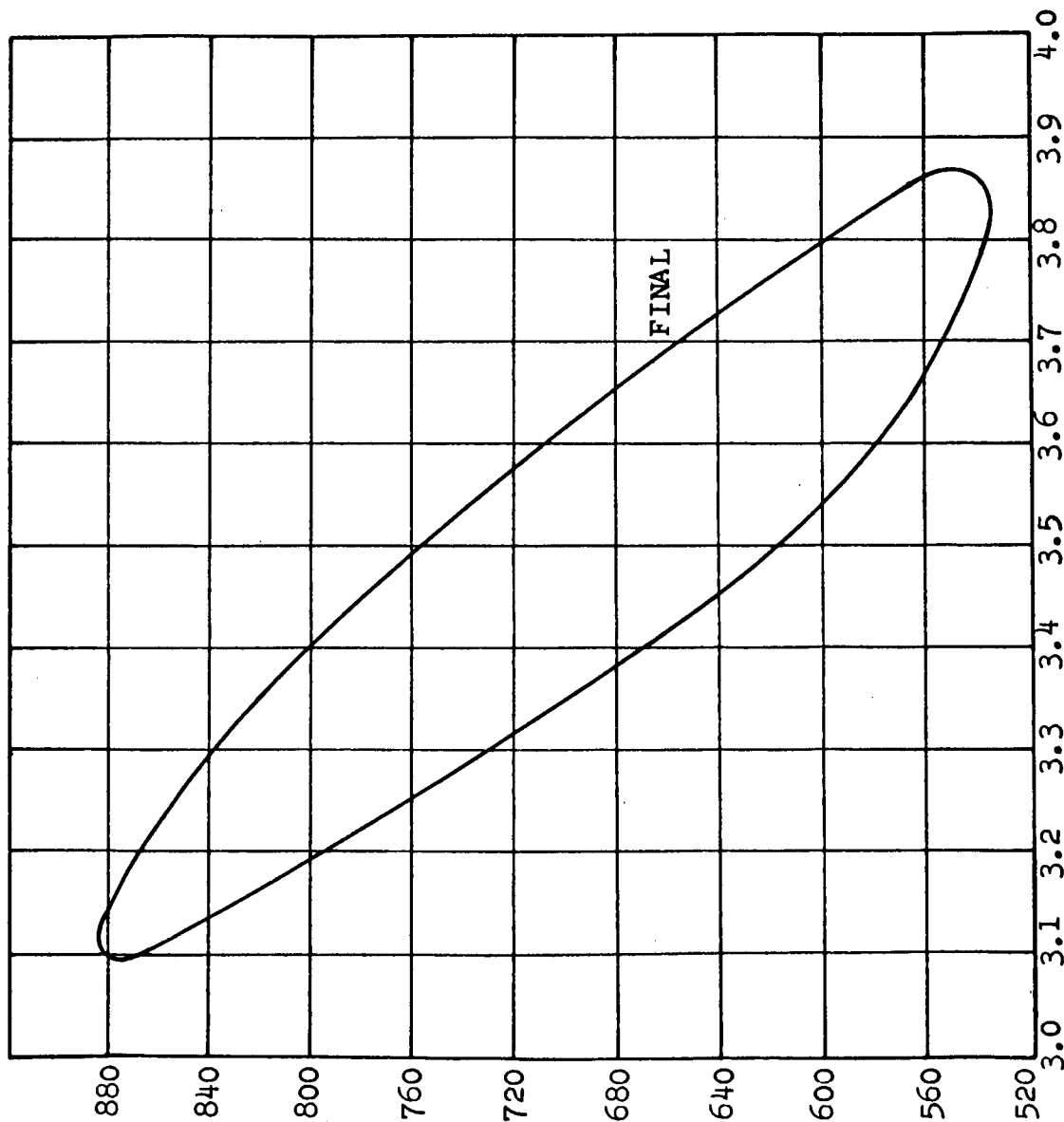
A31546

FIGURE 14



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ABSOLUTE VELOCITY C - FT/SEC.



RADIUS - INCH
COMPRESSOR DIFFUSER BLADE LOADING
FIGURE 16

A215117



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K. Stator and rotor aerodynamic physical dimensions:

See Drawing 369757 for stator physical dimensions.

See Drawing 369747 for rotor physical dimensions.

See Drawing 369758 for diffuser physical dimensions.

L. Velocity diagrams:

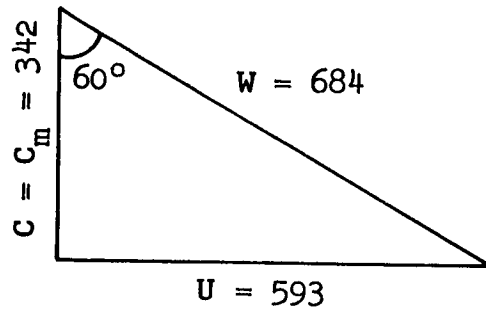
See Figure 16 for impeller velocity diagram.

See Figure 17 for diffuser and scroll velocity diagrams.

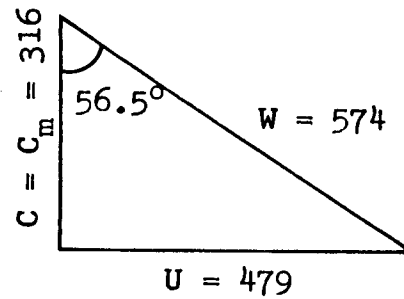


FIGURE 16

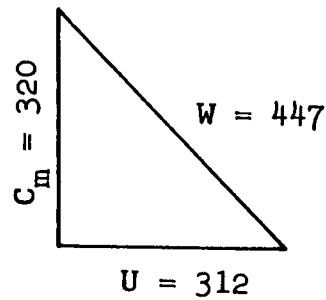
INLET VELOCITY TRIANGLES (JUST UPSTREAM OF LEADING EDGE)



INLET SHROUD



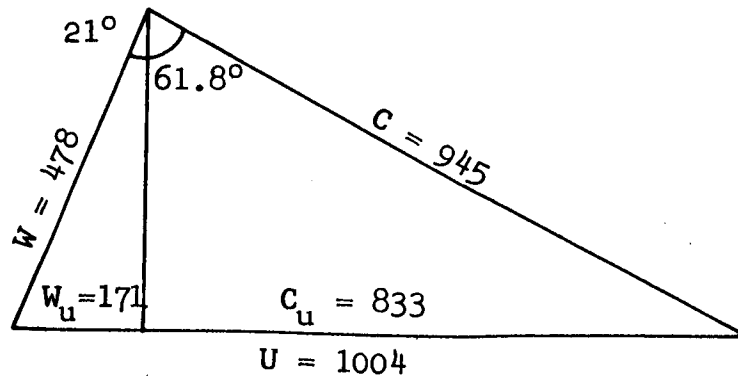
INLET MEAN



INLET HUB

IMPELLER

EXIT VELOCITY TRIANGLE (AFTER MIXING OF B. L. AND BLADE WAKES)



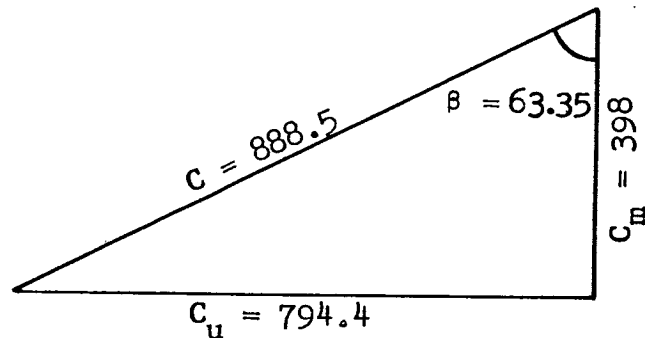
EXIT MEAN

A31474

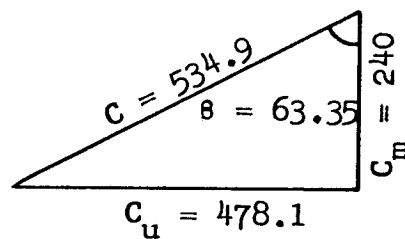


FIGURE 17

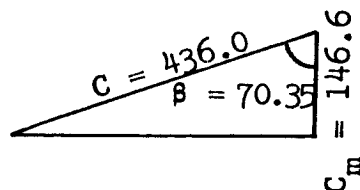
1. DIFFUSER INLET: (INSIDE BLADE AT $R = 3.10$ IN.)



2. DIFFUSER EXIT: (INSIDE BLADE AT $R = 3.875$ IN.)



3. SCROLL INLET: (AFTER MIXING OF DIFFUSER B.L. AND BLADE WAKES)



4. THE MEAN EXIT VELOCITY AT THE SCROLL EXIT IS 208.6 F/S
YIELDING $M_{exit} = .161$

A31475



3.3 Mechanical Design Analysis

3.3.1 Dynamic Analysis

The dynamic analysis of the compressor research package was accomplished on a digital computer by use of a program written for turbomachinery critical speed and bearing load analyses. A 4.0-inch bearing spacing and 25-millimeter bearings were chosen for the research package. The rear bearing was rigidly mounted to limit the rotor radial motion within the required shroud clearance. This gave an effective spring rate of 450,000 pounds per inch for the rear mount. The critical speeds for the compressor were analyzed by using front bearing spring rates from 5,000 to 30,000 pounds per inch (see Figure 18). Using a front bearing spring rate of 15,000 pounds per inch, bearing loads for a 0.0005-inch c.g. eccentricity were determined. Figure 19 shows the results of the bearing load analysis.

3.3.2 Impeller Stress Analysis

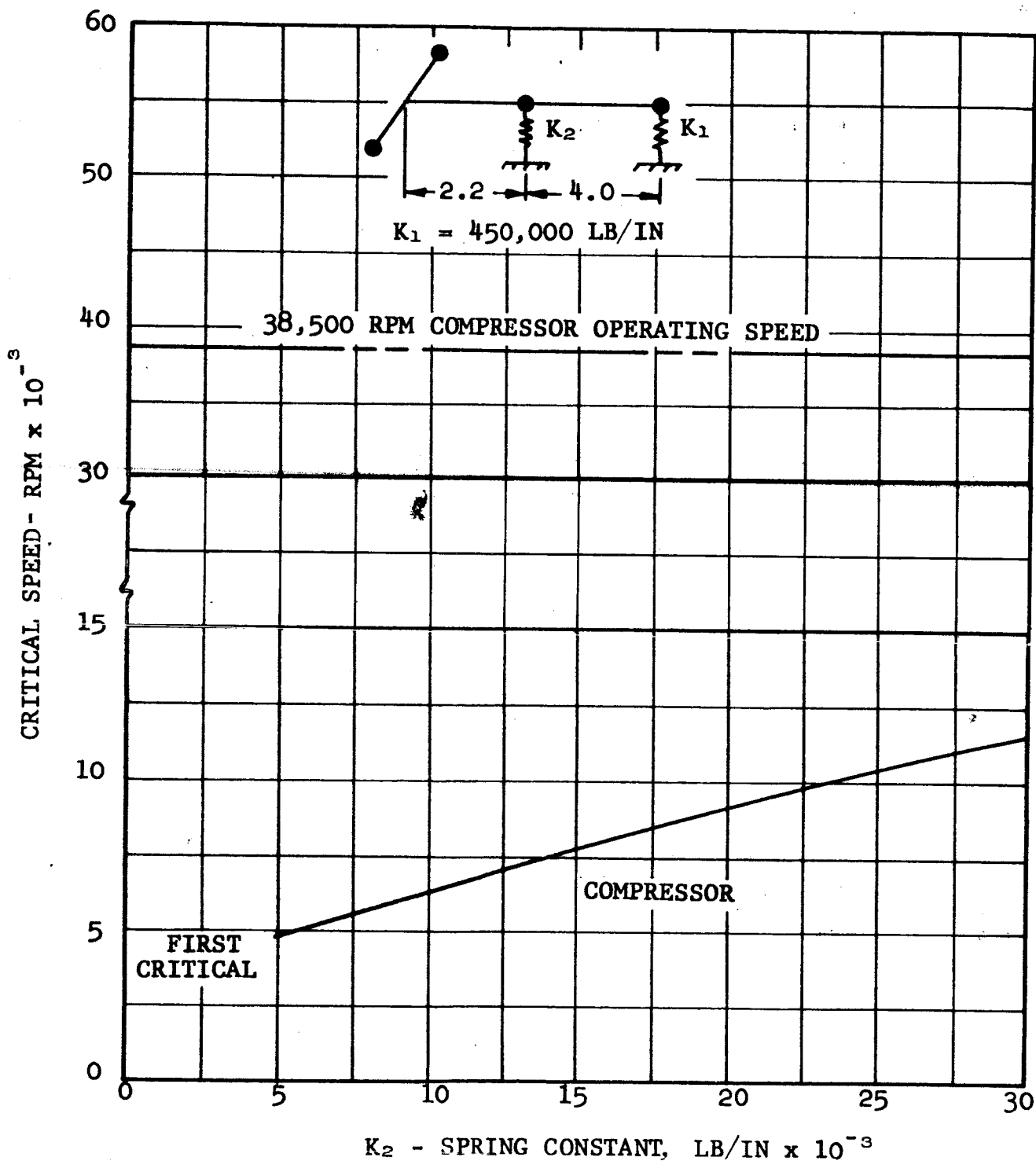
A stress analysis of the compressor wheel was performed for determining the centrifugal disc and blade stresses at the operating speed of 38,500 rpm. Disc stresses were calculated using the Bendix G-20 digital computer. Blade stresses were calculated using lamination theory. Radial and centrifugal stresses are plotted on Figures 20 and 21 respectively.

Yielding of the disc may be expected at a minimum speed of 82,000 rpm and in the blades at 122,000 rpm. The burst speed range may be expected to be between 119,000 rpm and 133,000 rpm.



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COMPRESSOR
 $M = 0.0071 \text{ LB SEC}^2/\text{IN.}$
 $J = -0.00048 \text{ IN. LB SEC}^2$
 $D = 6.0 \text{ IN.}$

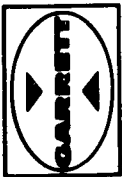


CRITICAL SPEED VS. SPRING CONSTANT FOR
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 18

APS-5109-R
Page 37

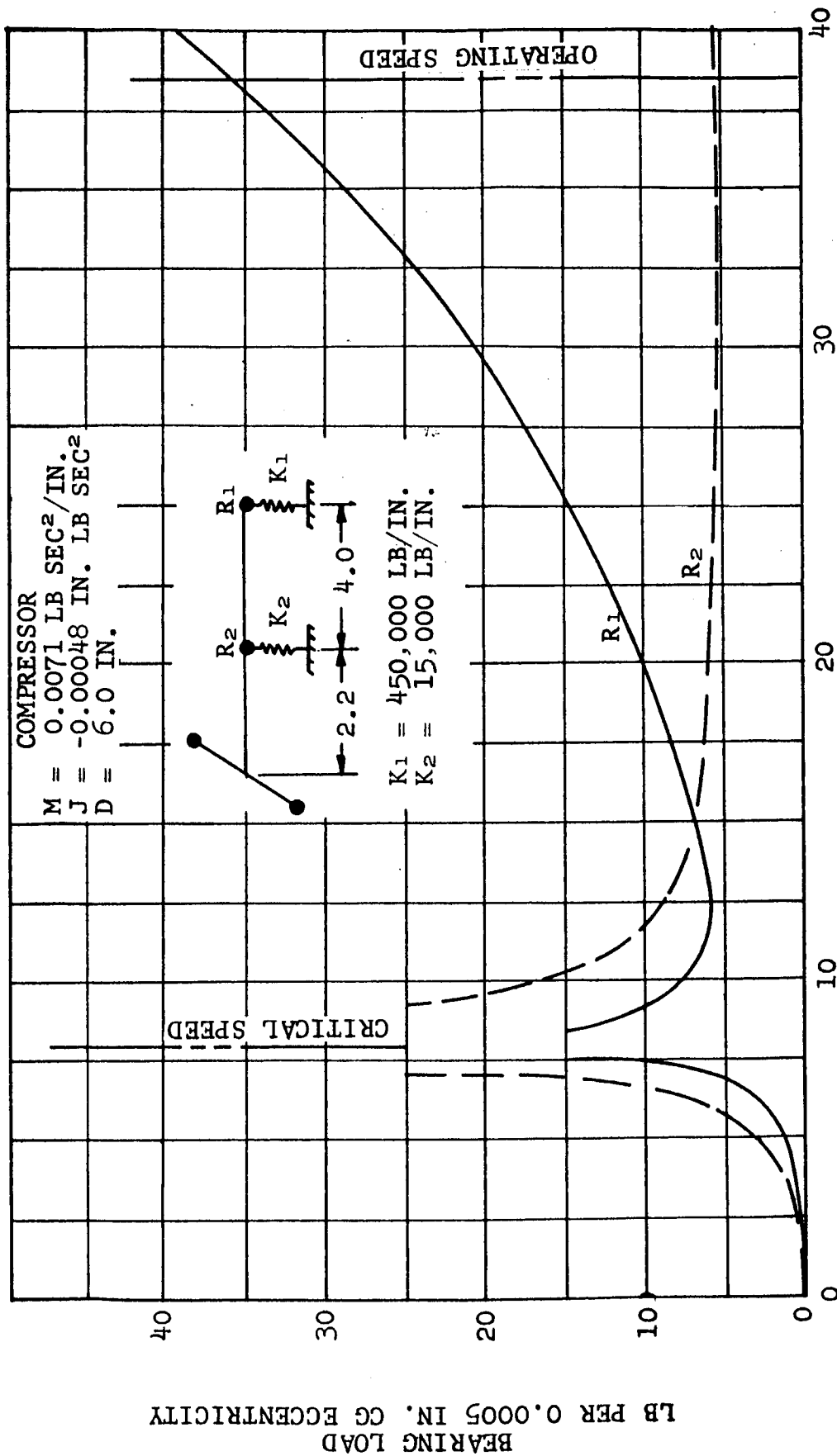
A30736-2



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SPEED - RPM x 10⁻³

**BEARING LOADS VS SPEED FOR
NASA COMPRESSOR BALL BEARING
TEST RIG WITH $K_2 = 15,000 \text{ LB/IN.}$**

FIGURE 19

A30737-1

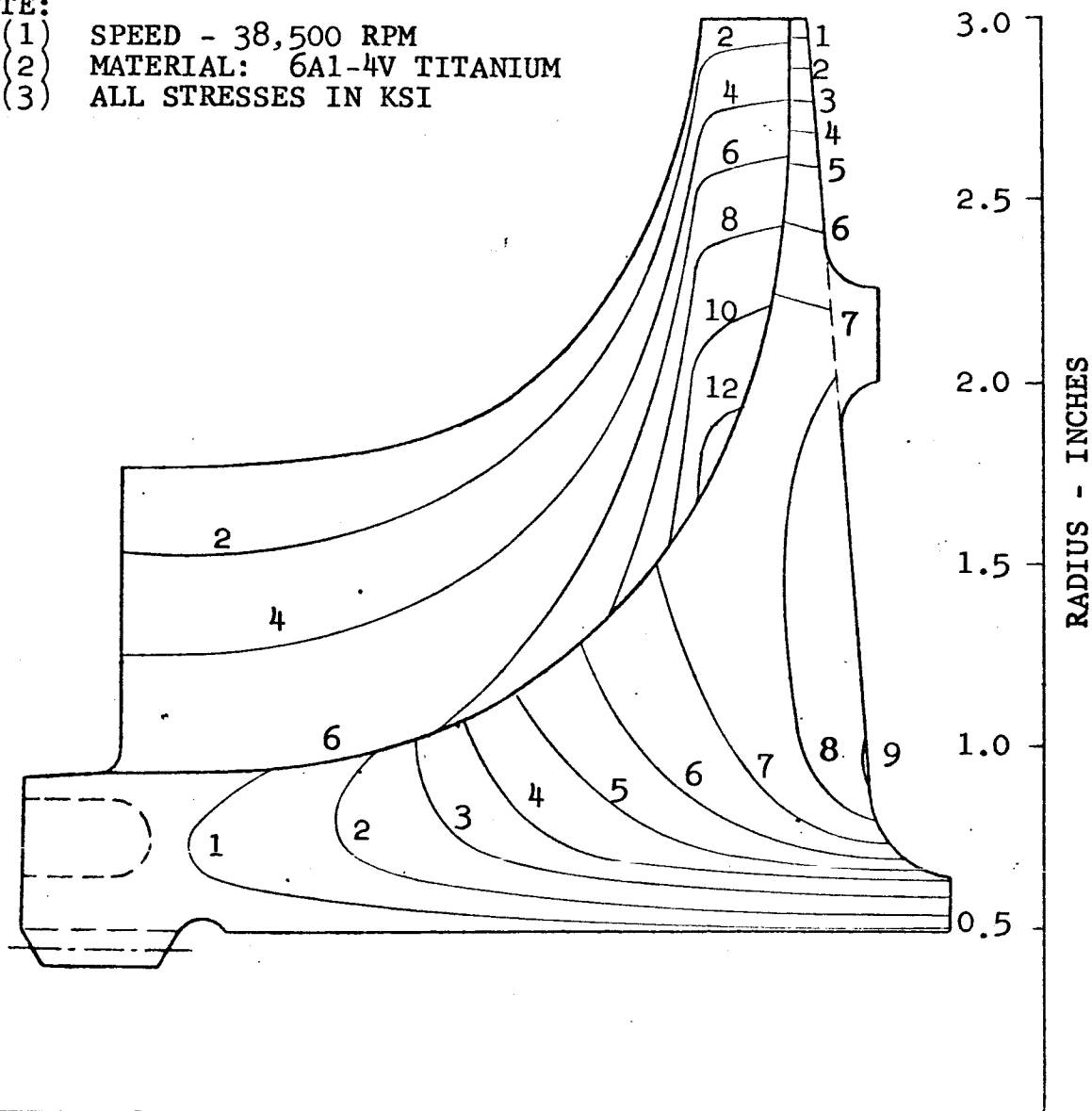


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NOTE:

- (1) SPEED - 38,500 RPM
- (2) MATERIAL: 6Al-4V TITANIUM
- (3) ALL STRESSES IN KSI



PREPARED	DW	10-63	RADIAL STRESS DISTRIBUTION FOR NASA BRAYTON-CYCLE IMPELLER	A12037
WRITTEN				
APPROVED	<i>R. W. N.</i>	10-63	AiResearch Manufacturing Company of Arizona	

FORM 7792A-1

FIGURE 20

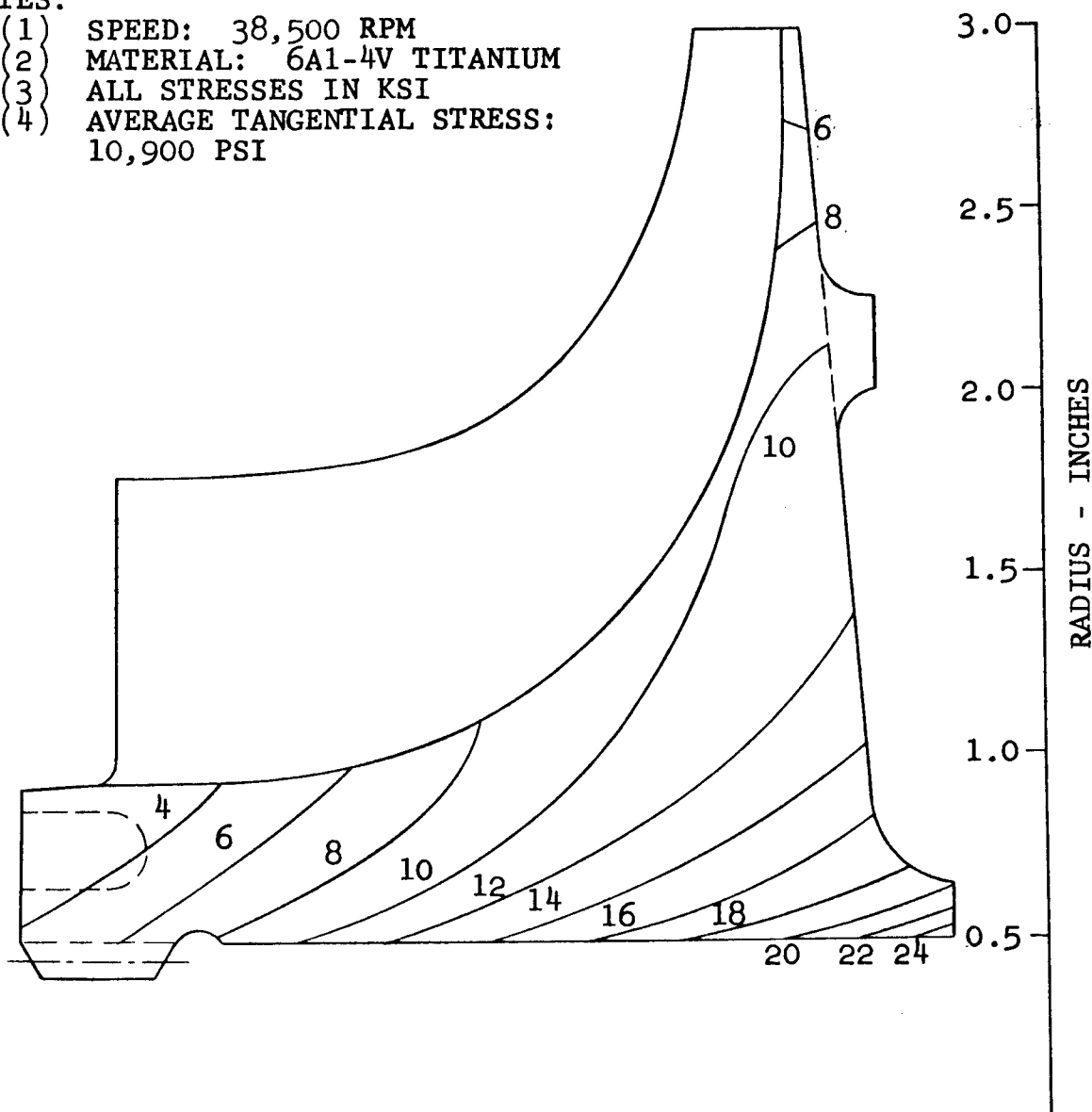


AIResearch MANUFACTURING COMPANY

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PHOENIX, ARIZONA

NOTES:

- (1) SPEED: 38,500 RPM
- (2) MATERIAL: 6Al-4V TITANIUM
- (3) ALL STRESSES IN KSI
- (4) AVERAGE TANGENTIAL STRESS:
10,900 PSI



PREPARED	DW	10-63	TANGENTIAL STRESS DISTRIBUTION NASA BRAYTON CYCLE IMPELLER	A12038
WRITTEN				
APPROVED	<i>R.M.</i>	10-63	AiResearch Manufacturing Company of Arizona	

FORM P702A-1

FIGURE 21

APS-5109-R

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Since creep of the titanium alloy is negligible at the temperatures encountered in this application, the impeller may be considered essentially an infinite life wheel.

The wheel was machined from a 6 Al-4V titanium forging and piloted on the shaft (rather than on the splines) with a 0.0003 to 0.0006 inch diametral interference

3.3.3 Stresscoat Growth and Burst Tests

Results of the Stresscoat test conducted on the compressor impeller are shown in Figures 22 and 23. The maximum stress at 38,500 rpm as indicated by the Stresscoat in Figure 22, Area 1, was 21,200 psi. The maximum calculated stress in the hub was 24,800 psi at an area covered by the arbor. Calculated stresses of approximately 20,000 psi in the hub radius correspond closely with the Stresscoat results, as does the Stresscoat results for the remainder of the back face. The maximum radial stress in the blade indicated by the Stresscoat was 14,400 psi at 38,500 rpm compared to a calculated value of 13,200 psi.

A growth test was conducted on the subject wheel with residual diametral deformations recorded for the OD and for the ID at both the spline (inducer) end and the bearing (hub) end. Results of this test are shown in Figure 24. In an attempt to burst the wheel, substantial growth of the ID (0.026- to 0.030-inch residual deformation) occurred and allowed the c.g. of the wheel to shift relative to the arbor. Violent precession resulted, and the wheel dropped in the whirlpit (see Figure 25) at 124,000 rpm. This speed falls within the calculated burst speed range of



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STRESSCOAT TEST OF NASA BRAYTON-CYCLE
COMPRESSOR WHEEL, PART 369747
STRESSES BELOW ARE FOR UNIAXIAL STRESSES AT 38,500 RPM.
ESTIMATED ACCURACY OF STRESSES: ± 20 PERCENT.

AVERAGE STRESSCOAT SENSITIVITY: 0.0007.

MATERIAL: 6Al-4V-90 TITANIUM ALLOY.

$E = 16.0 \times 10^6$ PSI

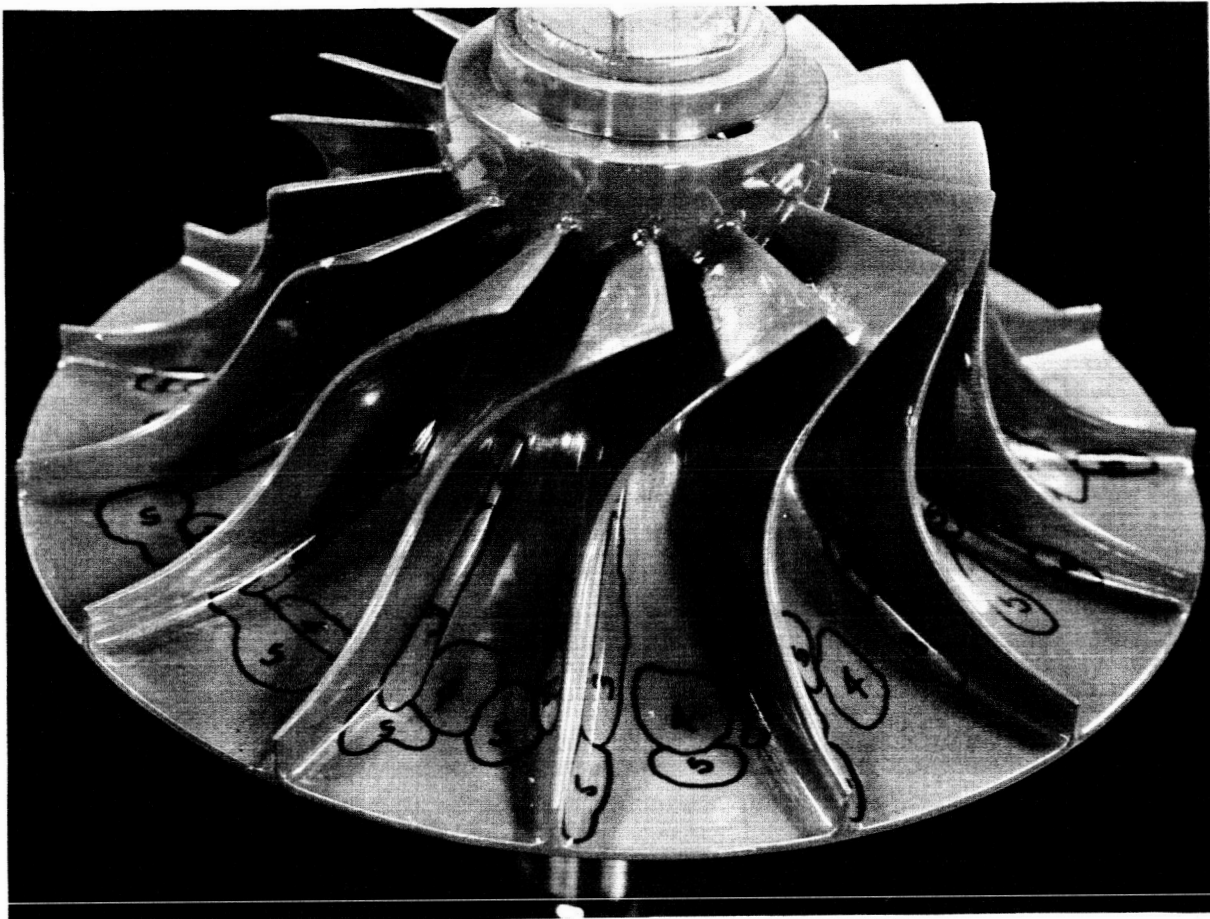
MARCH 6, 1964

<u>AREA</u>	<u>STRESS (PSI)</u>
1	21,200
2	17,300
3	14,400
4	11,950
5	9,850

FIGURE 22

APS-5109-R

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STRESSCOAT TEST OF NASA BRAYTON-CYCLE
COMPRESSOR WHEEL, PART 369747
STRESSES BELOW ARE FOR UNIAXIAL STRESSES AT 38,500 RPM.
ESTIMATED ACCURACY OF STRESSES: ± 20 PERCENT.
AVERAGE STRESSCOAT SENSITIVITY: 0.0007.
MATERIAL: 6Al-4V-90 TITANIUM ALLOY.
 $E = 16.0 \times 10^6$ PSI
MARCH 6, 1964

AREA	STRESS (PSI)
1	21,200
2	17,300
3	14,400
4	11,950
5	9,850

FIGURE 23

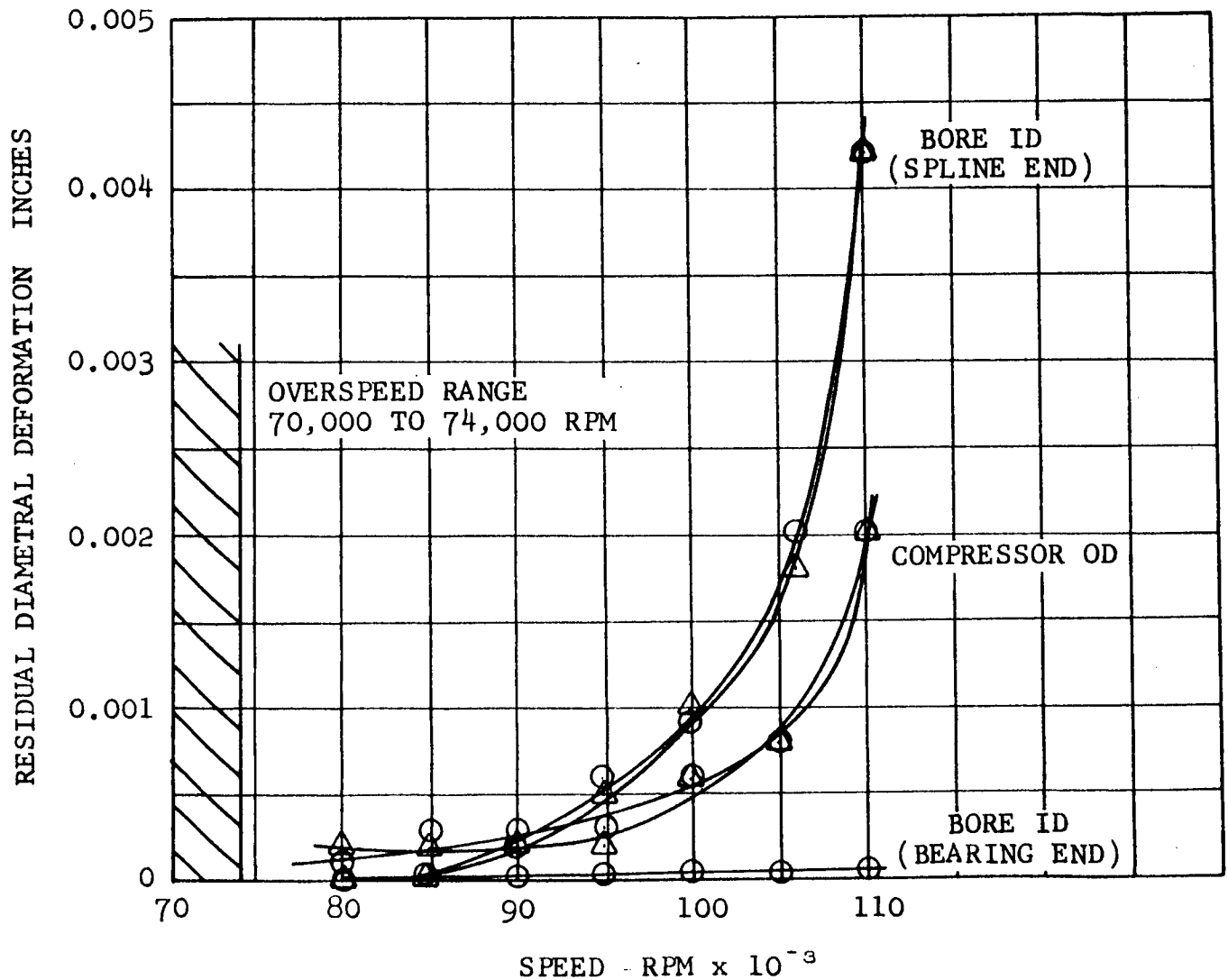


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S/N 4X 101
MATERIAL: 6 Al - 4V - TITANIUM
○ AND △ INDICATE MEASUREMENTS TAKEN AT
RIGHT ANGLES

PRERUN DIAMETERS:

COMP. OD 5.9800 AND 5.9799
BORE ID 0.9841 AND 0.9842 (BEARING END)
BORE ID 0.9847 AND 0.98475 (SPLINE END)



GROWTH OF NASA BRAYTON CYCLE
COMPRESSOR WHEEL

A60261-1

FIGURE 24



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BURST TEST RESULTS
NASA COMPRESSOR IMPELLER

FIGURE 25



119,000 to 133,000 rpm; and while actual fracture of the wheel did not occur because of the large elongation of the material, the burst speed is in excess of the minimum calculated value, and a rerun of the burst test is not considered necessary.

3.4 Mechanical Design

3.4.1 General Unit Description

A cross-sectional view of the compressor research package is shown in Assembly Drawing 369731. The unit consists of the compressor impeller (369747) and shaft (369754) mounted in the main housing (369722) on two antifriction bearings (358313). The front bearing (impeller end) is resiliently mounted with a spring rate of 15,000 pounds per inch using the bearing mount described in Drawing 369733. This spring rate was chosen so that the first critical speed could be far removed from the operating speeds. Bearing loads are also small. The rear bearing is rigidly mounted to limit the impeller-to-shroud clearances, and a coil spring (111917) provides 30 pounds of axial preload on the bearings. An oil jet (369728) supplies pressurized oil to each bearing, and a carbon-face-type oil seal (358319 and 358321) is provided at each end of the housing. The impeller end seal (358321) has an argon purge chamber located between the face seal and an annular shaft seal to prevent any minute amount of oil leakage past the face seal from contaminating the system argon. A spline-connected stub shaft (369746) located at the input end of the compressor permits adaptation to an external power source.



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The compressor diffuser (369758) and the scroll assembly (369757) attach to the main housing by a bolted flange. Shimming to obtain the desired compressor-wheel-shroud-face clearance is accomplished at this flange by providing a shim of predetermined thickness between the housing and the scroll flanges (369745). A compromise value of the clearance was established at 0.010- to 0.012-inch. From an aerodynamic consideration, a zero clearance would be optimum, however, a clearance of 0.002 inches per inch of wheel diameter can be utilized without serious performance penalty. From a mechanical consideration, it is advantageous to maintain large clearances so that, with rotor radial and axial displacements due to tolerance stackup and flexible-bearing displacements, the rotor does not rub on the shroud. Sealing at this shim is accomplished with two O-rings (369813). Bolted flanges at the compressor inlet and discharge permit adaptation of appropriate ducting (369778 and 369779). A rigid mounting base (369752) provides for mounting the compressor research package to a test stand bed plate. Table 5 is a summary of the parts used on the compressor research package.

3.4.2 Instrumentation

Provisions were made for providing certain instrumentation on the compressor as follows:

(a) Inlet

- (1) Three static pressure taps in the same plane, 120 degrees apart.
- (2) Three total pressure taps, 1/2-inch downstream of the static pressure taps, 120 degrees apart.

(b) Rotor

- (1) One L.C. Smith probe mount, 0.3 inch upstream of the rotor leading edge.



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TABLE 5
NASA COMPRESSOR RESEARCH PACKAGE
DRAWING AND PARTS LIST

DRAWING OR PART NUMBER	TITLE	USED ON	NUMBER REQUIRED
369730	Compressor Outline, Test, Brayton Cycle		
369731	Compressor Assembly, NASA		
369722	Housing Assembly, Main	369731	1
369723	Spacer	369731	1
369727	Carrier, Seal	369731	1
369728	Nozzle Assembly, Oil	369731	1
369729	Spacer, Labyrinth (Optional)*	369731	1
369732	Spacer, Bearing	369731	1
369733	Mount, Bearing, Resilient	369731	1
369734	Carrier, Bearing	369731	1
369735	Carrier, Bearing	369731	1
369737	Carrier Assembly, Labyrinth Seal (Optional)	369731	1
397738	Carrier Assembly, Carbon Seal	369753	1
369743	Shim, Bearing Carrier	369731	As required
369744	Shim, Seal	369731	As required
369745	Shim, Sealing Spacer	369731	1
369746	Shaft, Quill**	369731	1
369747	Impeller, Compressor	369731	1
369752	Shaft Assembly, Mounting	369731	1
369753	Seal Assembly, Carbon	369731	1
369754	Shaft Assembly, Impeller	369731	1
369757	Scroll Assembly, Compressor	369731	1
369758	Diffuser, Compressor	369731	1
369759	Shim, Housing, Diffuser	369731	1
369770	Spinner, Impeller	369731	1
369771	Washer, Flat	369731	1
369778	Flange, Compressor Outlet**		1
369779	Flange, Compressor Inlet**		1
369813	Seal, O-Ring	369731	1
111917	Spring, Compression	369731	1
358313	Ball, Single Row, Angular Contact	369731	2
358319	Seal, Air-Oil	369731	1
358320	Seal, Argon, Oil-Metal Bellows (Optional)	369731	1
358321	Seal Set, Gas-Oil, Matched	369753	1
<u>COMMERCIAL PARTS</u>			
MS16555-617	Pin	369722	2
MS16555-625	Pin	369722	3
MS16555-646	Pin	369752	2
MS21045-C5	Nut	369731	12
MS24630-2	Type "F" Screw	369731	2
MS24673-2	Screw	369731	12
MS24673-5	Screw	369731	4
MS29561-015	O-Ring	369731	2
MS29561-235	O-Ring	369731	1

*Parts marked (optional) may be used as an alternate.

**These parts are shipped loose to the customer. The flanges are to be welded into the customer's inlet and exhaust ducting.

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PHOENIX, ARIZONA**TABLE 5 (Cont'd)**

DRAWING OR PART NUMBER	TITLE	USED ON	NUMBER REQUIRED
AN3CH5A	Screw	369731	6
AN5C-12	Screw	369731	12
AN6CH10A	Screw	369731	4
AN816-5-4S	Connector	369728	1
AN960C516L	Washer	369731	12
AN960C616	Washer	369731	4
S8152AT101-0-120	Pin Diam = 0.0469 Length = 0.120 Material: SAE 4340	Both ends chamfered 45° x 0.008 Hardness - R _c 40 369734	1
S8152AT101-0-220	Pin Diam = 0.0469 Length = 0.220 Material: SAE 4340	Both ends chamfered 45° x 0.008 Hardness - R _c 40 369733	1
S8152BG17-0-590	Pin	369738	2
S8152BG17-0-590	Pin Diam = 0.2032 Length = 0.590 Material: 321 Cres Steel	Both ends chamfered 45° x 0.015 Condition: A annealed 369737	2
S8152BG40-0-310	Pin Diam = 0.2500 Length = 0.310 Material: 321 Cres Steel	Both ends chamfered 45° x 0.025 Condition: A annealed 369728	1
S8171AP102	Packing "O" Ring Dia of "O" Ring = 7.445. Dia of Ring Material = 0.137 Material: MIL-R-25897 Type I, Class 1 (Viton "A" Material)	369731	1
S8860C1P1	Plate, Identification Thickness: 0.016 Material: 301 Cres Steel Condition: 1/2 Hard	369731	1
362-506-9012	Gasket - Metal O-Ring 3/32-inch tube, 0.010-inch wall Ring OD = 5.252 inch Vendor: The D.S.D. Manufacturing Company Hamden, Connecticut Fed Sup Code 97968 Vendor Part No. C5250C-AG	369731	1
362-522-9002	Gasket - Metal O-Ring 1/16-inch tube, 0.010-inch wall Ring OD = 0.502 inch Vendor: The D.S.D. Manufacturing Company Hamden, Connecticut Fed Sup Code 97968 Vendor Part No. A0500C-AG	369731	2
525-518-9047	Nut, Self-Locking, Hex, Thin, 750°F Thread: 3/4 - 16 NF-3 Vendor: Standard Press Steel Co Jenkintown, Pennsylvania Fed Sup Code 56878 Vendor Part No. 50FK1216	369731	1
525-577-9006	Nut, Self-Locking, Round, Bearing Retaining Nominal OD = 1 9/16 inch Vendor: Shur-Lok Corporation Anaheim, California Fed Sup Code 97393 Vendor Part No. S658N05C	369721	1
655-601-9208	Screw, Self-Locking, Socket, Head, 1200°F Length: 0.50 Vendor: Long-Lok Corp. Los Angeles, California Fed Sup Code 03038 Vendor Part No. DT 100000-82-8	369731	1



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- (2) Three sets of five static pressure taps along the rotor shroud, each set 120 degrees apart.

(c) Diffuser

Two sets of five static pressure taps along the diffuser 50 percent streamline, one set on each side of the diffuser passage.

(d) Scroll Exit

- (1) Four static pressure taps in the same plane 90 degrees apart.
- (2) Four total pressure taps 0.5 inch downstream of the static taps, rotated 45 degrees to the static taps and 90 degrees apart.

(e) Bearings

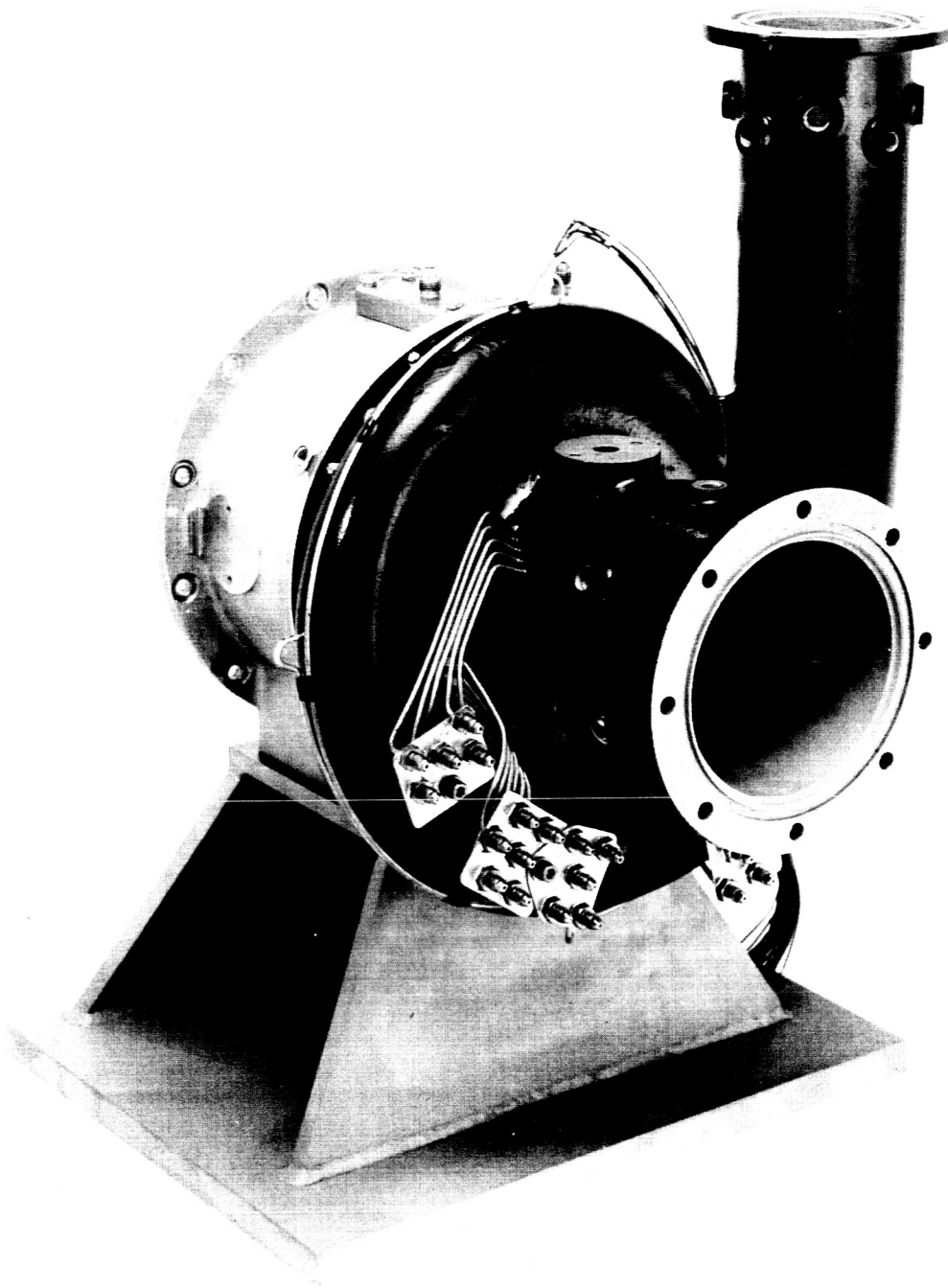
Three I.C. thermocouples on each bearing.

On the first shipping unit, Serial No. P-A, drilled and tapped bosses and the associated instrumentation plumbing was provided as shown in Figures 26, 27, and 28. For the second shipping unit, Serial No. P-B, undrilled bosses were provided.



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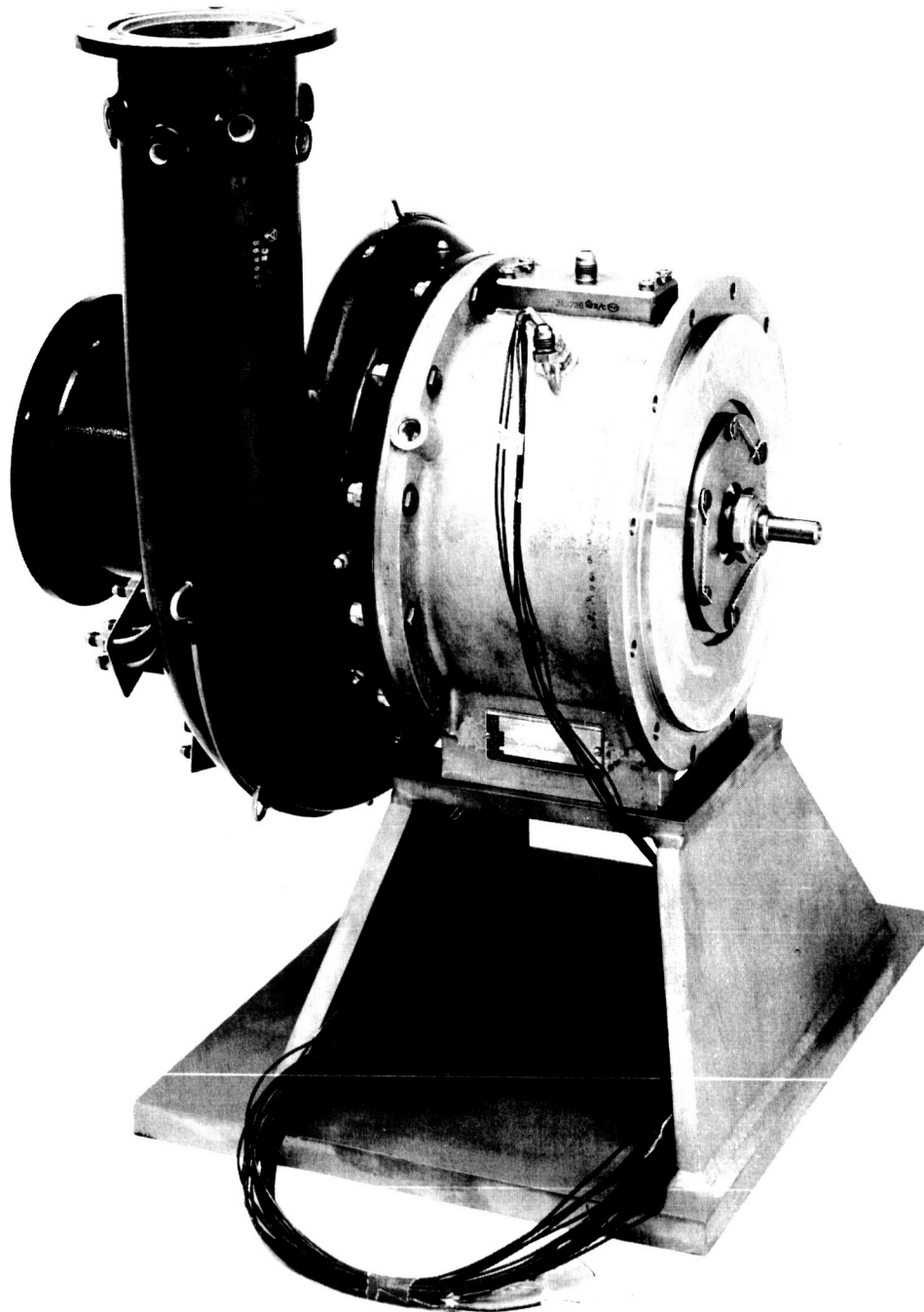
SERIAL NO. P-A
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 26



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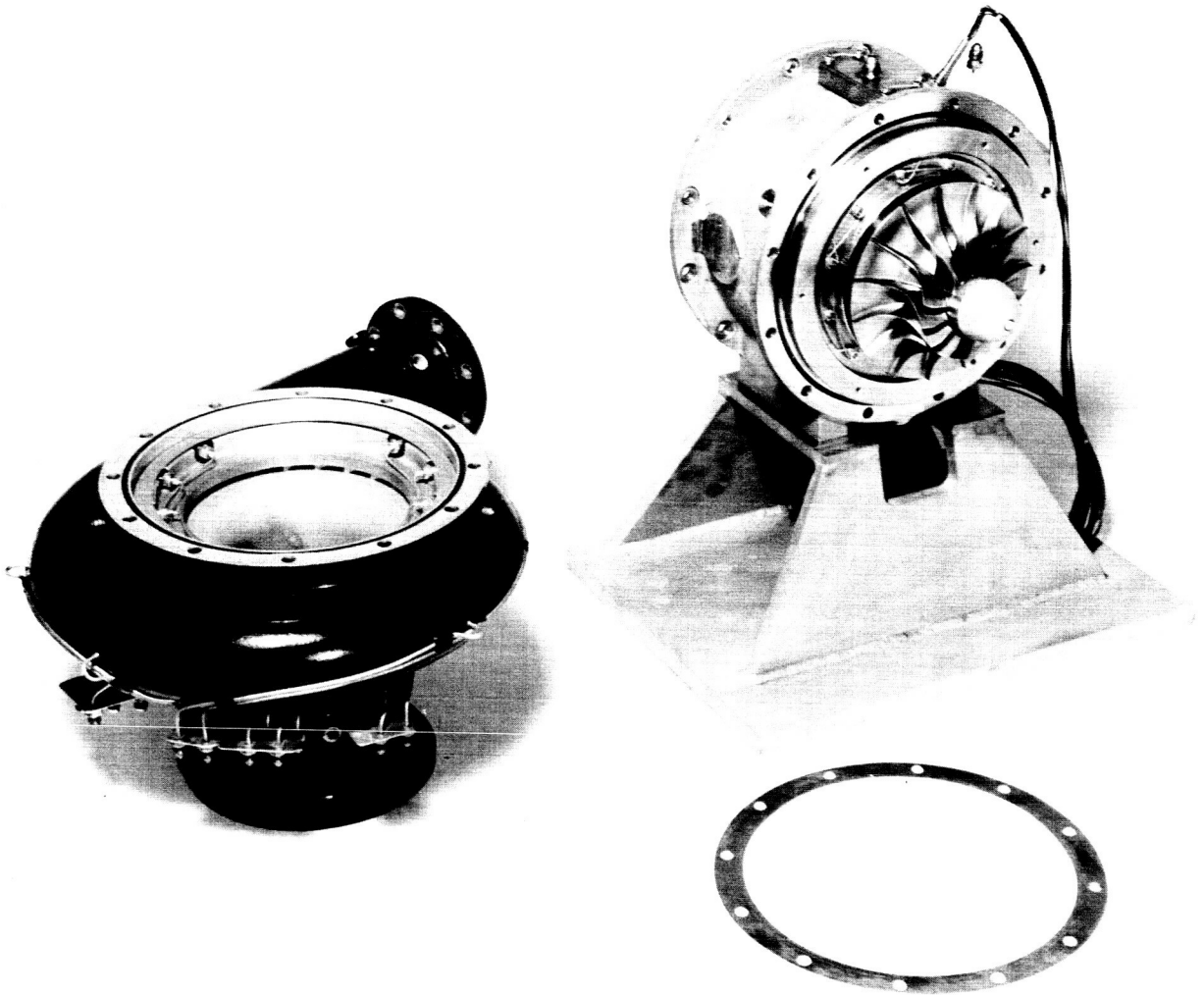
SERIAL NO. P-A
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 27



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SERIAL NO. P-B
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 23



3.4.3 Inspection

Inspection of the compressor research package parts and final assembly were conducted in accordance with the quality-assurance program as outlined in AiResearch Report RC-5130-R. Figure 29 shows the Critical Parts Inspection and Serialization Record for both compressor impellers. Figures 30 and 31 show the impeller after balance inspection. Figures 32 and 33 show both sides of the Assembly Inspection and Laboratory Traveler for the two compressor research packages.

As can be seen from Figure 33, the first shipping unit was started six times and ran a total of 6.9 hours for the acceptance test. The second unit was started three times and ran for a total of 1.2 hours for its acceptance test. Following completion of the acceptance tests, the acceptance tag was completed, as shown in Figure 34.

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A 793-A-5

FORM P2741-F													
Forging/Casting No.		C/L	Serial Number		O.S. GROWTH DATA				Machined Part No.				
*			4X 103						*369747		N/C		
Oper. Raw Matl.	Date	Insp. Stamp	Oper. Mach. Part	Date	Insp. Stamp	Control Dim/Dia	After G.R.	After O.S.	Z/M After	MRB Disp. After			
											Final	Assy	R&O
Dim.			Dim.			5.976800				USE			
Ultra			Ultra/Mch			5.9785		5.9785		RWK			
HT/Stress			HT/Stress							SCP			
Zyglo/Mag			Zyglo/Mag							RTV			
Radiogr. REISSER 291160 Heat No.			Pull Test							ITR NUMBER			
			Balance								31704	E.D	
2nd H.T.			Overspeed	MAR 18 1964									
Remarks:			Green Run							*Raw Matl. Mfg. Co.			
			2nd O.S.							*Mach. Part Mfg. No.			
			R.R. No.			Part No. Changes							

**COMPRESSOR RESEARCH PACKAGE
SERIAL NO. P-A**

H2

FORM P2741-F													
Forging/Casting No.		C/L	Serial Number		O.S. GROWTH DATA				Machined Part No.		C/L		
			4X 102						*369747		C		
Oper. Raw Matl.	Date	Insp. Stamp	Oper. Mach. Part	Date	Insp. Stamp	Control Dim/Dia	After G.R.	After O.S.	Z/M After	MRB Disp. After			
											Final	Assy	R&O
Dim.			Dim.			5.9763		5.9763		USE			
Ultra			Ultra/Mch							RWK			
HT/Stress			HT/Stress							SCP			
Zyglo/Mag			Zyglo/Mag							RTV			
Radiogr. REISSER 291160 Heat No.			Pull Test							ITR NUMBER			
			Balance								3045360		
2nd H.T.			Overspeed	MAR 18 1964									
Remarks:			Green Run							*Raw Matl. Mfg. Co.			
			2nd O.S.							*Mach. Part Mfg. No.			
			R.R. No.			Part No. Changes							

COMPRESSOR RESEARCH PACKAGE

SERIAL NO. P-B

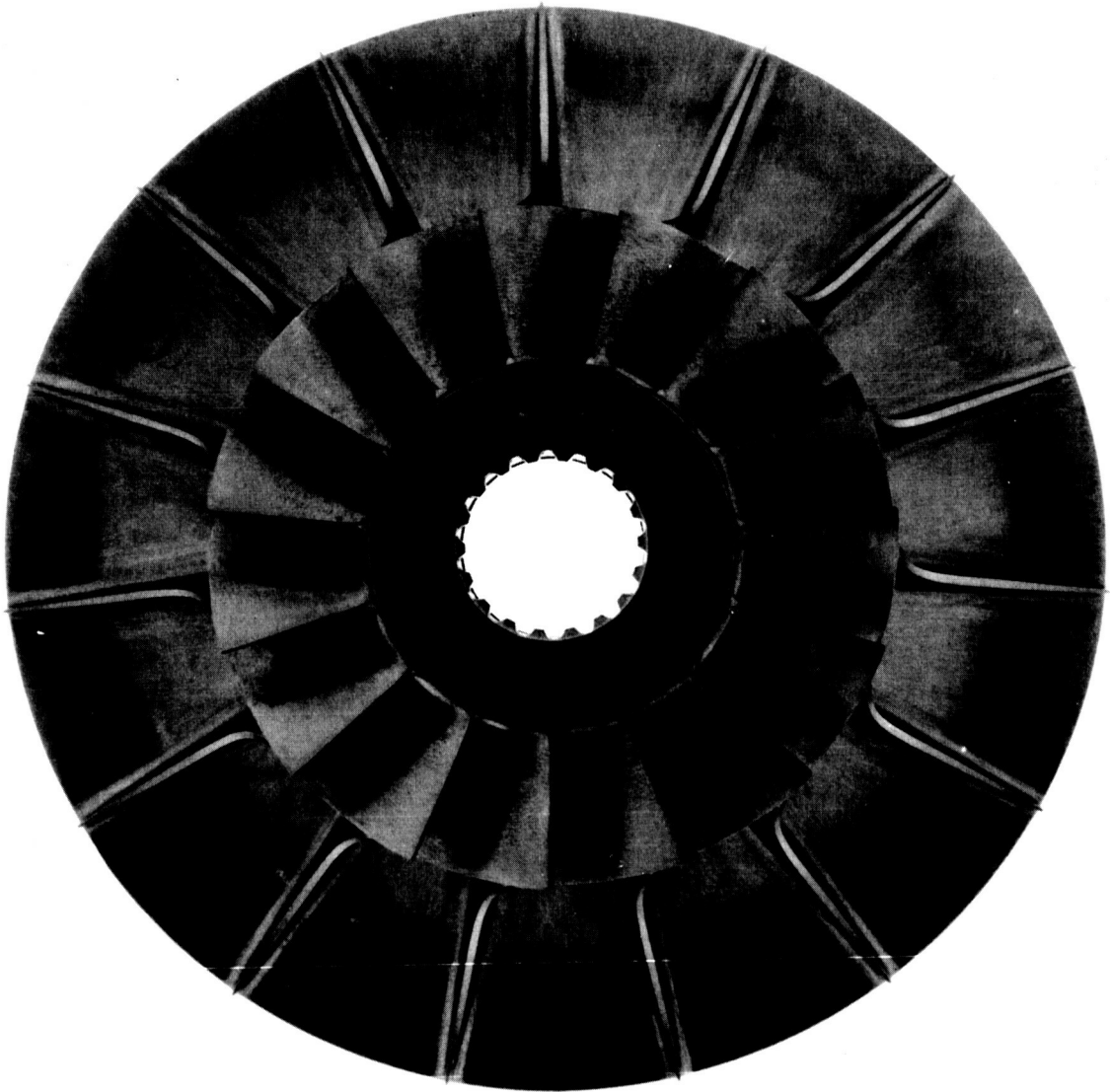
FIGURE 29

APS-5109-R

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AIRESEARCH MANUFACTURING COMPANY
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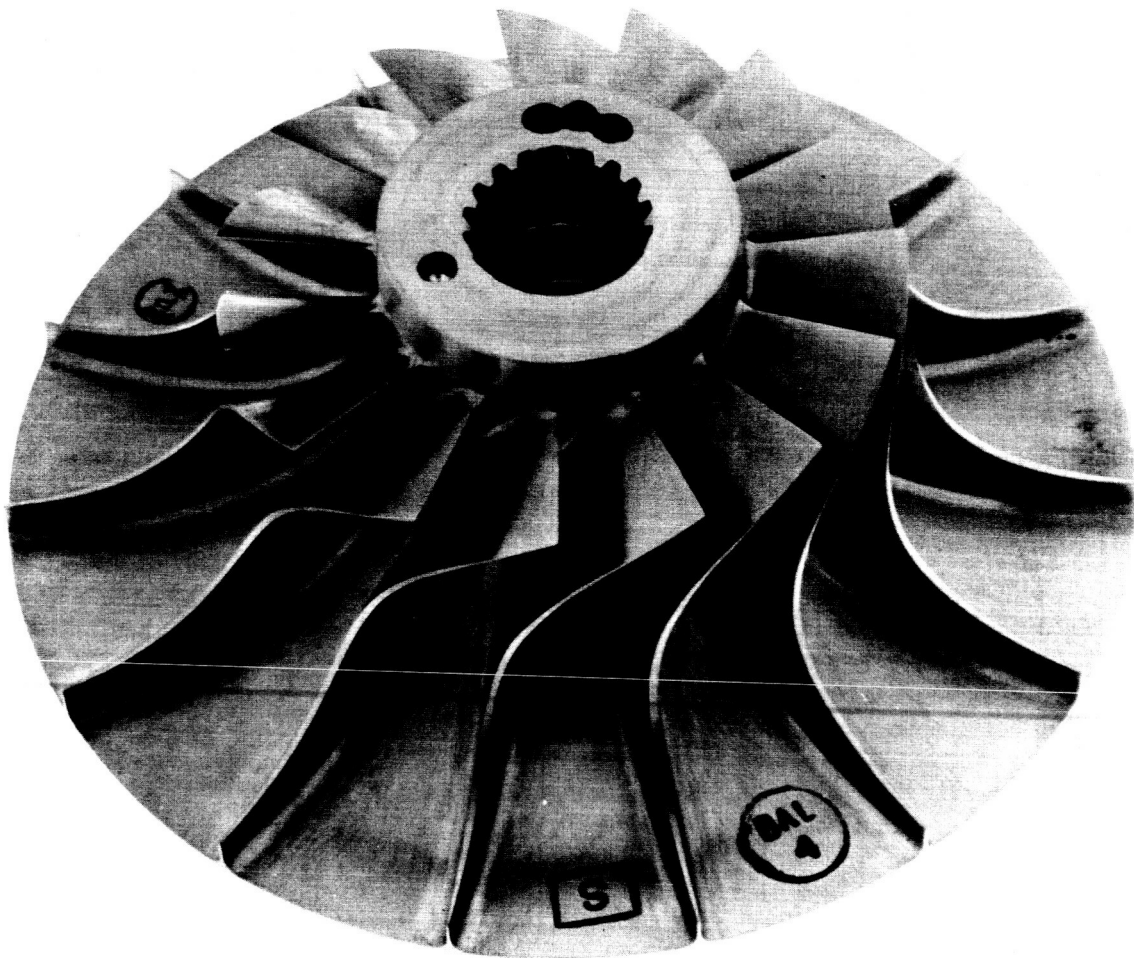
COMPRESSOR IMPELLER
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 30



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COMPRESSOR IMPELLER
NASA COMPRESSOR RESEARCH PACKAGE

FIGURE 31



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1		AIRESEARCH MANUFACTURING COMPANY OF ARIZONA		OR/C	
369739		MODEL		P-B	
COMPRESSION ASSY, NASA		CUSTOMER		NASA	
START DATE		RELEASE OR P.O. NUMBER		CONTRACT-NAS3-2778	
3409-27136-13-0501					
PRODUCTION		REPAIR		MILITARY	
M.O.T. PER ASSY RECORD NO.		W.D.T. PER ASSY RECORD NO.		ASSY RECORD NO.	
NUMBER NO.		S.O. DIMENSIONS		S.O. DIMENSIONS	
SPEC 7 0W 1041		SPEC 43 0W			
13 0W 1043					
17 0W 1045					
23 0W 300					
27 0W 1010					
33 0W 1010					
37 0W 1066					
PART NUMBER		MODEL		SERIAL NO.	
369747				4X-102	
SHIPPED 5-8-64					
O.W. all					
DATE		ASSEMBLY SIGNATURE		INSPECTION SIGNATURE	
5-7-64		O.W. all			

SERIAL NO. P-B

AIRESEARCH MANUFACTURING COMPANY OF ARIZONA		OR/C	
369739		MODEL	
COMPRESSION ASSY, NASA		CUSTOMER	
START DATE		RELEASE OR P.O. NUMBER	
3409-27136-13-0501			
PRODUCTION		REPAIR	
M.O.T. PER ASSY RECORD NO.		W.D.T. PER ASSY RECORD NO.	
NUMBER NO.		S.O. DIMENSIONS	
SPEC 7 0W 1040		SPEC 43 0W	
13 0W 1043			
17 0W 1045			
23 0W 300			
27 0W 1010			
33 0W 1010			
37 0W 1073			
PART NUMBER		MODEL	
369747			
SHIPPED 5-8-64			
O.W. all			
DATE		ASSEMBLY SIGNATURE	
		O.W. all	

SERIAL NO. P-A

COMPRESSOR RESEARCH PACKAGE
FIGURE 32



AIRESEARCH MANUFACTURING COMPANY

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PA 1372-1

AIRESEARCH MANUFACTURING COMPANY OF ARIZONA

LABORATORY TRAVELER

TRAVELER NO.

S/N

PARTS REPLACED IN LABORATORY

PART NO. SERIAL NO. REASON REPLACED

LABORATORY REJECTION REASONS

LAB SIG.
ENGINEERING REWORK RECOMMENDATIONS

ENG. SIG.

ASSEMBLY REWORK (LIST ALL PARTS EXCEPT AN 8 STD LINE PARTS)

ASSY SIG.
DEVIATION

CUSTOMER		T. I.		UNIT PRESERVED	
SINCE O'HAUL	TOTAL	FUEL	OIL		
STARTS	3				
HOURS	1.2				
SAFETY WIRED		DATE		GRAMS	
ACCEPTED	REJECTED	WT. FLOW	ACCEL. TIME	SECS.	
LAB SIGNATURE					

SERIAL NO. P-B

COMPRESSOR RESEARCH PACKAGE

FIGURE 33

PA 1372

AIRESEARCH MANUFACTURING COMPANY OF ARIZONA

LABORATORY TRAVELER

TRAVELER NO.

S/N

PARTS REPLACED IN LABORATORY

PART NO. SERIAL NO. REASON REPLACED

LABORATORY REJECTION REASONS

LAB SIG.
ENGINEERING REWORK RECOMMENDATIONS

ENG. SIG.

ASSEMBLY REWORK (LIST ALL PARTS EXCEPT AN 8 STD LINE PARTS)

ASSY SIG.
DEVIATION

CUSTOMER		T. I.		UNIT PRESERVED	
SINCE O'HAUL	TOTAL	FUEL	OIL		
STARTS	6				
HOURS	6.4				
SAFETY WIRED		DATE		GRAMS	
ACCEPTED	REJECTED	WT. FLOW	ACCEL. TIME	SECS.	
LAB SIGNATURE					

SERIAL NO. P-A

COMPRESSOR RESEARCH PACKAGE

FIGURE 33



AIRESEARCH MANUFACTURING COMPANY

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Research Manufacturing Company, of Arizona		S. O. NO.	END ITEM SERIAL NO.	SUB-UNIT CONFIGURATION			
ACCEPTANCE TAG				PART NO.	SERIAL NO.	MODEL NO.	MEO NO'S
PRODUCT NOMENCLATURE COMPRESSOR ASSY RESEARCH, NASA		27136	P-A	369747	4X-103	H. 6.7	5. 6
PART NO.			P-B	369747	4X-102	H. 1.2	5. 3
369730							
MODEL NO.							
NONE							
MEO'S							
NONE							
QTY 2	INSP. STAMP						
DATE 5-8-64	(Stamp)						
INSP. SIGNATURE							
R. SANDERSON							
ALLOCATION							
AUTHORIZED SIGNATURE							
DATE							
ENG. REC.-SPLIT							

COMPRESSOR RESEARCH PACKAGE

FIGURE 34



4.0 COMPRESSOR TESTING

4.1 Test Loop

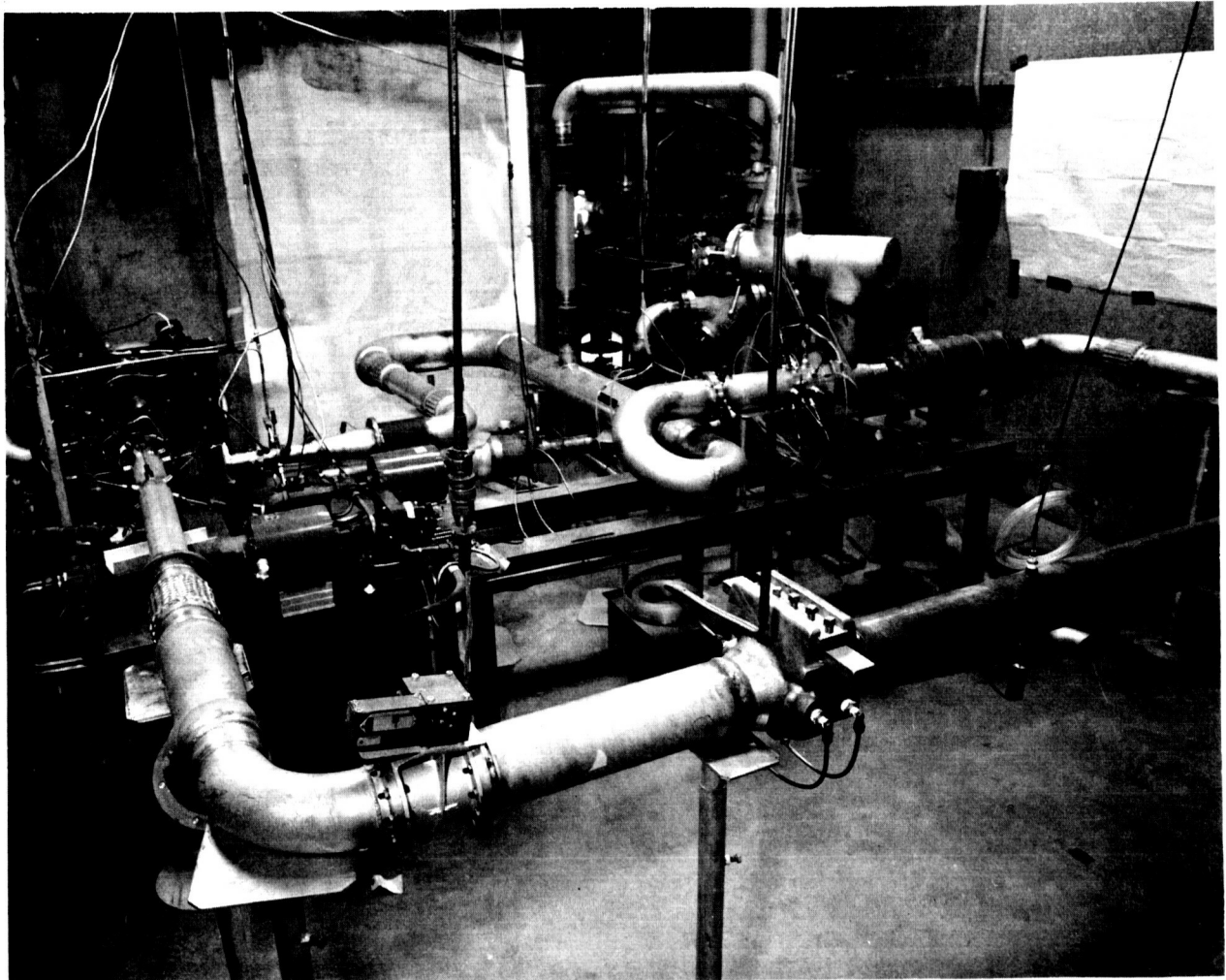
The development and shipping compressor was tested in the closed loop, shown in Figure 35, with argon used as the working fluid. The loop was fabricated from stainless steel tubing of a nominal 4-inch diameter with appropriate transitions to the inlet and discharge of the compressor and other components in the loop. To reduce the system leakage, welded joints were used throughout the system with the exception of the flanged joints at the compressor and the motorized control valve. The loop consisted of the following components:

- (a) Orifice measuring section
- (b) Motorized valve
- (c) Filter
- (d) Vacuum pump
- (e) Compressor
- (f) Drive turbine
- (g) Heat exchanger (cooler)
- (h) Cooling turbine and heat exchanger
- (i) Instrumentation

The filter was used as a precaution against the induction of any particle that might damage the impeller blades. The filter body was also the station for evacuation of the loop. The vacuum pump was used to purge the loop of air and to control the argon pressure level during the tests. (As the pressure ratio of the compressor was varied, the loop pressures would change, which would require the addition or removal of argon to maintain a constant compressor inlet pressure.)



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BRAYTON-CYCLE TURBOMACHINERY TEST LOOP

FIGURE 35



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The compressor under test was installed in the loop with flanged joints used to permit easy removal for changes of the diffuser or impeller cutback. The compressor was insulated with 1 inch of foil-backed fiber glass during all testing to facilitate accurate temperature measurement. The compressor was driven through a quill shaft by an air turbine motor, shown in Figure 36. The speed of the turbine was controlled by a pneumatically controlled valve installed in the plant air system.

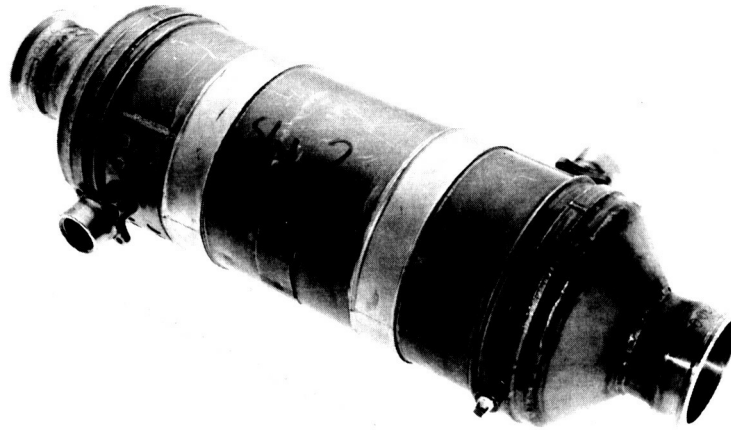
A water-cooled gas heat exchanger, shown in Figure 36, was used to control the compressor inlet gas temperature. Motorized valves on the water side of the heat exchanger permitted control of the compressor inlet temperature to within 1°F. At compressor flows and pressure ratios where full flow of the plant water supply would not control the loop gas temperature, chilled water from a second air-to-water heat exchanger was piped through the test loop heat exchanger. The water was chilled by air from an air-driven cooling turbine.

The basic compressor instrumentation consisted of thermocouples, static pressure taps, and total pressure probes located at the inlet and discharge ducts of the compressor. Special static pressure taps were installed in the inlet scroll to facilitate impeller cutback. The thermocouples were designed for maximum accuracy over the range of Reynolds numbers to be encountered in testing.

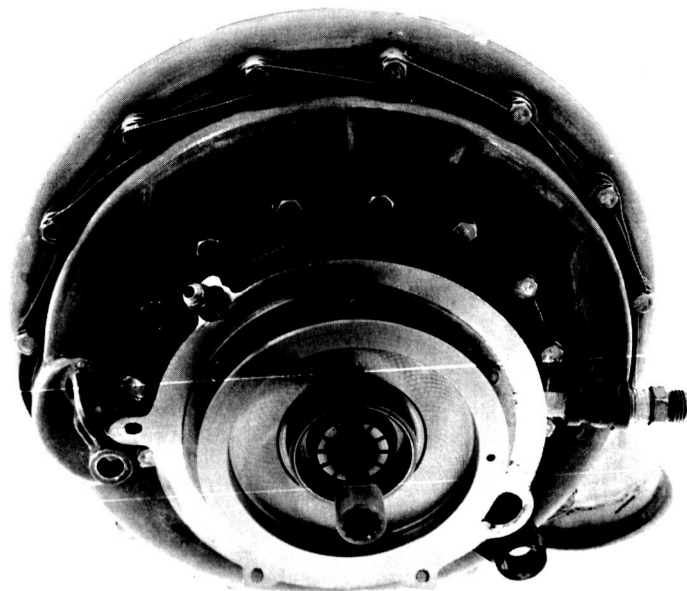


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WATER-TO-GAS HEAT EXCHANGER



AIR TURBINE MOTOR

COMPONENTS OF BRAYTON-CYCLE TURBOMACHINERY CLOSED TEST LOOP
FIGURE 36

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4.2 Development Testing

The development compressor impeller was tested with three diffusers--a nominal diffuser, a negative 3-degree diffuser, and a positive 3-degree diffuser. The compressor impeller was fabricated with an extended leading edge so that the optimum match of diffuser and impeller could be obtained. With the extended leading edge, the flow at design pressure ratio was expected to be lower than the design value. The negative 3-degree diffuser should have given the best performance with the uncut impeller.

The uncut impeller was tested with the three diffusers and the results (see Figures 37 and 38) indicate that at the design pressure ratio (2.30) and speed (38,500 rpm), the flow was about 81 percent of the design corrected flow rate (1.524) when the negative 3-degree diffuser was used. Increasing the diffuser area (using the nominal diffuser) results in a considerable increase in flow rate and a somewhat smaller additional increase in flow with the positive 3-degree diffuser. The trend shown in Figures 37 and 38 indicates that further increases in diffuser area would have a small effect, since the inducer flow limit is being approached.

In order to increase the flow, the inducer leading edge was cut back so that the blade angles were increased to the design value. The cutback wheel was tested with the nominal diffuser, and the test results showed a 5-percent deficiency in flow at the design condition. This deficiency was probably due to an under-estimation of the boundary-layer clogging at the diffuser throat. Accordingly, the positive 3-degree diffuser was tested. As shown in Figure 35, the design flow and efficiency were exceeded in this test. The efficiency at the design corrected mass flow is 0.783.

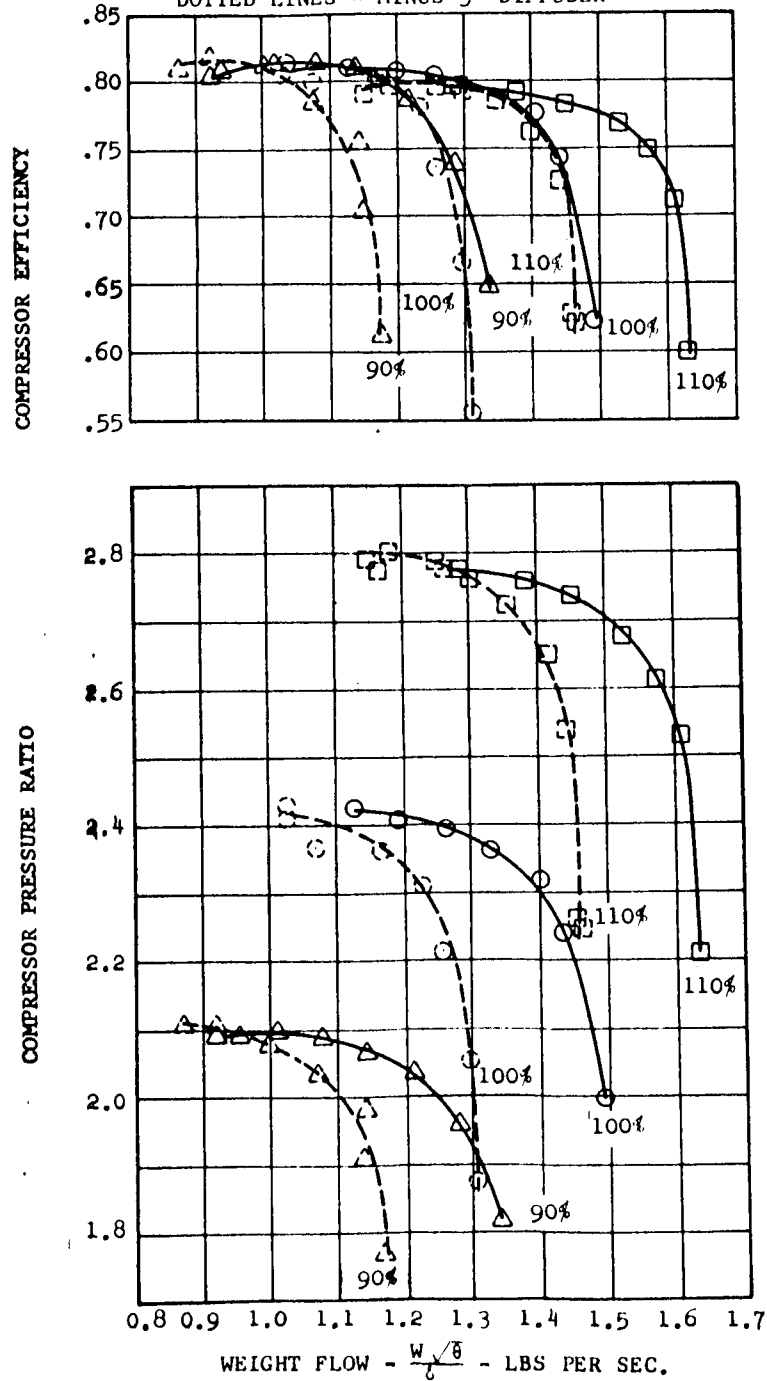


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TEST FLUID - ARGON
COMPRESSOR INLET TOTAL PRESSURE - 12.0 TO 12.5 IN. Hg ABS
COMPRESSOR INLET TOTAL TEMPERATURE - 536°R

100 PERCENT N₂/G = 37,900 RPM

SOLID LINES - NOMINAL DIFFUSER
DOTTED LINES - MINUS 3° DIFFUSER



NASA BRAYTON CYCLE COMPRESSOR
UNCUT IMPELLER, NOMINAL AND MINUS 3° DIFFUSERS

FIGURE 37

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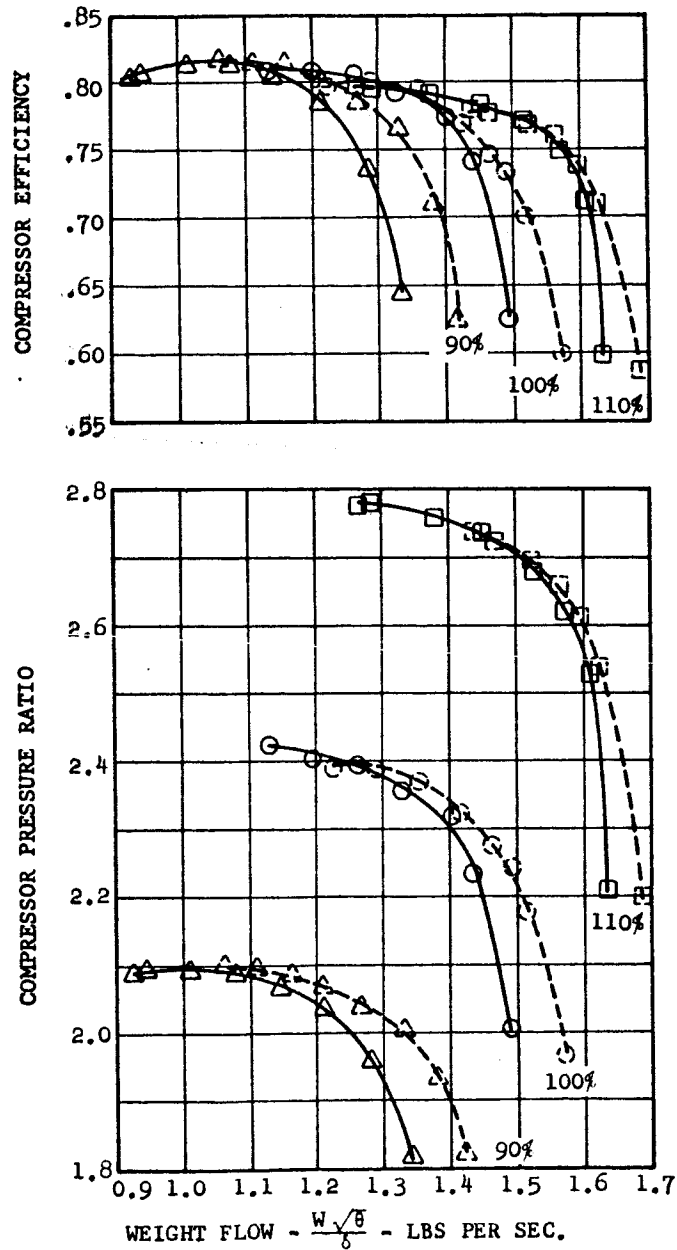
AIRESEARCH MANUFACTURING COMPANY

A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA

TEST FLUID - ARGON
COMPRESSOR INLET TOTAL PRESSURE - 12.0 TO 12.5 IN. Hg ABS
COMPRESSOR INLET TOTAL TEMPERATURE - 536°R

100 PERCENT $N/\sqrt{\theta} = 37,900$ RPM

SOLID LINES - NOMINAL DIFFUSER
DOTTED LINES - PLUS 3° DIFFUSER



NASA BRAYTON CYCLE COMPRESSOR
UNCUT IMPELLER, NOMINAL AND PLUS 3° DIFFUSERS

FIGURE 33



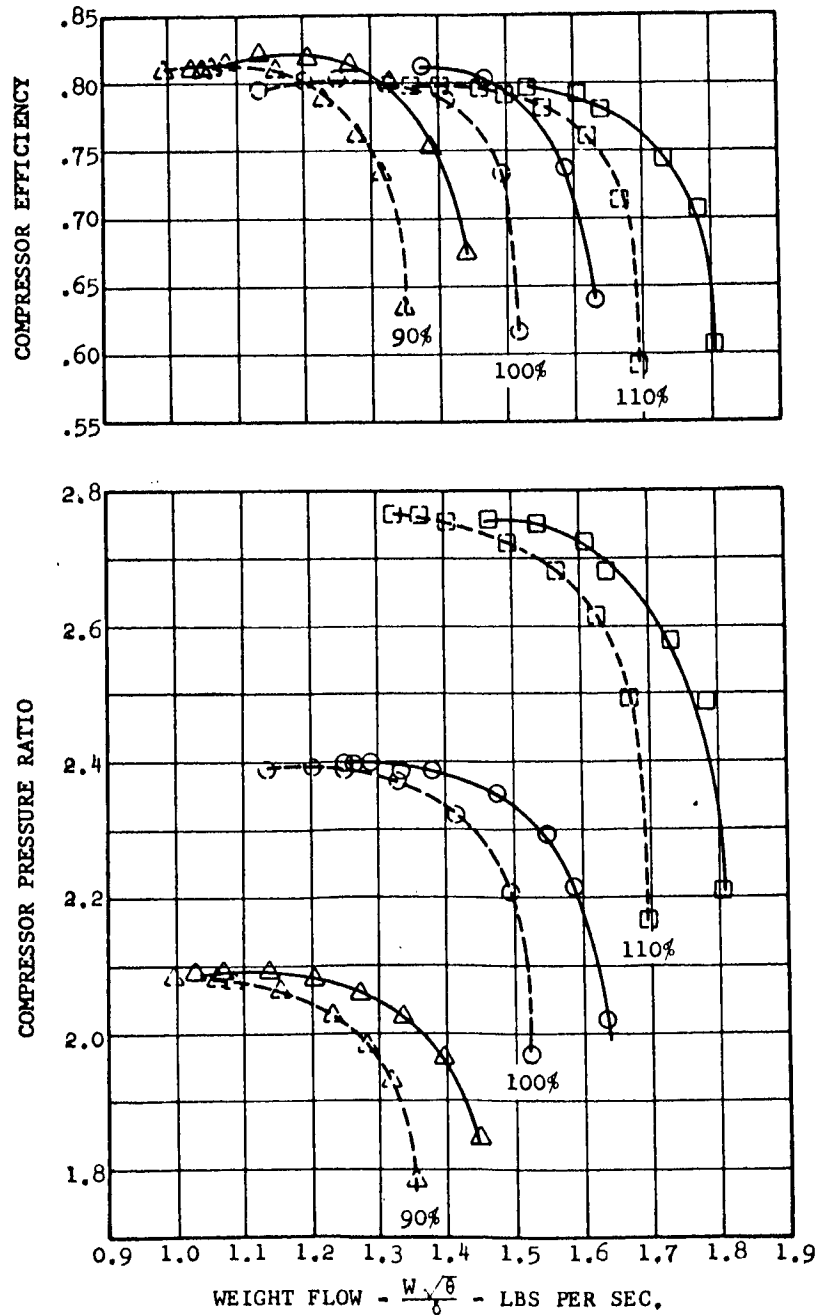
AIRESEARCH MANUFACTURING COMPANY

A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA

TEST FLUID - ARGON
COMPRESSOR INLET TOTAL PRESSURE - 12.0 TO 12.5 IN. Hg ABS
COMPRESSOR INLET TOTAL TEMPERATURE - 536°R

100 PERCENT $N/\sqrt{\theta} = 37,900$ RPM

SOLID LINES - PLUS 3° DIFFUSER
DOTTED LINES - NOMINAL DIFFUSER



NASA BRAYTON CYCLE COMPRESSOR
CUTBACK IMPELLER, NOMINAL AND PLUS 3° DIFFUSERS

A31365-1

FIGURE 39

APS-5109-R

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PHOENIX, ARIZONA

Since the peak efficiency at design speed is about 0.81, additional performance increase may be possible with further impeller cutback and/or larger diffuser.

4.3 Acceptance Testing

4.3.1 Testing Requirements

The acceptance testing requirements, established in NASA letter 1443, dated May 18, 1964, and signed by the contracting officer, John E. Dilley, are listed below:

- (a) Instrumentation shall be added to the existing bosses in the exit scroll of the No. 1 Compressor, and the unit operated in Argon over a range of pressure ratios at speeds of 90, 100, and 110 percent of design speed. Upon completing these runs and recording the test data, the unit shall be operated at 120 percent of design speed for five (5) minutes and then shutdown.
- (b) The Number 2 Compressor shall be operated for a minimum of one (1) hour, and until bearing temperatures have stabilized. Operation shall be at design speed and with any convenient cold gas. Finally, the unit shall be operated at 120 percent of design speed for five (5) minutes and then shutdown.

4.3.2 Acceptance Testing

Based on the results of the development testing cutback, the impeller for the first shipping unit (Serial No. P-A) was cut back to the same configuration. The completed



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PHOENIX, ARIZONA

compressor was fully instrumented and mapped with the positive 3-degree diffuser. Figures 26 and 27 show this unit and Figure 40 shows the resulting compressor map. The second shipping unit (Figure 28) was run for a total of 1.2 hours at the design conditions. No performance map was generated since the compressor was shipped with undrilled instrumentation bosses.

Figures 41, 42, and 43 show the acceptance test logs for both units and Figures 44 and 45 show the data sheets for the acceptance tests.



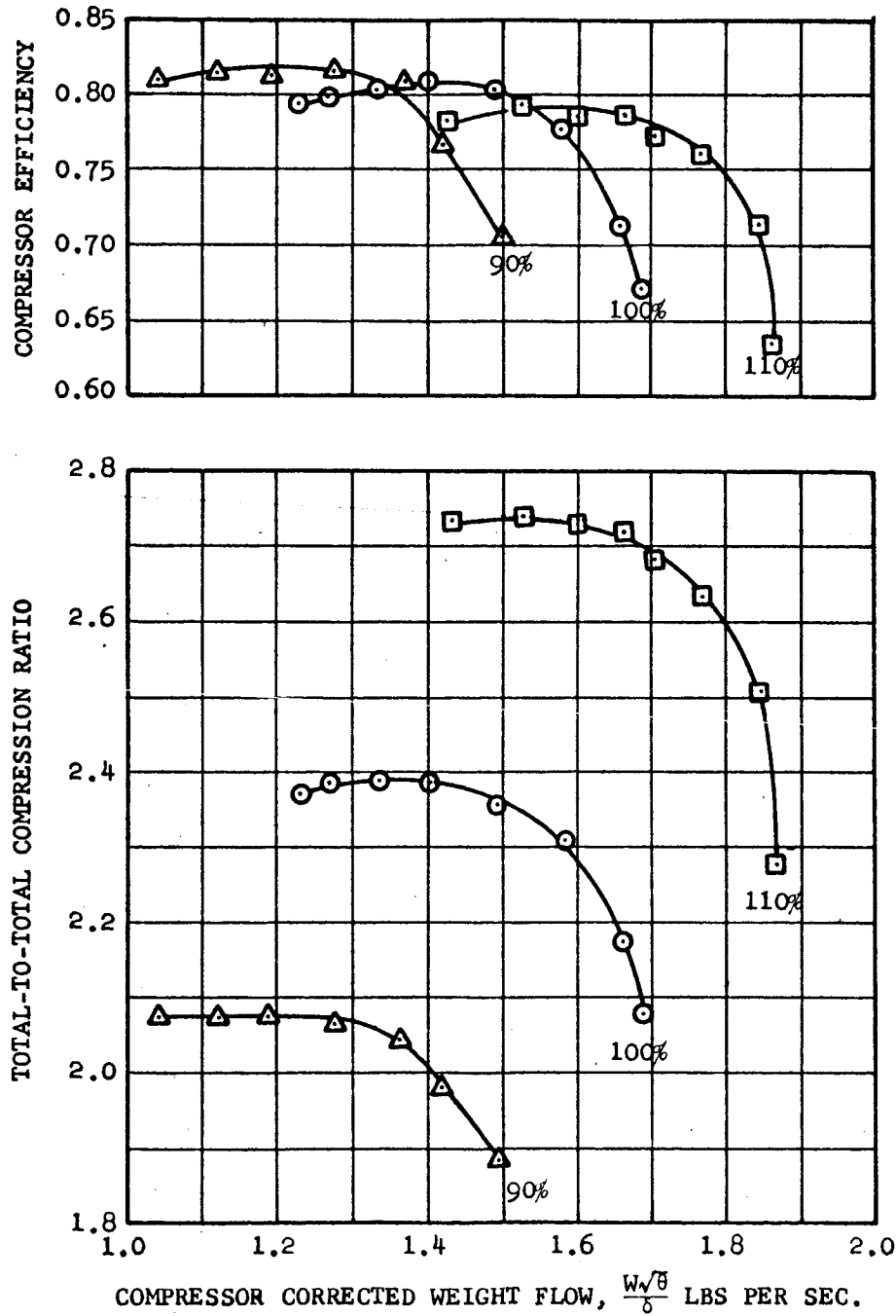
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A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA

TEST FLUID - ARGON

COMPRESSOR INLET TOTAL PRESSURE - 12.0 TO 12.5 IN. Hg ABS

COMPRESSOR INLET TOTAL TEMPERATURE - 536°R

100 PERCENT $N/\sqrt{\theta} = 37,900$ RPM



6 IN. DIA COMPRESSOR TEST

FIGURE 40

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FORM NO. P8330
800 BOOKS 2-62 AMPCO

AirResearch Manufacturing Company of Arizona

Page No. 19 of 2

QUALIFICATION TEST LOG			
E.W.O. No. 3409-27136-16-0506		Date 4-23-64	Test Cell or Station No. D-107
Assembly No.		Model No.	Unit Serial No.
Development Engineer D. McILWROU		Technician BENDER	Grp. Ldr. COLLINS
Test Type		Test Schedule	Modification
TIME START STOP	Event		O.C.
	DAYS 4-23-64		
	CLEANED GEAR. FLUSHED WITH SOLVENT THE ATM- 100-44 WHEEL ASSY USED TO DRIVE THE COMPRESSOR ON PREVIOUS RUNS, RELUBRICATED SAME WITH 7808 OIL.		
	INTERFERED WITH		
	TOTAL RUN TIME 23.3 HR		
	STARTS 26.0 STRS.		
	4-5-64 DAYS		
	MOUNTED ONE OF THE 2 COMPRESSORS THAT ARE TO BE SHIPPED ON THE PORTABLE STAND FOR AN OPEN AIR RUN, (IN P-13)		
	BEGAN INSTALLING SECOND COMP. IN LOOP (IN P-13)		
	5-5-64 DAYS COMPLETED INSTALLATION OF COMP IN (P-13) IN LOOP FILLED LOOP TO 28.5" Hg & BACK PURGED WITH ARGON 4 TIMES		
	CHANGED ARGON BOTTLES		
SUMMARY: Total Running Time _____ hrs. _____ min. Ref. Data Page _____			
Total Manual Starts _____			
Total Automatic Starts _____ Engineering _____			

TOTAL TIME ON
DEVELOPMENT
COMPRESSOR

ORIGINAL

FIGURE 41



QUALIFICATION TEST LOG			
E.W.O. No. 3409-27136-16-0505		Date 5-6-64	Test Cell or Station No. D-107
Assembly No.		Model No.	Unit Serial No.
Development Engineer D McHARRON		Technician BENDISIL	Grp. Ldr. COLLINS
Test Type		Test Schedule	Modification
TIME START STOP	Event		O.C.
1420	SET SPD @ 25000 RPM F012		
	SHORT BREAK-IN, P011012		
1430	DN. To 12 PLACE COIL & LEAD		
1512	SET SPD @ 40% (34650 RPM)		
	FOUND SURGE @ 8.2" H ₂ O		
	(P. 25.4" H ₂ A)		
	TOOK DATA COLL AB-138 PAGE 12		
1605	DN. - END OF SHIFT		
	5-6-64 DAYS		
0830	SET SPD @ (34650 RPM)		
	TOOK DATA COLL CC-66 PAGE 12		
1005	DN. TO PUMP & PURGE		
	BETWEEN SPD. LINES		
1015	SET SPD. @ 100% (38500 RPM)		
	FOUND SURGE @ 10.8" H ₂ O		
	TOOK DATA PT3 HH-KK PAGE 12		
1155	DN. FOR LUNCH		
1245	SET SPD. @ (38500 RPM)		
	TOOK DATA LINE LL=00 PAGE 12		
1320	DN. TO PUMP DOWN SYSTEM		
	& PURGE WITH ARGON		
1345	SET SPD @ (42350 RPM)		
	FOUND SURGE @ 12.3" H ₂ O		
	TOOK DATA COLL JA-HH PAGE 13		
	SET SPD @ 100% (46200 RPM)		
	FOR 5 MIN RUN TOOK DATA COLL II-JJ PAGE 13		
1545	DN		
SUMMARY: Total Running Time hrs. min. Ref. Data Page			
Total Manual Starts			
Total Automatic Starts Engineering			

START ACCEPTANCE
TEST ON UNIT
SERIAL NO. P-A

END ACCEPTANCE
TEST ON UNIT
SERIAL NO.
P-A

ORIGINAL

FIGURE 42



FORM NO. P5330
500 BOOKS 2-62 AMPCO

AiResearch Manufacturing Company of Arizona

Page No. 21 of 2

QUALIFICATION TEST LOG			
E.W.O. No. 3409-27136-16-052		Date 5-6-64	Test Cell or Station No. D-107
Assembly No.		Model No.	Unit Serial No.
Development Engineer D McKEARON		Technician BINDER	Grp. Ldr. COLLINS
Test Type		Test Schedule	Modification
TIME START STOP	Event		O.C.
	DRYS INTO SWING		
	HOOKED UP COMP. (IN P-B) & READY FOR RUN		
1745	START = 5:15 3RD @ 20,000 RPM		
	FOUND VIB. CRITICAL @ 22,000 RPM		
	SIDE 1.1415		
	BUT 1.08 INCHES		
1835	DN TO 5.15 3RD @ 20,000 RPM & INSTALL PRESSURIZATION LINES TO COMP. BEARING HOUSING.		
1745	START & 5:15 3RD @ 100% (20,000 RPM)		
	1810 BEGIN 30 MIN. QUIET RUN		
1840	DN TO CLEAN SCREEN ON COMP.		
	INLET.		
1845	START		
	1850 BEGIN		
1859	18:59 END		
	100% DATA = COLL AB = GG PAGE 14		
	120% DATA = COLL HH = JS PAGE 14		
	5-7-64 DRY		
	REMOVED COMPRESSORS IN P-A & P-B FROM CELL & DELIVERED TO DAV. ASSY.		
	CLEANED CELL.		
SUMMARY: Total Running Time _____ hrs. _____ min. Ref. Data Page _____			
Total Manual Starts _____			
Total Automatic Starts _____ Engineering _____			

START ACCEPTANCE
TEST ON UNIT
SERIAL NO. P-B

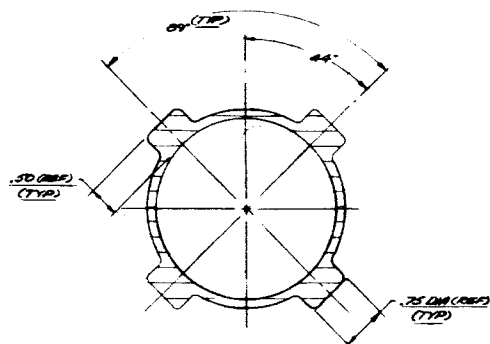
END ACCEPTANCE
TEST ON UNIT
SERIAL NO. P-B

ORIGINAL

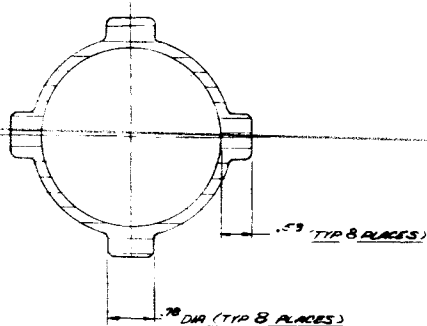
FIGURE 43

[illegible]

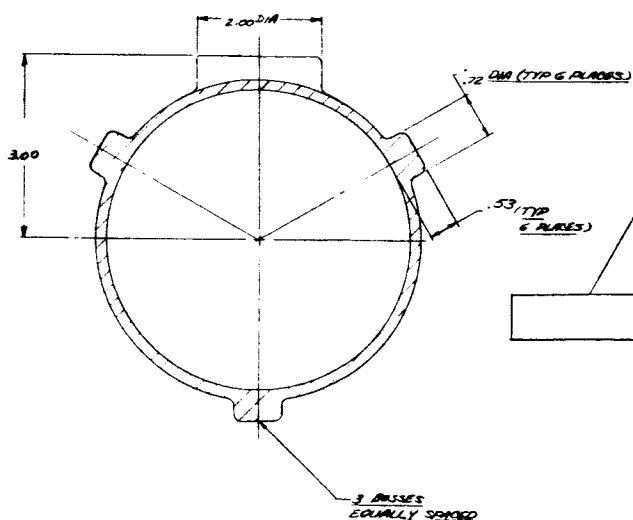
11PS-5 29-E



SECTION J-J



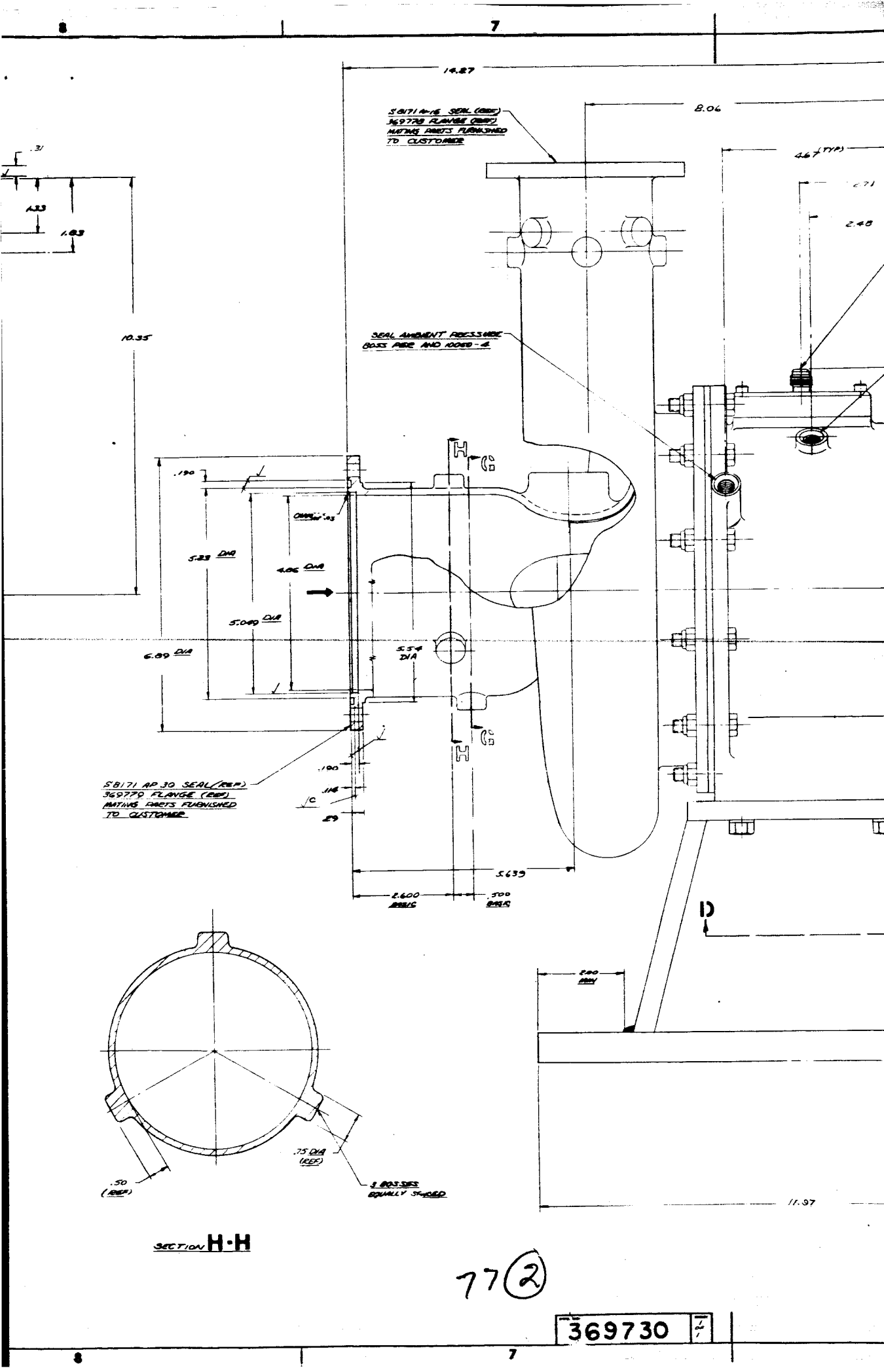
SECTION K-K



SECTION G-G

.343 DIA - 8 HOLES
EQUALLY SPACED6.200 DIA
BASIC

SEAL PURGE OUT ORBIT



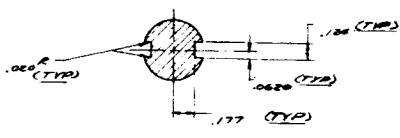
77(2)

369730

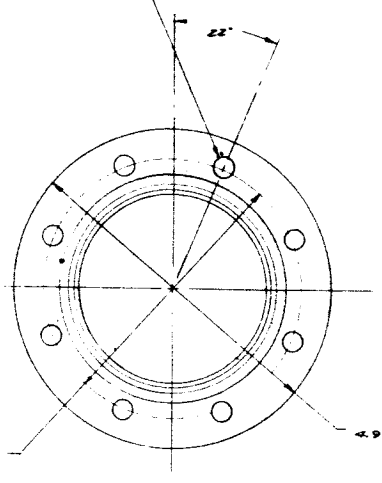
OIL NOZZLE FITTING
PER MS 93656-5

SUMP PRESSURE BOSS
PER AND 10050-6

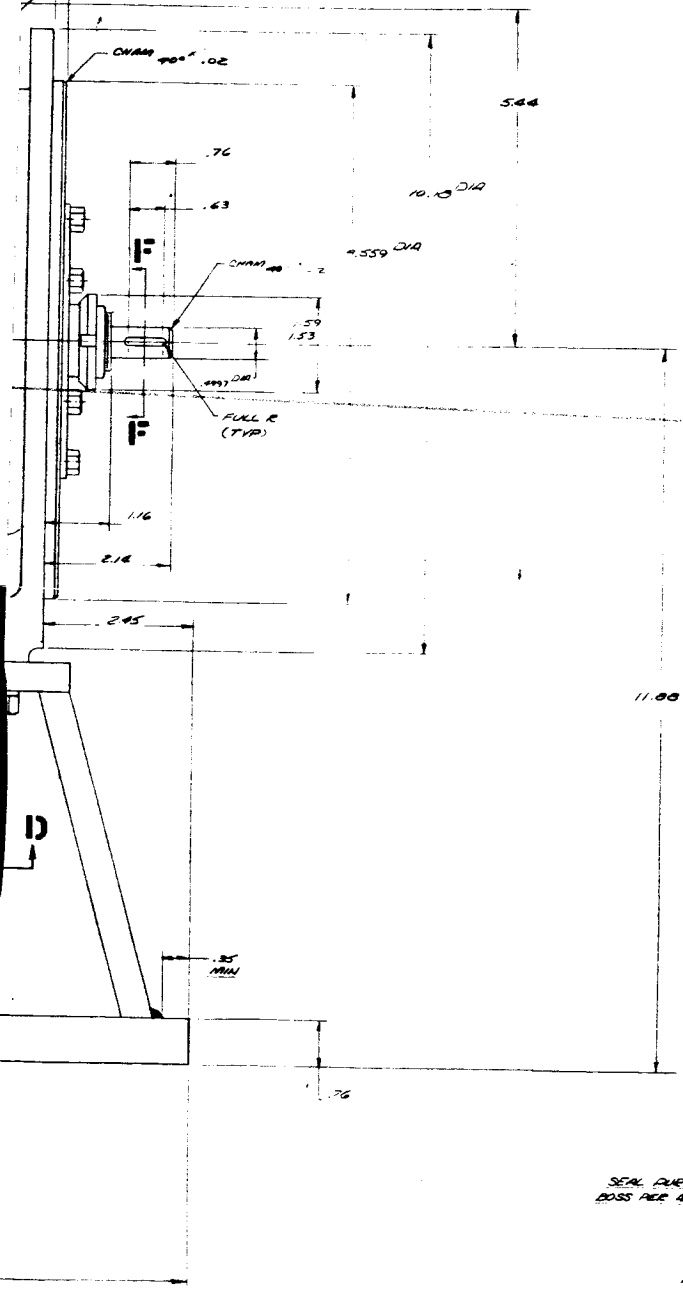
.363 DIA - 8 HOLES
SPACED ON OTHER SIDE
.64-.61 DIA X DIA NOTED
EQUALLY SPACED



SECTION E-E
TWICE SCALE

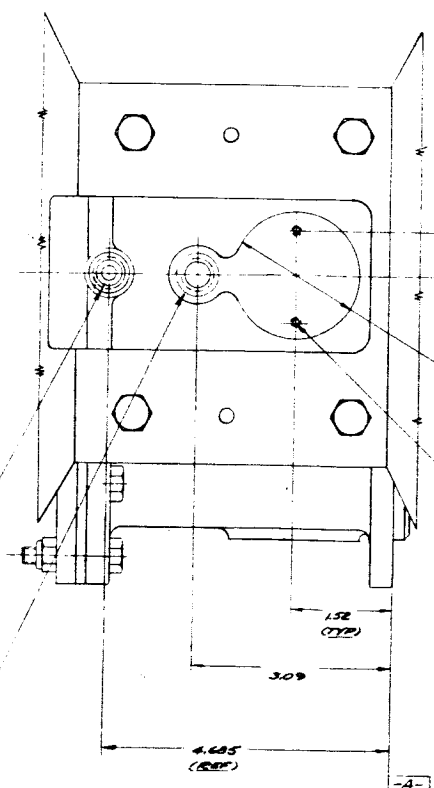


VIEW E-E



SEAL CHARGE OUT
BOSS PER AND 10050-6

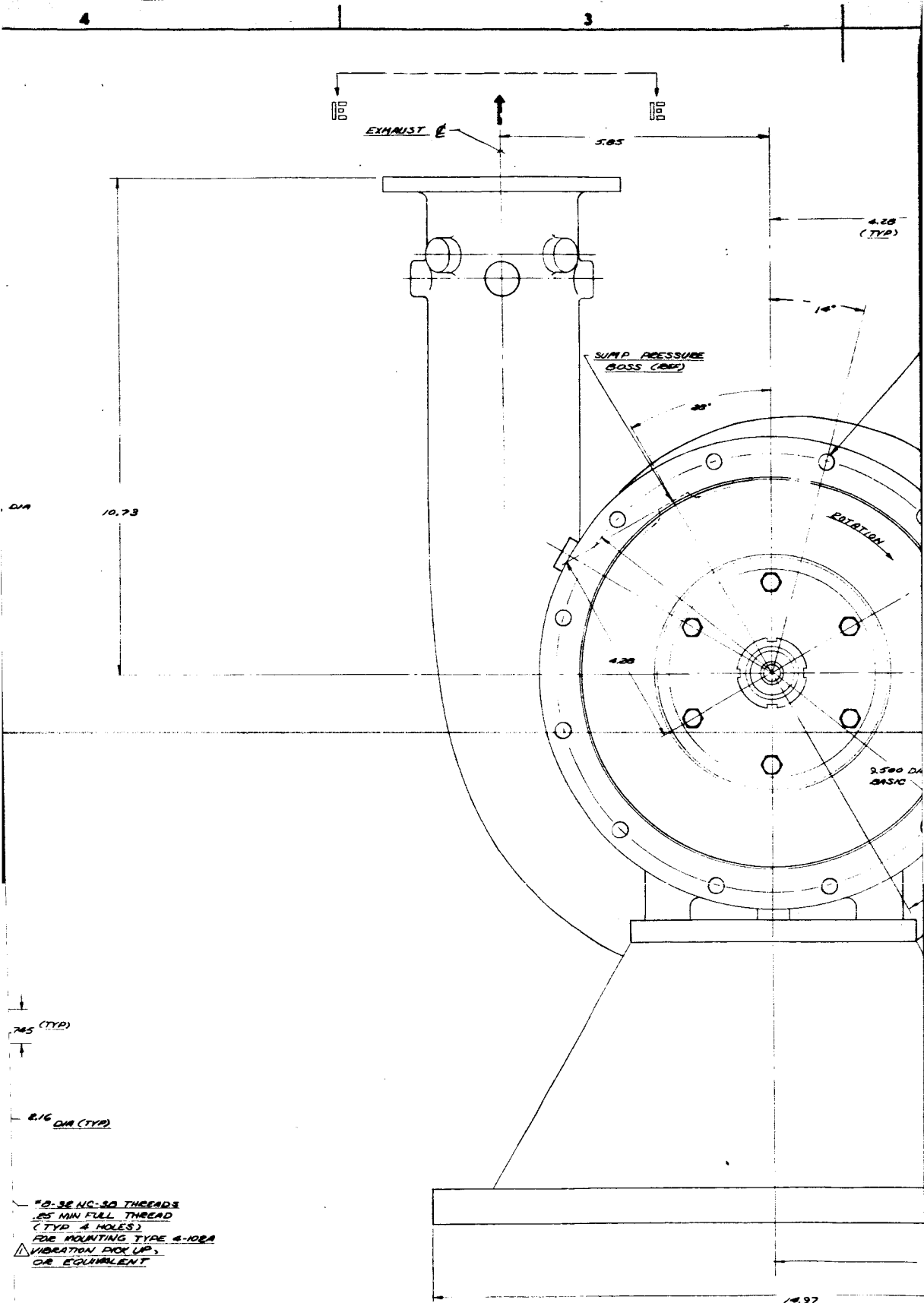
OIL OUT BOSS
PER AND 10050-6



VIEW D-D

77 (3)

-A-
(REF)



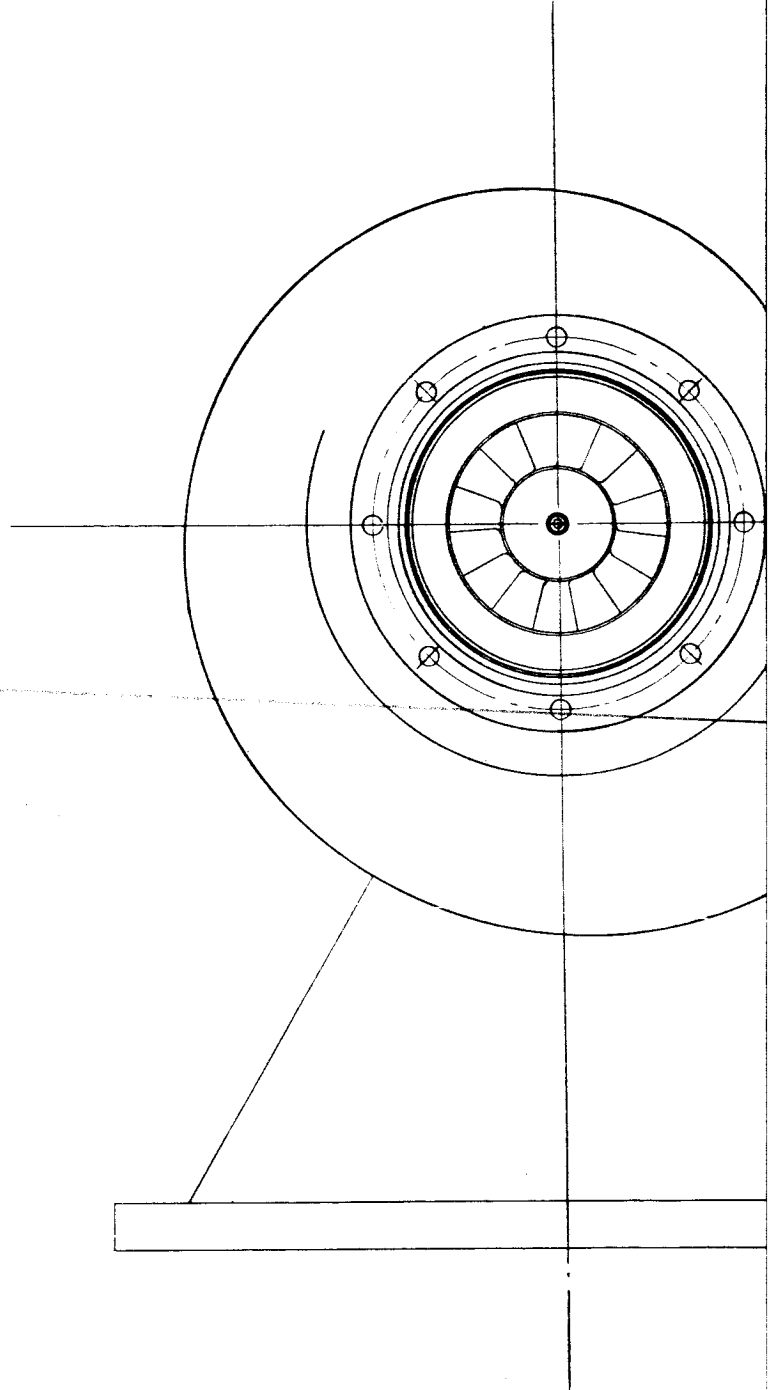
77 (4)

11. LUBRICATION REQUIREMENTS
2.00 GAL PER MIN. OF OIL AT
60-80 PSI & 150°F (MIN)
12. MAY BE PURCHASED FROM OXYN
CORPORATION, PASADENA, CA

369730 7

[illegible]

78 - ①



THESE SURFACES TO BE
PARALLEL

NOTE: REMOVE MATERIAL FROM
THIS SIDE ONLY TO BE
FLAT

NOTE: USE 10 LB THRUST IN

GRIND AT ASSEMBLY TO 0

.010 (REF) AXIAL WM

SHIM DENSE CHROME PL

DIMEN

WEEKLY AFTER PLATING

OPERATED AFTER

FEEL 369745 IN ACCO

MC-14, CLASS II (REF)

VIEW 13

369745 SPACER 1 REQ'D

369758 DIFFUSER 1 REQ'D

58171A102 O RING 1 REQ'D

MS24673-2 SCREW 12 REQ'D

LOCKWIRE PER NOTE 10

TIGHTENING TORQUE 30-25

369753 SEAL ASSY 1 REQ'D

SEE DETAIL C FOR

ALTERNATE SEAL

369733 MOUNT 1 REQ'D

369771 WASHER 1 REQ'D

525-518-9047 NUT 1 REQ'D

655-601-9208 SCREW 1 REQ'D

369770 SPINNER 1 REQ'D

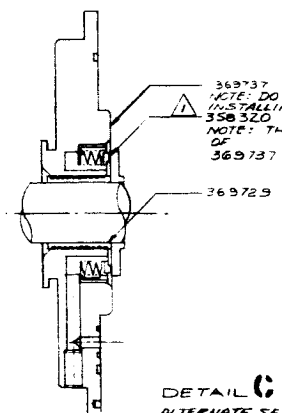
369747 IMPELLER 1 REQ'D

369754 SHAFT 1 REQ'D

362-522-9002 SEAL 2 REQ'D

369757 SCROLL ASSY 1 REQ'D

369759-3-5-7 SHIM AS REQ'D
SHIM TO OBTAIN .002 CRUSH
ON DIFFUSER VANES
4NSC-12 SCREW 12 REQ'D
4SEIDRS C5 NUT 12 REQ'D
4NS600516L WASHER 12 REQ'D
TIGHTENING TORQUE 180-20 IN LB



78-2

2 REQ'D.

4 REQ'D 369735 CARRIER 1 REQ'D

1 REQ'D

REQ'D

111917 SPRING 1 REQ'D

369734 CARRIER 1 REQ'D

369734-3-3 S. M. AS REQ'D
 TO OBTAIN .002 GAP (REF) WITH THRUST APPLIED
 IN DIRECTION OPPOSITE ARROW. THIS GAP IS TO BE DETERMINED
 BEFORE 525-577-9006 NUT (REF) IS TIGHTENED ON SHAFT.

MS20561-235 O-RING 3 REQ'D

358313 BEARING 2 REQ'D

358319 SEAL 1 REQ'D
 NOTE: THIS SEAL HAS A SHRINK
 FIT OF .002 WITH BORE IN
 369727 (REF.) SEAL CARRIER

AN3CH5A SCREW
 LOCKWIRE PER NOTE 10

6 REQ'D

369732 SPACER 1 REQ'D

369746 SHAFT 1 REQ'D
 NOTE: THIS IS A LOOSE PART.
 CARE MUST BE TAKEN TO PREVENT
 DAMAGE DURING HANDLING

525-577-9006 NUT 1 REQ'D
 TORQUE TO 300-350 IN LB.

369723 SPACER 1 REQ'D

369727 CARRIER 1 REQ'D
 NOTE: DO NOT HEAT ABOVE 400°F
 WHEN INSTALLING 358319 (REF.) SEAL
 369744-3-3 3MM AS REQ'D
 SHIM TO OBTAIN .393-.375 DIM (REF)

MS24630-2 TYPE F SCREW 2 REQ'D
 J58860 CIP NAMEPLATE 1 REQ'D

369722 HOUSING ASSY 1 REQ'D

MS16555-646 PIN (REF)

AN6CH10A SCREW 4 REQ'D
 AN360C616 WASHER 4 REQ'D
 LOCKWIRE PER NOTE 10

369752 STAND 1 REQ'D

78-④

12. TORQUE VALUES GIVEN ARE
 FRICTION TORQUE DEVELOP
 IS TO BE ADDED TO FAST
 BE DETERMINED FOR EA

11. LUBRICATE ALL THREADS &
 COMPOUND BEFORE ASSEM

10. LOCK WIRE PER MS 33540

9. VENDOR ITEM: SEE APPROPRIATE

369731

DIRECTION OF ARROW
OBTAIN
REL CLEARANCE

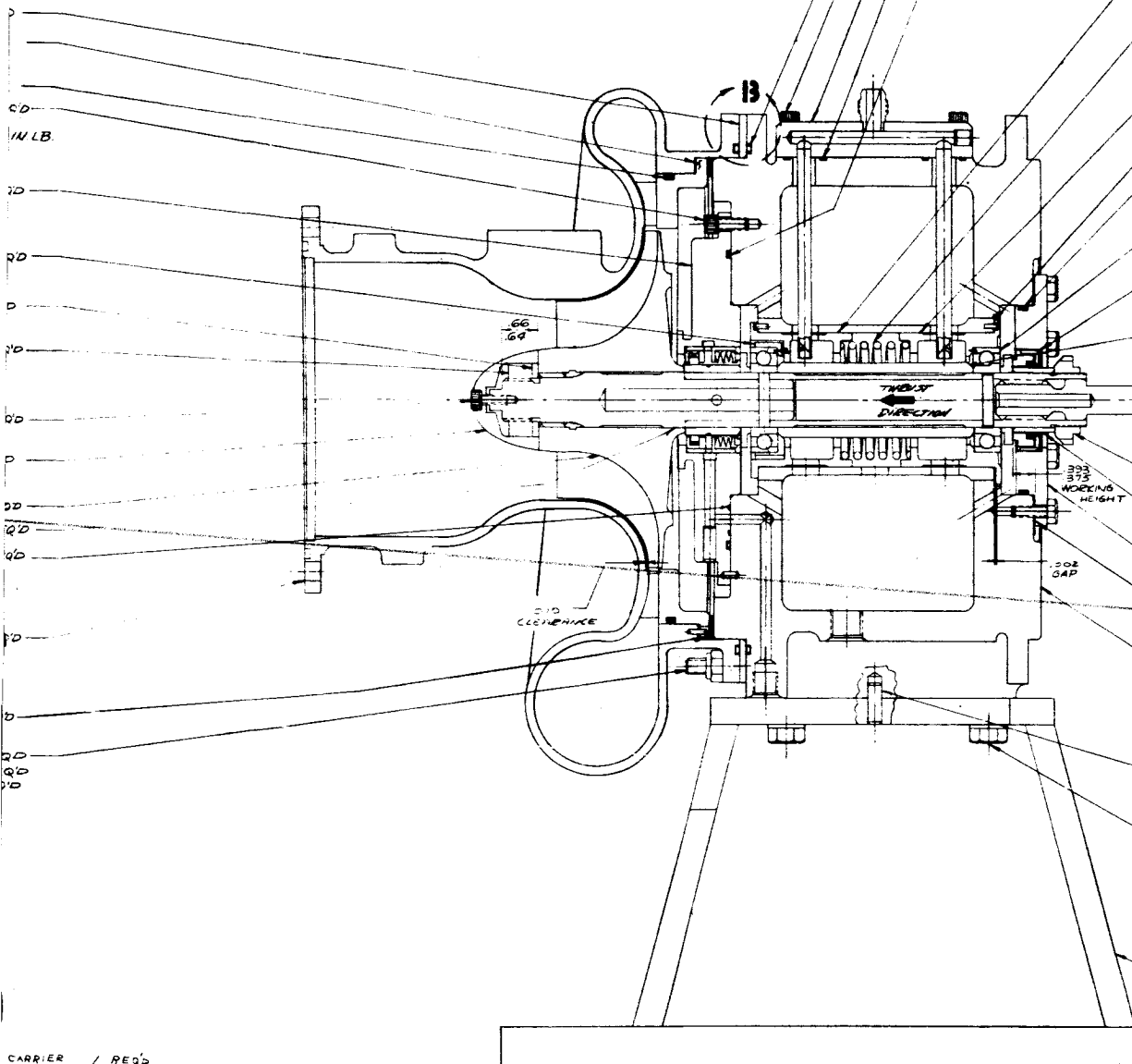
FE
SIZES & FINISH
IF PART NO. IS
PLATING, REGRIND
DANCE WITH

3608/3 SEAL
3624673-E SCREEN
LOCKWAS PER NOTE 10

360728 NOZZLE ASSY

4529581-01E O-RING

362-506-9012
SEAL 1 REQ'D

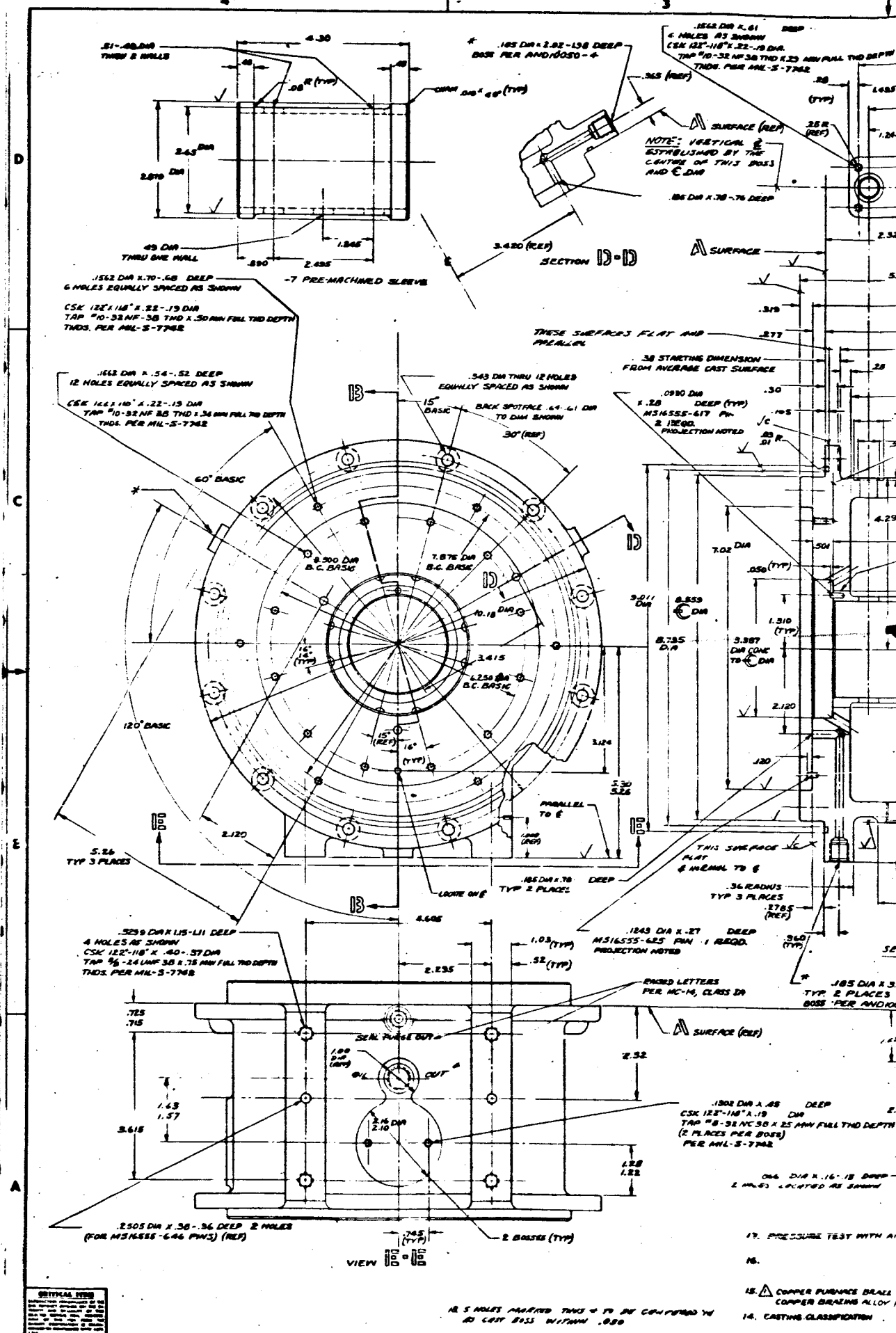


CARRIER 1 REQ'D
NOT HEAT ABOVE 500°F WHEN
362506-9012 SEAL
SEAL 1 REQ'D
5 SEAL HAS A SHRINK FIT
0005 WITH BORE IN
REF. CARRIER

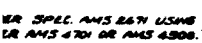
SPACE 1 REQ'D

SECTION A-A


78-3



REVISIONS		
DATE	REVISION	BY
A	SEE ENGINEERING ORDER	2-28-68
B	SEE ENGINEERING ORDER	1-10-68
C	SEE ENGINEERING ORDER	2-2-68
D	SEE ENGINEERING ORDER	1-10-68



9. MACHINED SURFACE PLAT WITHIN .005 PER INCH TO A MAX. OF .025 FOR ANY SURFACE
10. MACHINED SURFACES NORMAL OR NORMAL-TO-SPIN PER INCH TO A MAX. OF .012 FOR ANY SURFACE
11. MACHINED DIAS ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 THE UNMACHINED DIAS CONCENTRIC WITHIN .012 TB
12. DIMENSIONS LINEIT HELD AFTER PLATING
13. MACHINED FLAT RADI .010 .015
14. BREAK ALL CORNERS AND SHARP EDGES .010 .015
15. NO MACHING SURFACES ALLOWED
16. SURFACE ROUGHNESS PER MIL-STD-19
17. DIMENSIONS ARE IN INCHES UNLESS OTHERWISE SPECIFIED

1	-11	MACHINED ASSEMBLY (MAKE FROM -3)			
1	-9	BREAZED ASSEMBLY			
1	-7	PRE-MACHINED SKEWE	347 CRES	AHS 5646	
1	-5	PCE-MACHINED CASTING (MAKE FROM -3)			
1	-3	SAND CASTING	347 CRES	AHS 536.9	
QTY. ITEM REQD. IN	PART NO.	QTY.	DESCRIPTION	CODE MATERIAL	QTY. SPECIFICATION
ASSY'S			LIST OF MATERIAL		
		SIGNATURE	DATE	Housing Manufacturing Group of Office 	
3-27-68		3-27-68		FOR THE HOUSING ASSEMBLY, MAIN	
REQD. BY UNIT	UNIT QTY.	UNIT QTY.	3-27-68		
HEAT TREATMENT	PROCESS	DATE	3-27-68		
REMARKS	NOTED	DATE	3-27-68		
DATE	DATE	DATE	3-27-68		
			501093 E SCALE 1/4" = 1"	367922 SHEET 1 OF 1	

APS-5706-R
APS-5109-R

THESE SURFACES NORMAL TO
€ DIA
PARALLEL TO EACH OTHER

20 DIA
CONE TO .5 DIA

.985
€ DIA

PART NO. HERE PER
MC 14, CLASS II

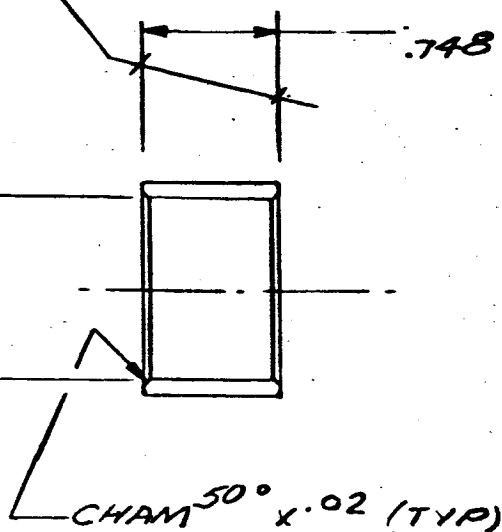
11. MAGNETIC INSPECTION PER MIL-I-6868
10.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
 5. DIMENSION LIMITS HELD AFTER PLATING.
 4. MACHINED FILLET RADII .030 - .015.
 3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

QTY. REQD.	ITEM NO.	PART NO.	SY
		← ASSYS	
1	369741	369740	
1	369731	369730	
1	369721	369720	
REQD.	NEXT ASSY.	USED ON	
HEAT TREATMENT		PROCESS	
HARDNESS		NAME	
Pc 40-46			
SPEC.		SPEC.	

REVISIONS

SYM	DESCRIPTION	DATE	APPROVED



CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE INTEGRITY AND RELIABILITY OF THIS SELECTED CRITICAL ITEM. PROCUREMENT OF THIS ITEM FROM THE GARRETT CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPR 1.313.

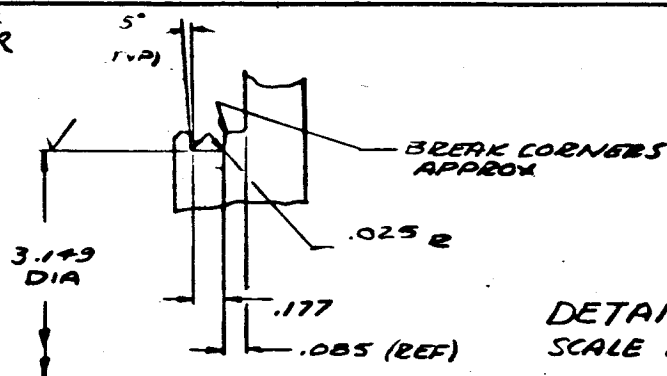
SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
			4340	AMS 6415	

LIST OF MATERIAL

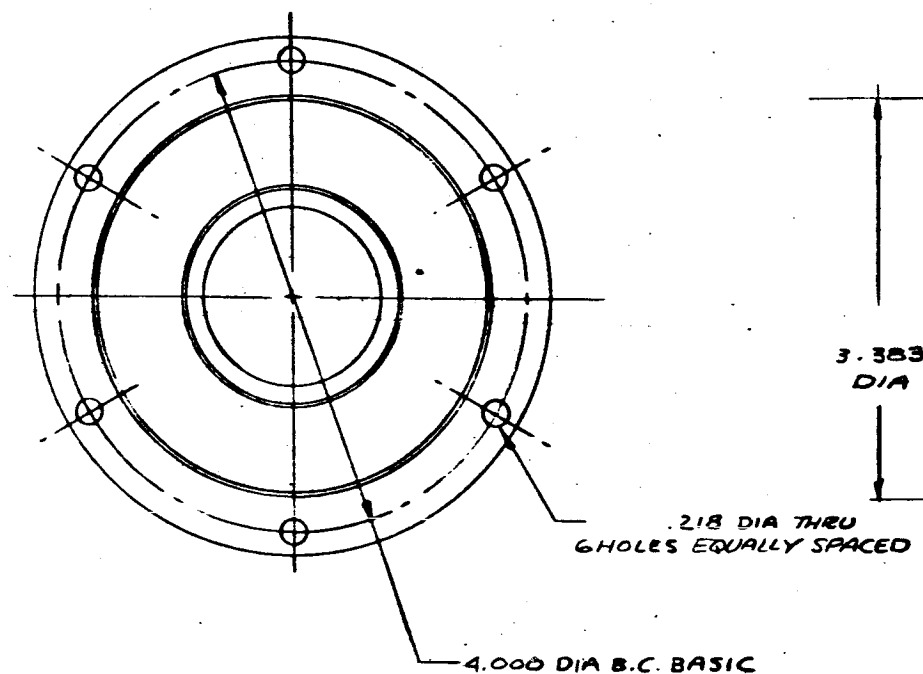
SIGNATURES		DATES		AirResearch Manufacturing Company of Arizona PHOENIX, ARIZONA		A Division of THE GARRETT CORPORATION	
MFG. ENG. <i>[Signature]</i>		8-24-63		DWG. TITLE <h1>SPACER</h1>			
MFG. ENG. <i>[Signature]</i>		8-24-63					
MFG. ENG. <i>[Signature]</i>		9-25-63					
MFG. ENG. <i>[Signature]</i>		9-25-63					
MFG. ENG. <i>[Signature]</i>		9-26-63					
DESIGN ACTIVITY APP.		9-26-63		CODE IDENT NO.	SIZE	DWG. NO.	
<i>[Signature]</i>				99193	B	369723	
OTHER ACTIVITY APP.				SCALE FULL		WT. SHEET 1 OF 1	

80(2)

AP3-5108-R
AP3-5109-R



DETAIL 10
SCALE 2/1



CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE
THIS PRODUCT DEPENDS ON THE DI-
Tegrity AND RELIABILITY OF THIS
SELECTED CRITICAL ITEM. PROCE-
MENT OF THIS ITEM FROM THE
GARRETT CORPORATION IS RECOM-
MENDED IN COMPLIANCE WITH ASPL
1.312.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR; UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
 5. DIMENSION LIMITS HELD AFTER PLATING.
 4. MACHINED FILLET RADII .030 - .015
 3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

11.

10. POSITIONAL & GEOMETRICAL
TOLERANCE SYMBOLS PER MIL-STD-8

9.

REVISIONS		
SYM	DESCRIPTION	DATE APPROVED
A	SEE ENGR ORDER	11-12-63 <i>[Signature]</i>
B	SEE ENGR ORDER	1-14-64 <i>[Signature]</i>

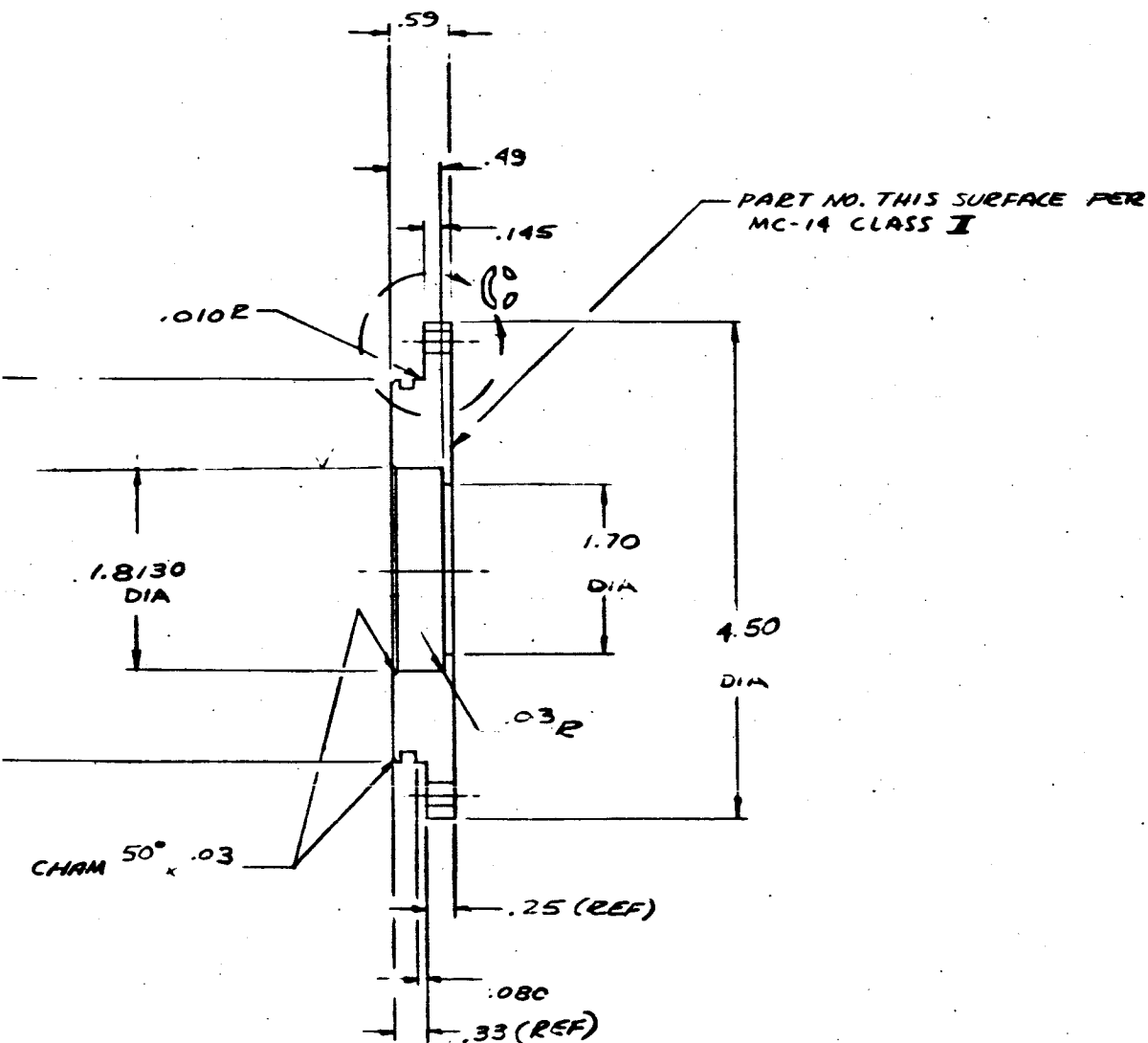


PLATE	CRES 347	MIL-S-6721
TY. REQD.	ITEM NO.	PART NO.
SYM	DESCRIPTION	CODE IDENT
MATERIAL	SPECIFICATION	UNIT WT.

← ASSYS		LIST OF MATERIAL	
SIGNATURES		DATES	
OFFER <i>[Signature]</i>		7-19-63	
CHK <i>[Signature]</i>		11-1-63	
MFG ENG			
MAT & PROCESS <i>[Signature]</i>		11-1-63	
STRESS <i>[Signature]</i>		11-1-63	
AIRC			
APP <i>[Signature]</i>		10-1-63	
APP			
DESIGN ACTIVITY <i>[Signature]</i>		10-9-63	
OTHER ACTIVITY APP.			

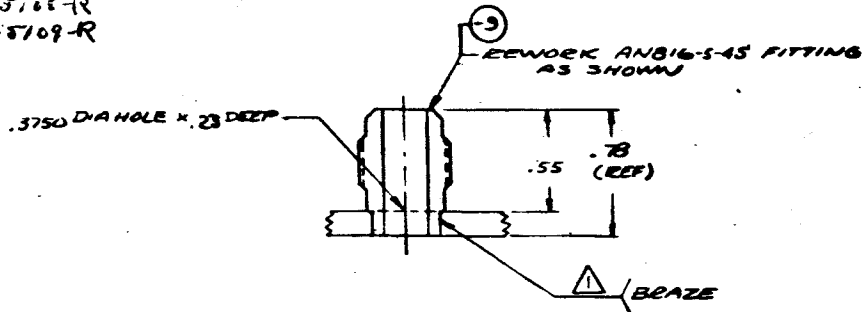
ADVICE		NAME	
VOTED			
IC		SPEC	
HT-38			

CODE IDENT NO.	SIZE	DWG. NO.
99193	C	369727
SCALE FULL	WT.	SHEET / OF 1

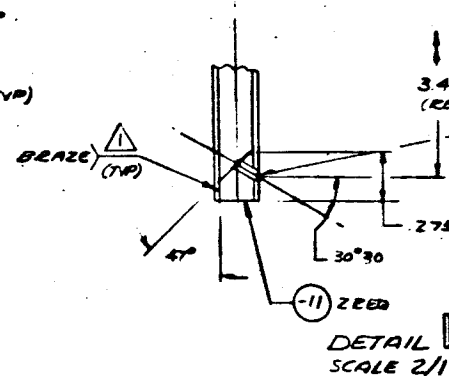
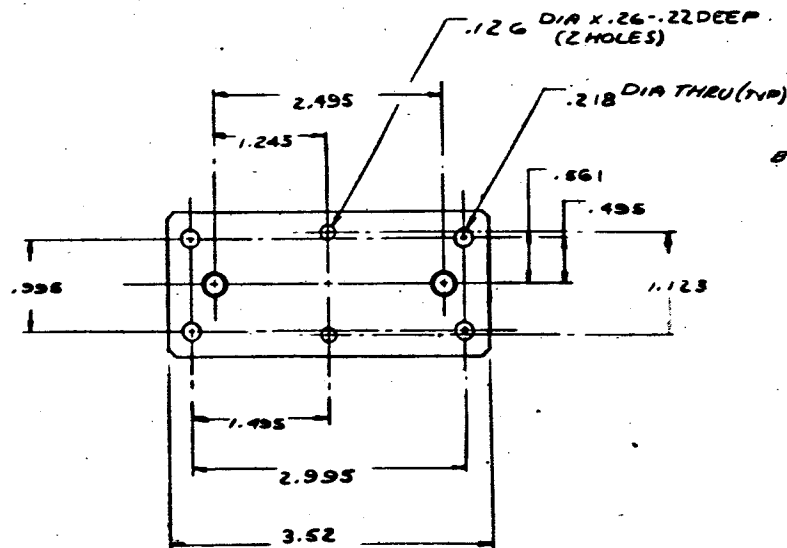
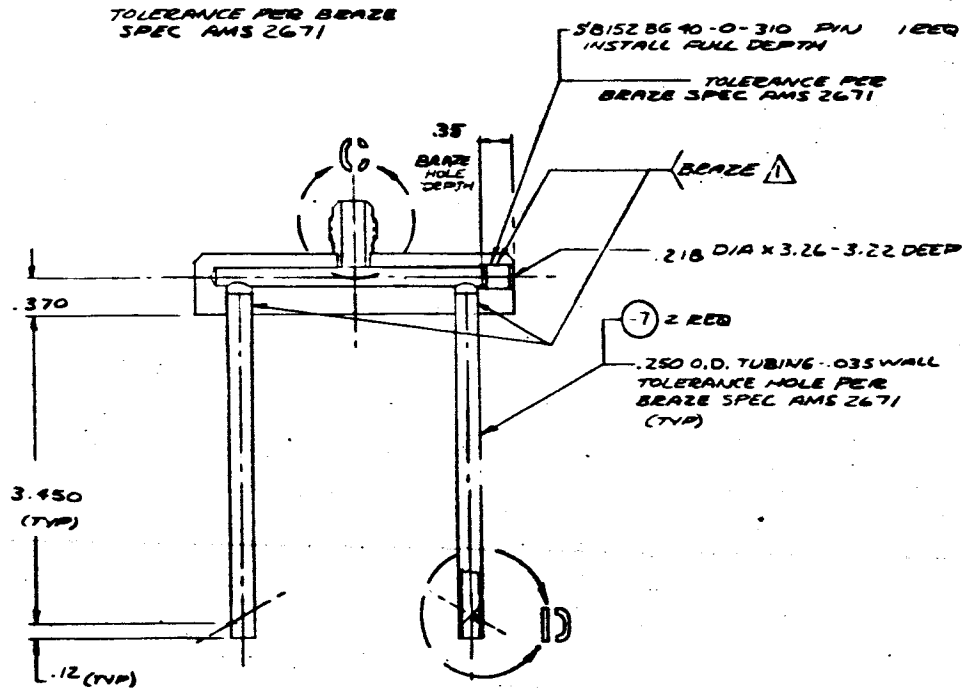
LAST
B
LTP
369727

81(2)

APS-5165-R
APS-5169-R



DETAIL 10
SCALE 2/1

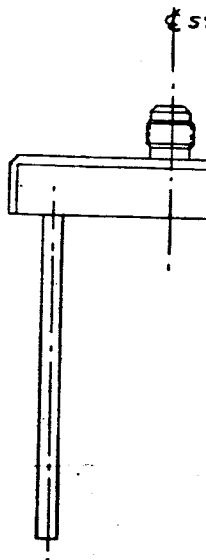


CRITICAL ITEM
SATISFACTORY PERFORMANCE OF THE
BMS PRODUCT DEPENDS ON THE SA-
TISFACTORY AND RELIABILITY OF THE
BMS CRITICAL ITEM. PROVIDE-
MENT OF THE ITEM FROM THE
BMS CORPORATION IS RECO-
MMENDED IN COMPLIANCE WITH AMS.

PS-40-0100 R00 100 HENCULBMS

11. PRESSURE TEST WITH OIL AT 150 PSIG FOR
ONE MINUTE.
△ 10. COPPER BRAZE PER AMS 2671

82 ①



(REP)

- | | | | |
|-----------------|--|----------|----------|
| | | | |
| 2 | | | - 11 |
| 1 | | | - 9 |
| 2 | | | - 7 |
| 1 | | | - 5 |
| | | | - 3 |
| QTY. REQD. | | ITEM NO. | PART NO. |
| ← ASSYS | | | |
| | | | |
| | | | |
| 369741 | | 369790 | |
| 369781 | | 369730 | |
| 1 369721 | | 369720 | |
| REQD NEXT ASSY. | | USED ON | |
| HEAT TREATMENT | | PROCESS | |
| FURNACE | | FURNACE | |
| SPEC. | | SPEC. | |

2

82

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED

IN (EXCEPT .218 DIA (REF) HOLE AS SHOWN)

PART NO. HERE PER MC 19, CLASS 2

	ORFICE PLUG		CRES 347	QQ-S-763	
	CONNECTOR				
	TUBE		CRES 347		
	BODY		CRES 347	ML-S-6721	
	BEADED ASSY				
SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	WT. LBS.

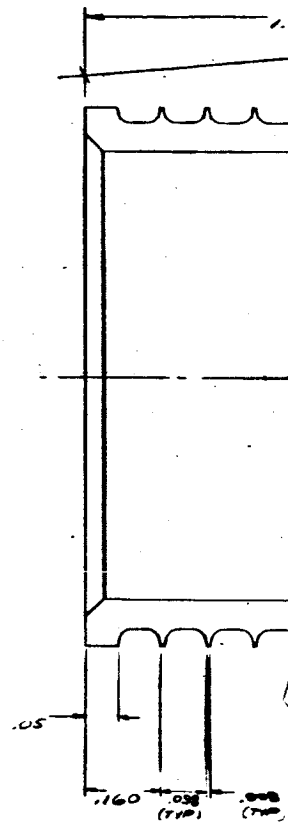
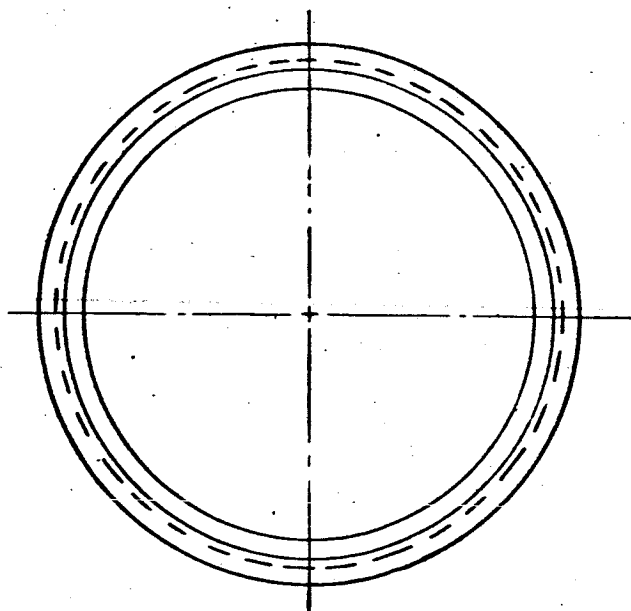
LIST OF MATERIAL

SIGNATURES		DATES		ADDRESS: Manufacturing Company of Address PHONE: 1-800-XXX-XXXX	
[Signature]		9-19-63		NOZZLE ASSEMBLY, OIL	
[Signature]		9-20-63			
[Signature]		9-25-63			
[Signature]		9-27-63			
[Signature]		9-26-63			
DESIGN ACTIVITY APP.		9-26-63		CODE IDENT NO. 99193	DES. NO. D 369728
OTHER ACTIVITY APP.				SCALE: FULL WT.	SET 1, 3, 5

82 3

369728

APS-5105-R
APS-5109-R



CRITICAL ITEM

EXTRINSIC PERFORMANCE OF THE
END PRODUCT DEPENDS ON THE QUALITY
AND QUANTITY OF THE
CRITICAL ITEM. EXCESSIVE
VARIATION OF THE ITEM FROM THE
DESIGN SPECIFICATION IS DISAPPROVED
IN COMPLIANCE WITH AMS
1.013.

11. PENETRANT INSPECT PER MIL-I-6866
10. PART NUMBERING: NONE

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED

CH OTHER

CONCENTRIC TO C DIA

DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	LIST WT.
		NS-25		

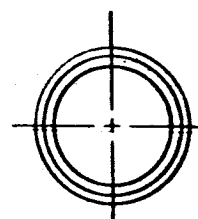
LIST OF MATERIAL	
DATE 7-18-63 7-25-63 7-25-63 7-25-63 7-26-63	PART TITLE SPACER, LABYRYNTH
CODE IDENT NO. 99193	SIZE D
SCALE 3/1	WT.
SHEET 1 OF 1	369729

369729

Q 3 (D) 3

APS-5108-R
APS-5109-R

PART NO HERE PER
MC 14, CLASS II



.585
DIA E

CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE
END PRODUCT DEPENDS ON THE IN-
TEGRITY AND RELIABILITY OF THIS
SELECTED CRITICAL ITEM. PRODU-
MENT OF THIS ITEM FROM THE
GARRETT CORPORATION IS RECOM-
MENDED IN COMPLIANCE WITH AMSPL
1.313.

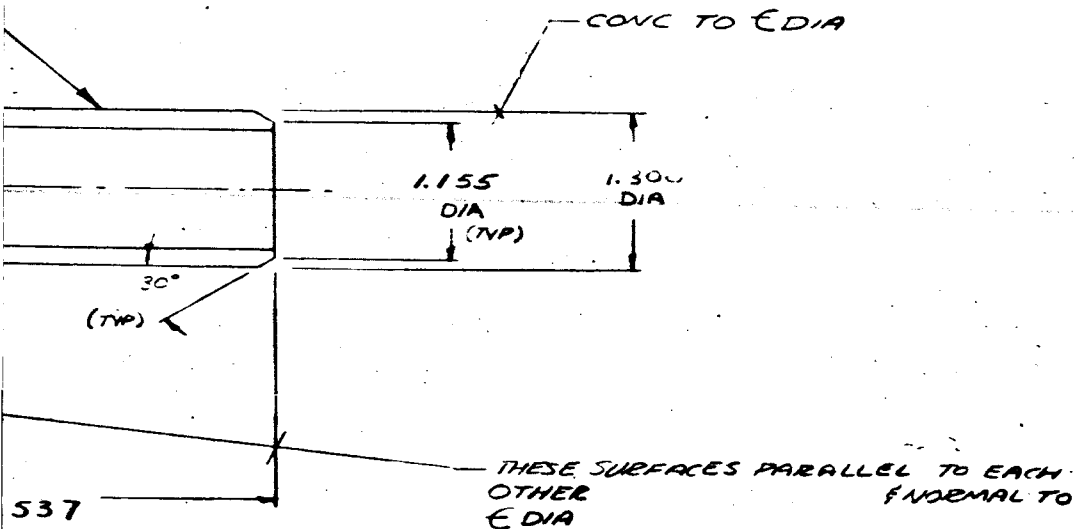
8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.
7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC
WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITH-
IN .032 TIR.
5. DIMENSION LIMITS HELD AFTER PLATING.
4. MACHINED FILLET RADIUS .030 - .015
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.
2. SURFACE ROUGHNESS PER MIL-STD-10.
1. DIMENSIONS ARE IN INCHES
UNLESS OTHERWISE SPECIFIED.

17. MAGNETIC INSPECTION PER MIL-I-6868
18. SIMILAR TO 369539

9

840

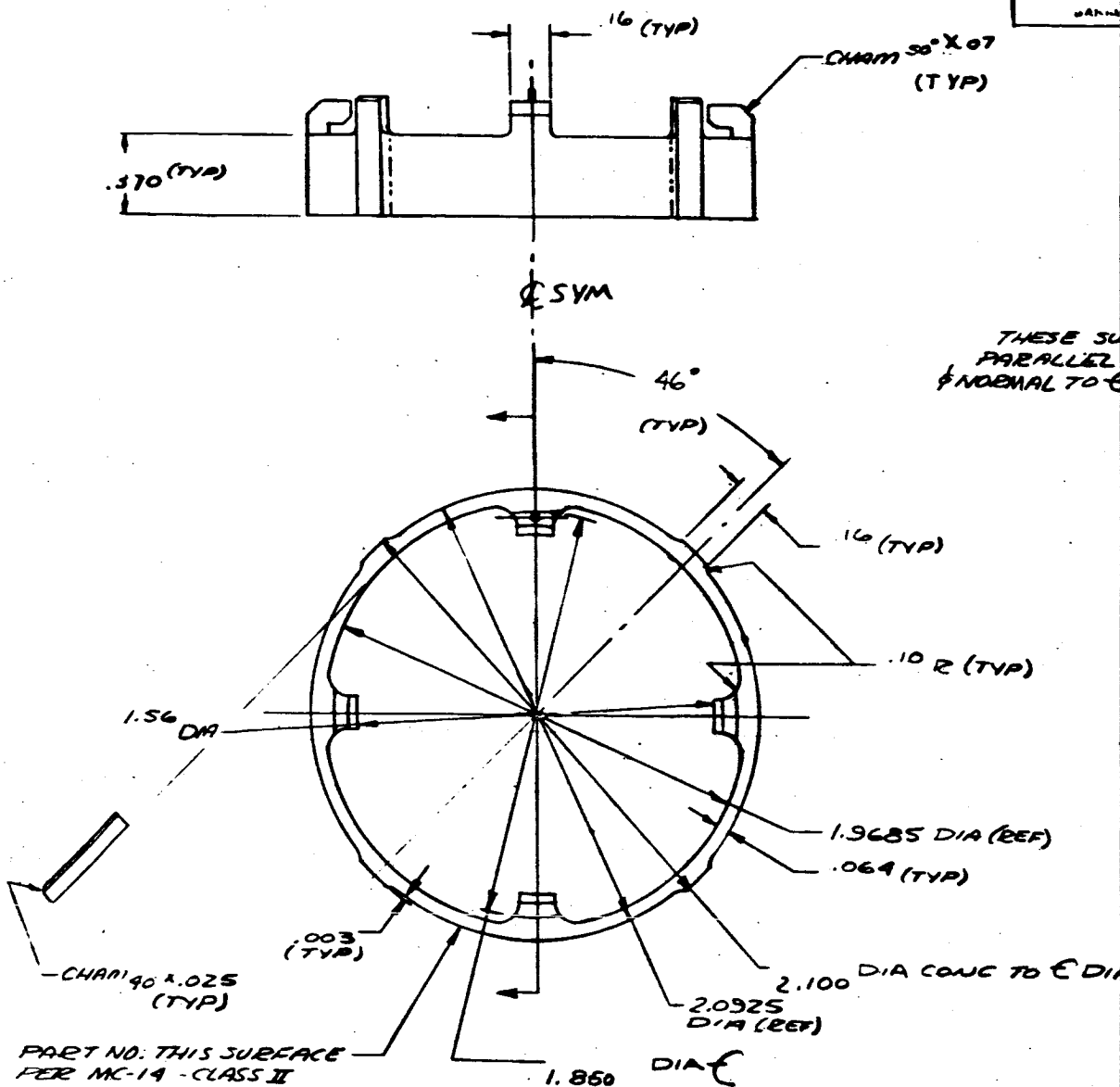
REVISIONS		
SYM	DESCRIPTION	DATE APPROVED



Y. REQD.		ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
					BAE OR TUBING		STEEL 4340	AMS 6415	
← ASSYS LIST OF MATERIAL									
SIGNATURES		DATES		ARResearch Manufacturing Company of Arizona PHOENIX, ARIZONA					
DR. [Signature]		9-16-63		DWG. TITLE SPACER, BEARING					
CHK. [Signature]		9-25-63							
MFG. ENG.									
MAT. & PROCESS		9-25-63							
STRESS		9-25-63							
AERO		9-26-63		CODE IDENT NO. 99193 SIZE C DWG. NO. 369732					
APP. [Signature]									
DESIGN ACTIVITY APP.		9-26-63							
OTHER ACTIVITY APP.				SCALE FULL WT. SHEET 1 OF 1					

84 (2)

APS-5101
APS-5101-R



CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE INTEGRITY AND RELIABILITY OF THIS SELECTED CRITICAL ITEM. PROCUREMENT OF THIS ITEM FROM THE GARRETT CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPL 1.312.

12. CONCENTRICITY SHOWN NEED NOT BE CHECKED IN FREE STATE.

11. MAGNETIC INSPECTION PER MIL-I-6868

10. SIMILAR TO PA 369572

9.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.

7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.

6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.

5. DIMENSION LIMITS HELD AFTER PLATING.

4. MACHINED FILLET RADII .030 - .015

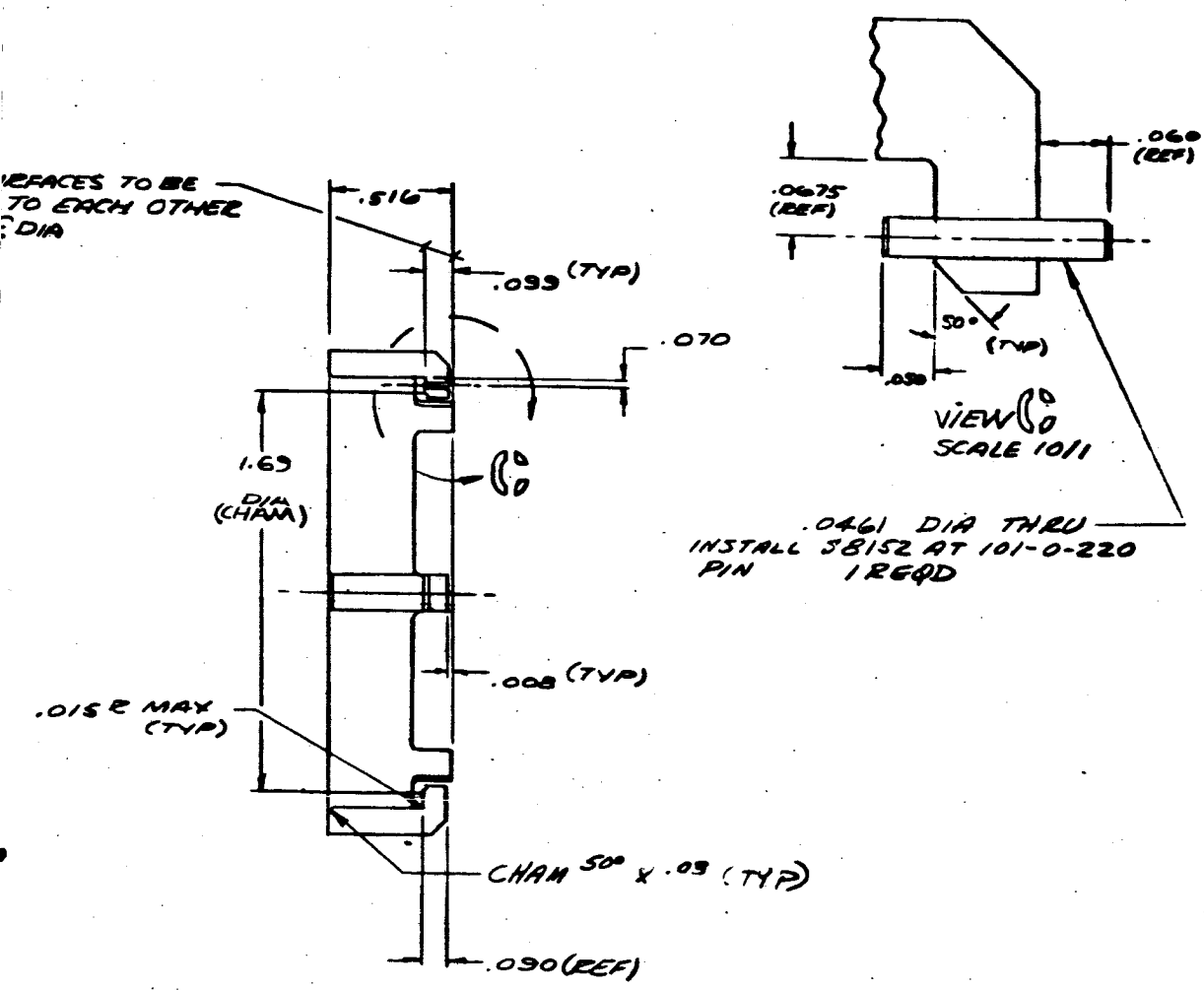
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.

2. SURFACE ROUGHNESS PER MIL-STD-10.

1. DIMENSIONS ARE IN INCHES

UNLESS OTHERWISE SPECIFIED.

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED

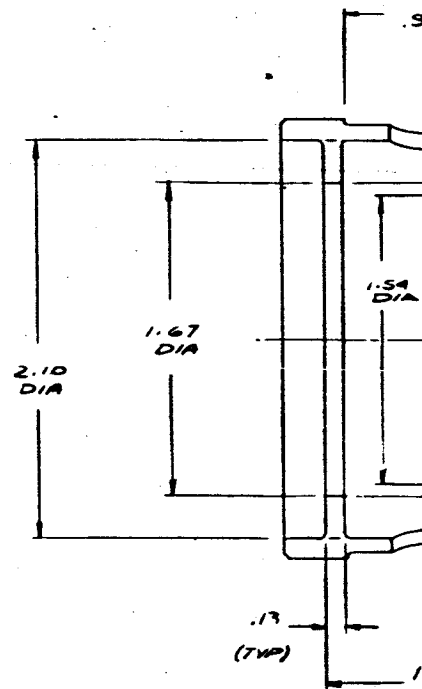
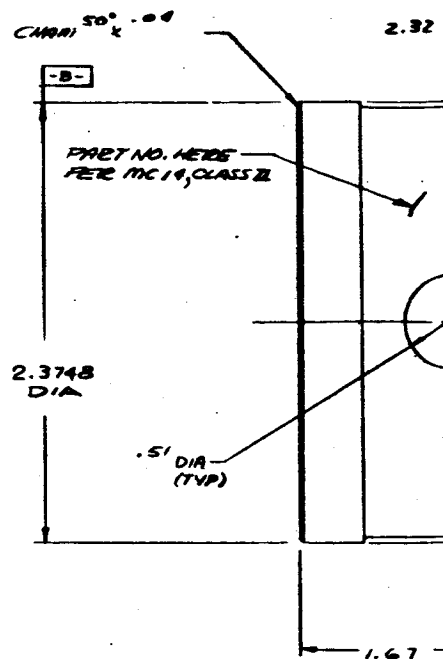
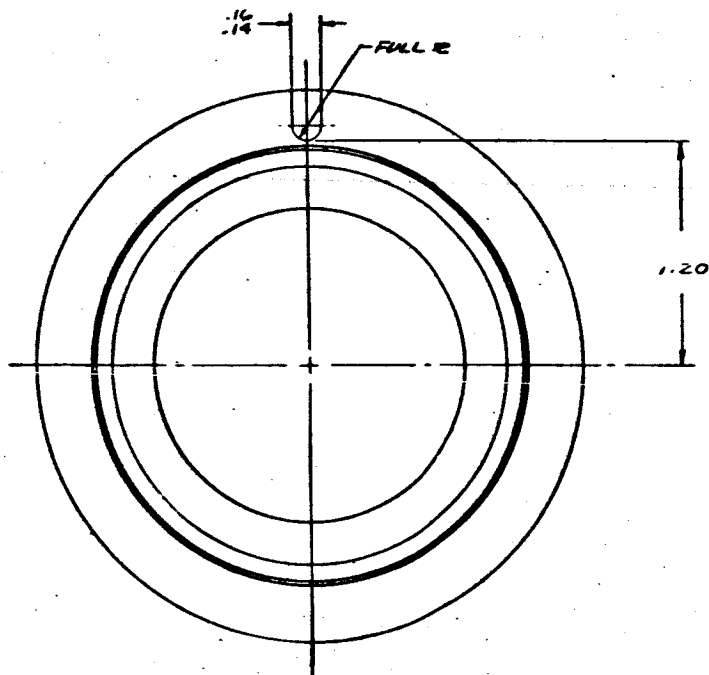


TY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
				BAR		STEEL 4340	AMS 6415	

← ASSYS		LIST OF MATERIAL	
369741 369740 369731 369730 369721 369720		DWT. TITLE MOUNT, BEARING - RESILIENT	
QD. NEXT ASSY. USED ON ATTREATMENT PROCESS		CODE IDENT NO. 99193 SIZE C DWT. NO. 369733	
NAME SPEC.		SCALE 2/1 WT. SHEET 1 OF 1	

85②

APS-5108-R
APS-5109-R



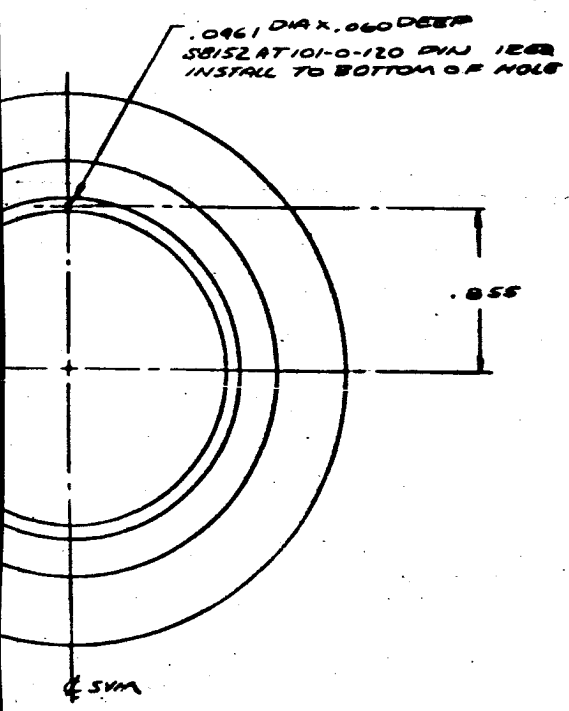
CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE
THIS PRODUCT DEPENDS ON THE SE-
THORITY AND RELIABILITY OF THE
DIRECTOR CRITICAL ITEM, INSPECTION
BY THE ITEM FROM THE
SAFETY CORPORATION IS RECO-
MMENDED BY COMPLIANCE WITH APS-
1,312.

- 12. POSITIONAL & GEOMETRIC SYMBOLS PER MIL-STD-8
- 11. SIMILAR TO PA 363573
- 10. MAGNETIC INSPECTION PER MIL-I-6868

850

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED
A	SEE ENGINEERING ORDER		



450734

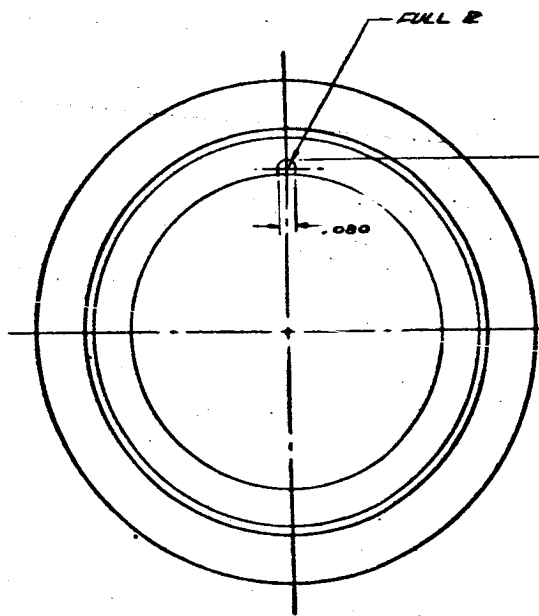
BAC		SYMBOL 450		AMS 6415	
SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.

LIST OF MATERIAL		
SIGNATURES <i>[Signature]</i> 9-18-63 <i>[Signature]</i> 9-25-63 DATE MAT. & PROC. <i>[Signature]</i> 9-25-63 STUDY <i>[Signature]</i> 9-25-63 APP. <i>[Signature]</i> 9-26-63 DESIGNED ACTIVITY APP. <i>[Signature]</i> 9-26-63 OTHER ACTIVITY APP.		TITLE CARRIER, BEARING CODE IDENT NO. 99193 SIZE D PART NO. 369734 SCALE 2/1 WT.

SHEET	1	OF 1
-------	---	------

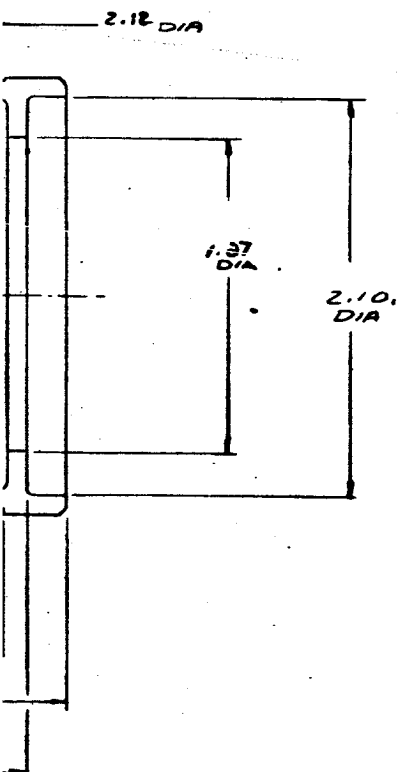
85 (T) 3

APS-5108-R
 APS-5109-R



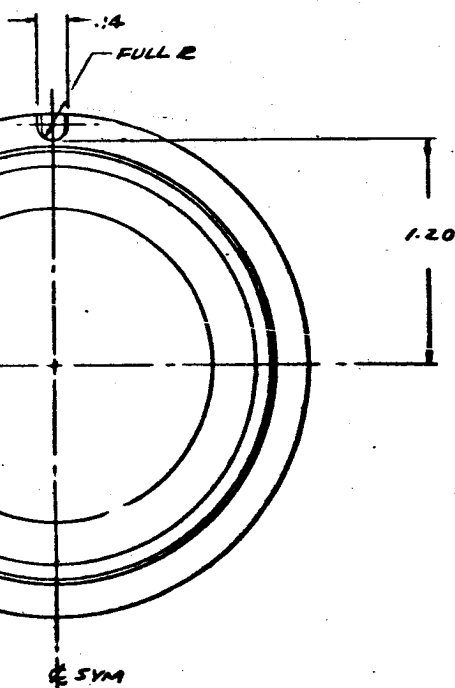
GENERAL NOTES
 1. ALL DIMENSIONS ARE TO BE TAKEN FROM THE FACE OF THE PART UNLESS OTHERWISE SPECIFIED.
 2. ALL DIMENSIONS ARE TO BE TAKEN FROM THE FACE OF THE PART UNLESS OTHERWISE SPECIFIED.
 3. ALL DIMENSIONS ARE TO BE TAKEN FROM THE FACE OF THE PART UNLESS OTHERWISE SPECIFIED.
 4. ALL DIMENSIONS ARE TO BE TAKEN FROM THE FACE OF THE PART UNLESS OTHERWISE SPECIFIED.

86①



- | | | | | | |
|---------------------|--|-----------|----------|----------|--|
| | | | | | |
| QTY. REQD. | | | ITEM NO. | PART NO. | |
| | | | ← ASSYS | | |
| | | | | | |
| | | | | | |
| | | | 365791 | 365 | |
| | | | 365781 | 365 | |
| 1 | | | 365721 | 365 | |
| REQD. | | NEXT ASSY | | USE | |
| HEAT TREATMENT | | | | PRO | |
| PARTS | | | | PART | |
| E-36-42 | | | | | |
| SPEC.
MIL-M-6875 | | | | SPEC. | |

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED
A	SEE ENGINEERING ORDER	1-5-63	<i>[Signature]</i>

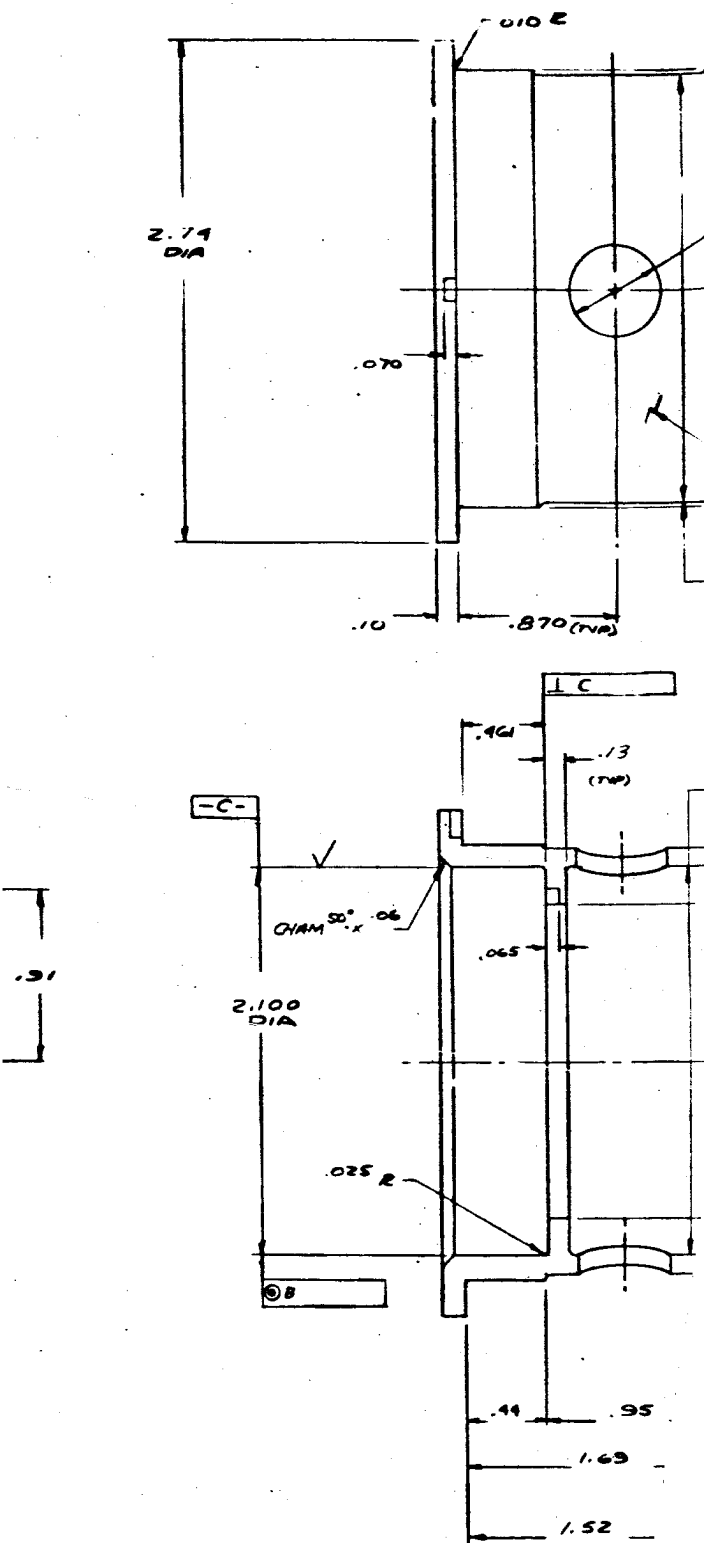


SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
	BAR		STEEL 4340	AMS 6415	

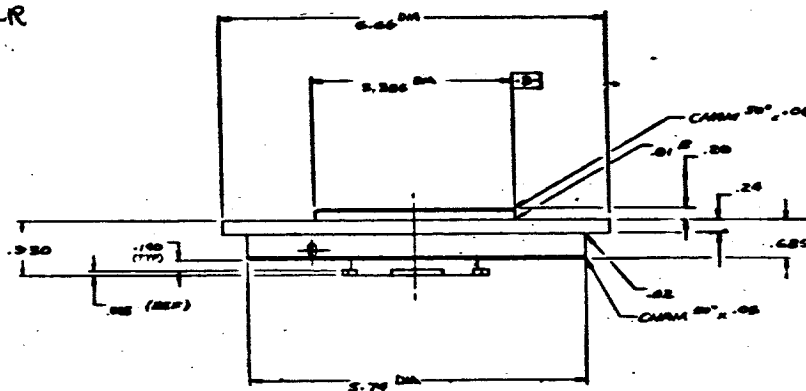
SIGNATURES		DATES		LIST OF MATERIAL All research Manufacturing Company of America PONTIAC, MICHIGAN DOW TYPE CARRIER BEARING
BY <i>[Signature]</i>	<i>[Signature]</i>	9-1-63		
CHE <i>[Signature]</i>	<i>[Signature]</i>	2-20-63		
APPR. <i>[Signature]</i>	<i>[Signature]</i>			
ANAL & PROCESS <i>[Signature]</i>	<i>[Signature]</i>	9-21-63		
STUD <i>[Signature]</i>	<i>[Signature]</i>	2-25-63		
ADG.				
APP. <i>[Signature]</i>		9-26-63		
APP.				
DESIGN ACTIVITY NO.				
<i>[Signature]</i>		9-26-63		
OTHER ACTIVITY APP.				

99193	D	369735
SCALE 2/1	WT.	

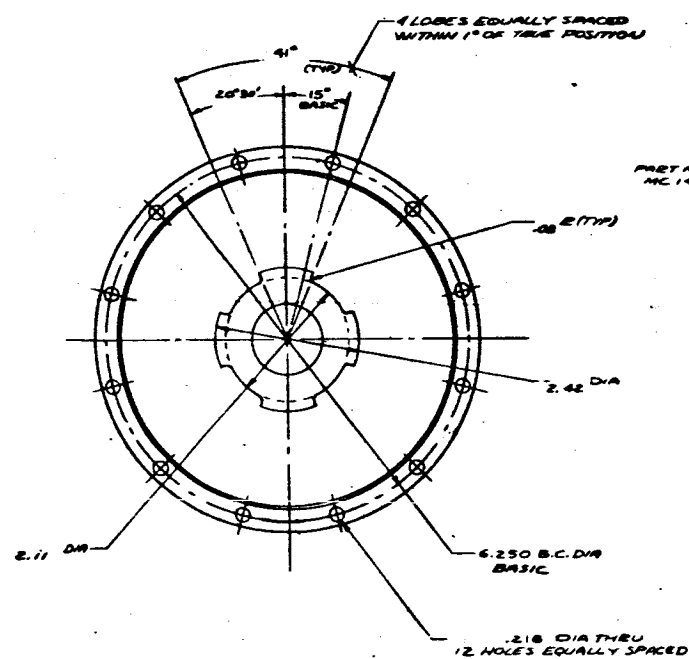
86 ① 4



12. PORTIONAL & GEOMETRIC SYMBOLS PER M.
 11. SIMILAR TO PA 363572
 10. MAGNETIC INSPECTION PER MIL-I-68



FRAME 25

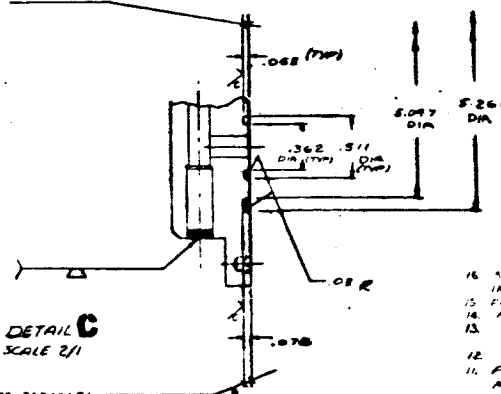


PART NO. HERE PER
MC 14, CLASS II

1.390
0/14

SECTA

THESE SURFACES PARALLEL
TO EACH OTHER



DETAIL C
SCALE 2/1

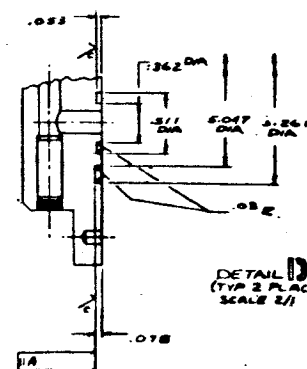
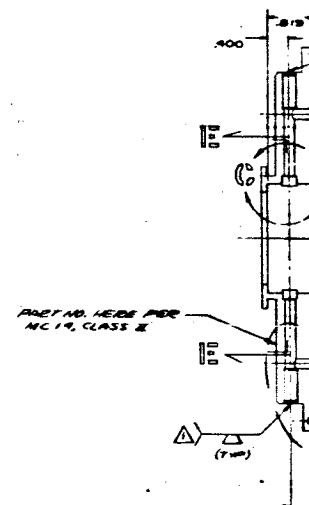
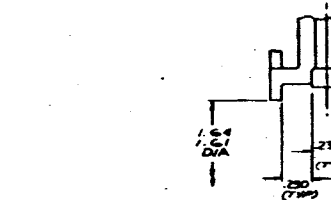
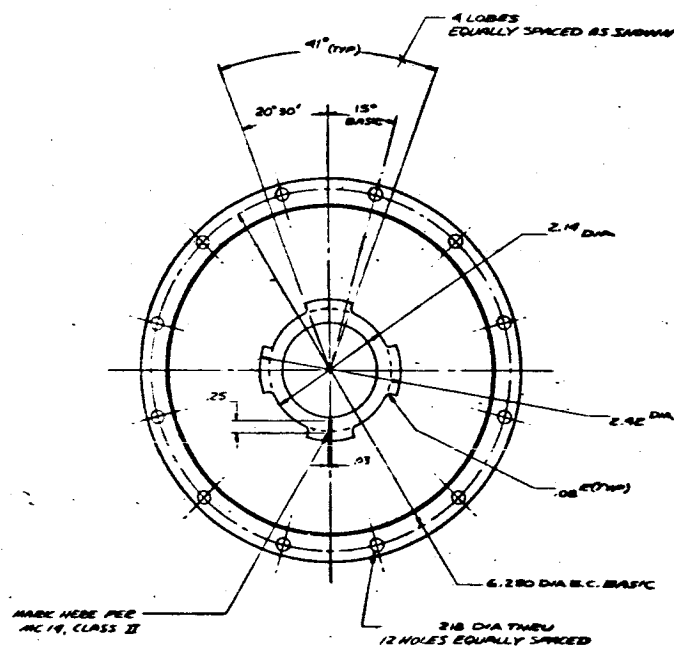
THESE SURFACES PARALLEL
TO EACH OTHER

16 9000 INST
15 FOOT
14 PER
13

12
11 PER
MIN
10

CERTICAL ITEM

870



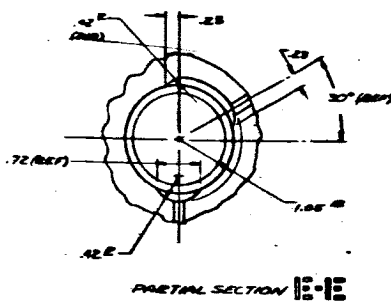
DETAIL D
(TYP 2 PLACES)
SCALE 2/1

10. POSITIONAL
11. PRESSURE
ONE MINUTE
12.
13.

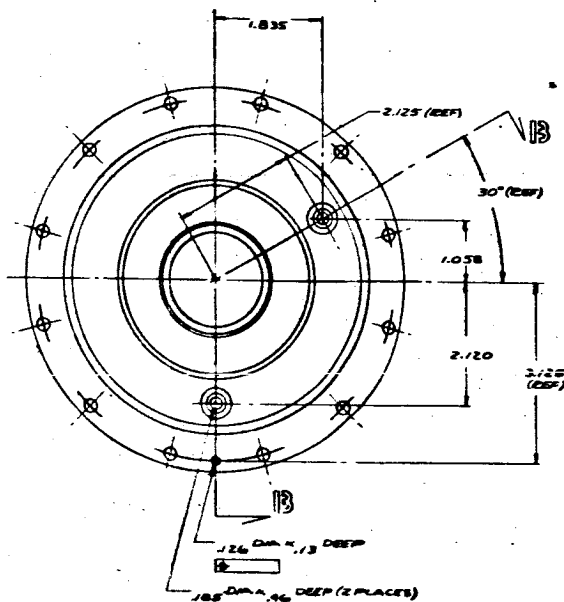
RECEIVED

88 (1)

REVISIONS				
DATE	BY	DESCRIPTION	DATE	APPROVED
1		SEE ENGINEERING ORDER	10/1/81	John
2		SEE ENGINEERING ORDER	10/1/81	John



PARTIAL SECTION **E-E**



.85 DIA. x 2.01- DEEP
 .205 DIA. x .68- CREEP
 (2 PLACES) INSTALL
 581528617-0-530 PIN 2880

③ WELDED ASSY CONSISTS OF
- 56 J81527617-0-520

SECTION 13-13

DIAMETRIC TOLERANCE SYMBOLS - REF. MAX-STD-2

EST WIND A/C AT 150 PSIG FOR

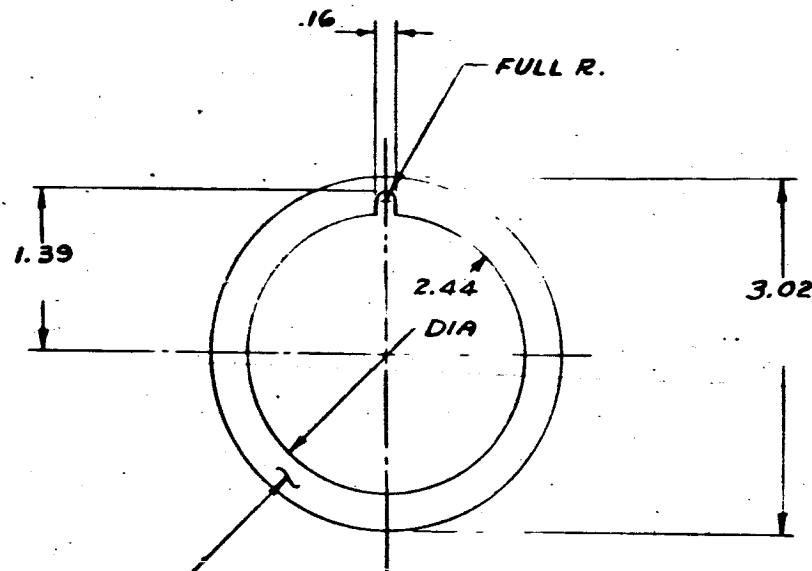
2. ENGRAINED SURFACES MUST BE WITHIN .005 INCH TO A MAX. OF .002 FOR ANY SURFACE.
3. ENGRAINED SURFACES NORMAL OR PLUG IN WITHIN .005 INCH TO A MAX. OF .002 FOR ANY SURFACE.
4. ENGRAINED DIAL OR CROWN GEAR TOOTH CONTACTS WITHIN .005 TO .001 (MIN).
5. ENGRAINED LINES HELD AFTER PLATING.
6. ENGRAINED PLUGS AND RINGS ARE.
7. BREAK ALL CORNERS AND ROUNDS EDGES .005 MAX. NO HANGING BURRS ALLOWED.
8. SURFACE ROUGHNESS PER RELATED.
9. ENGRAINING AND IN INCHES.
10. SURFACE FINISH SPECIFIED.

[illegible]

Q. 2.

PART NUMBER	T
369743-1	.013 - .011
369743-2	.004 - .002
369743-3	.007 - .005

APS-5168-R
APS-5169-R

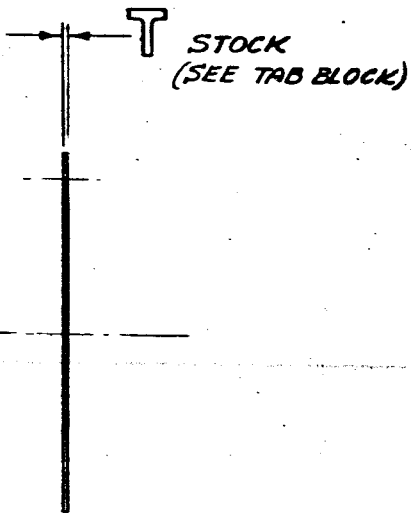


11. REMOVE BURRS.
10. MUST BE FREE FROM WRINKLES.
9. SHARP EDGES PERMISSIBLE.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
5. DIMENSION LIMITS HELD AFTER PLATING.
4. MACHINED FILLET RADII .030 - .015
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
2. SURFACE ROUGHNESS PER MIL-STD-10.
1. DIMENSIONS ARE IN INCHES.

UNLESS OTHERWISE SPECIFIED.

SYM	REVISIONS	
	DESCRIPTION	DATE APPROVED



DIA

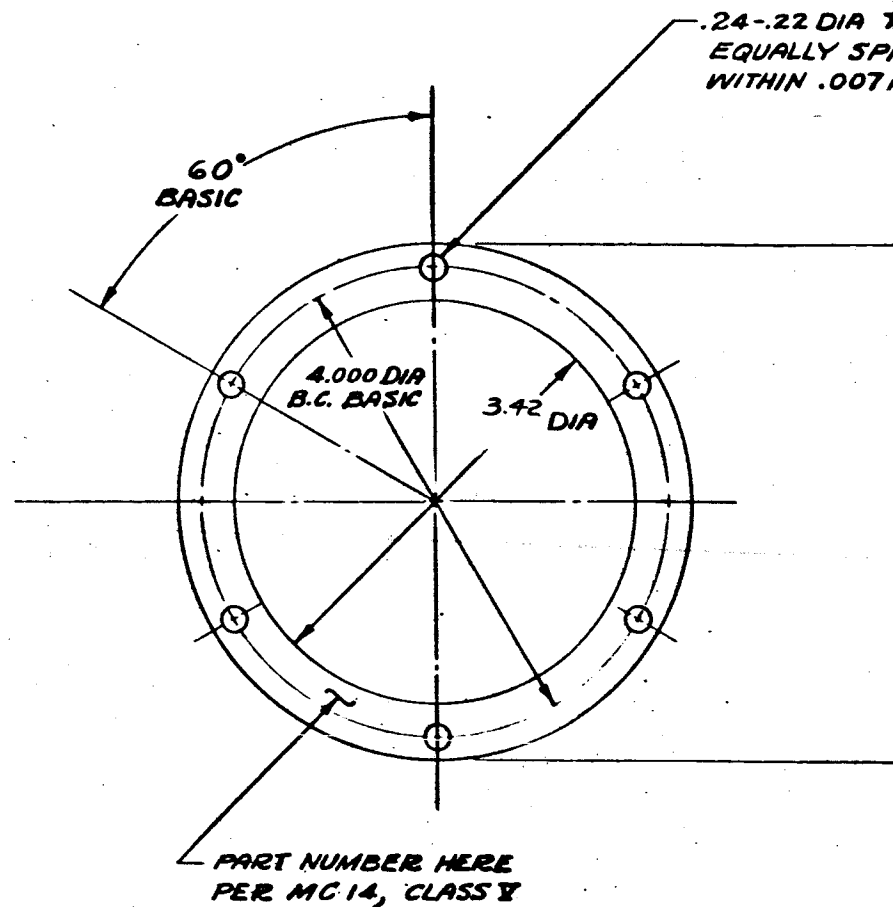
↑
LAST
369743

SEE TAB.		SHEET		CRES		MIL-S-6721	
Y. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION
← ASSYS				LIST OF MATERIAL			
369741 369740		SIGNATURES		DATES		AD Research Manufacturing Company of Arizona PHOENIX, ARIZONA	
3 369731 369730		DFT Ed. B. Bell		9-26-63		DWG. TITLE SHIM, BEARING CARRIER	
2) 369721 369720		CHK. Crawford		7-28-63			
1) 369721 369720		MFG ENG.					
ID. NEXT ASSY.		MATERIAL & PROCESS					
USED ON		STRESS					
AT TREATMENT		PROCESS		AERO			
DESIGN NAME		APP. J. P. ...		9-30-63			
SPEC.		DESIGN ACTIVITY		M. ...		9-30-63	
		OTHER ACTIVITY APP.					
		CODE IDENT NO.		SIZE		DWG. NO.	
		99193		C		369743	
		SCALE FULL		WT.		SHEET 1 OF 1	

89 (2)

PART NUMBER	T
369744-1	.013-.011
369744-2	.026-.024
369744-3	.007-.005

APS-5108-R
APS-5109-R

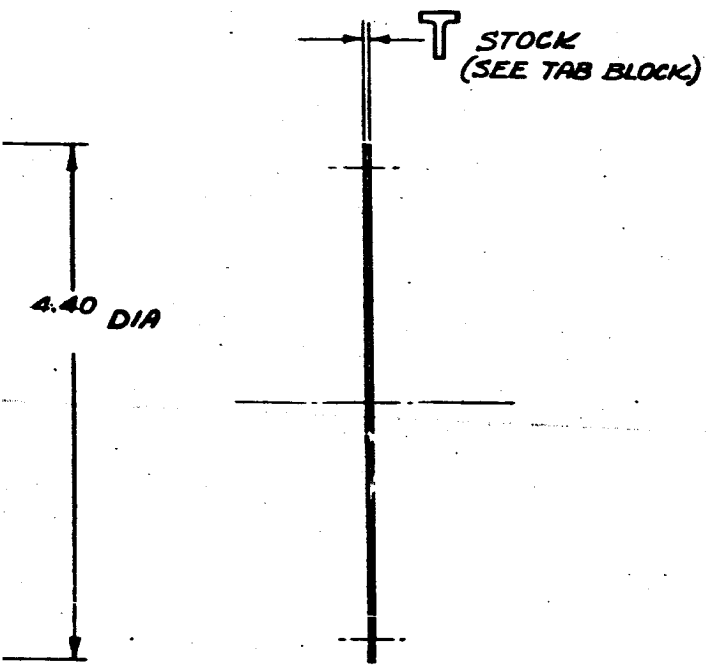


11. REMOVE BURRS.
10. MUST BE FREE FROM WRINKLES.
9. SHARP EDGES PERMISSIBLE.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
 5. DIMENSION LIMITS HELD AFTER PLATING.
 4. MACHINED FILLET RADII .030 - .015
 3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

REVISIONS	
YM	DESCRIPTION
A	SEE ENGINEERING ORDER
DATE	APPROVED
1-11-64	<i>[Signature]</i>

THRU 6 HOLES
 PLACED AS SHOWN
 IN FIG. 1 OF BASIC POSITION



↑
 LAST
 A
 369744

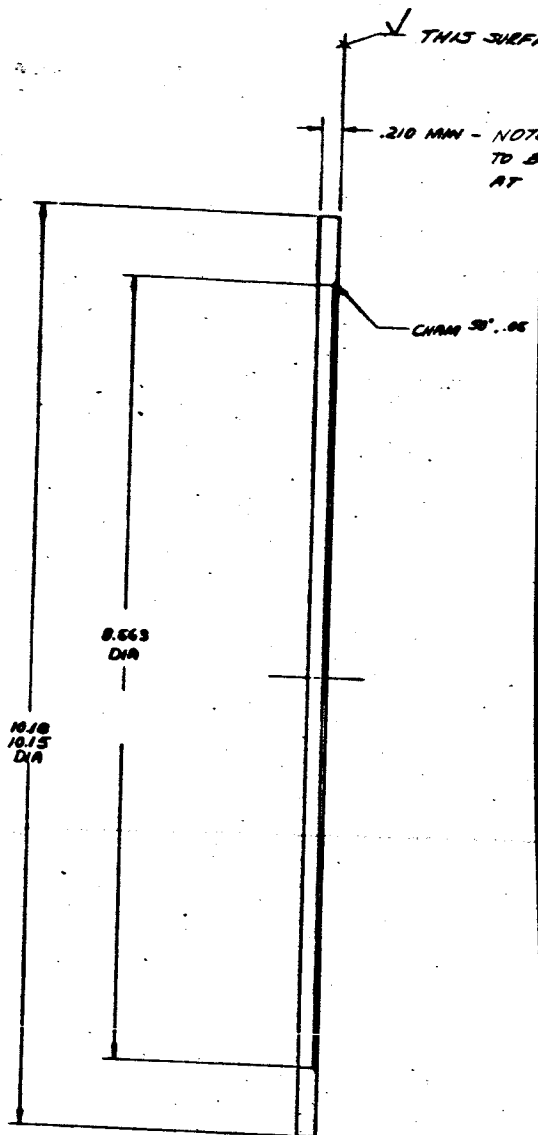
Y. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
		SEE TAB.		SHEET		CRES	MIL-S-6721	

← ASSYS		LIST OF MATERIAL	
SIGNATURES		DATES	
BPT. <i>[Signature]</i>		9-26-63	
CHK. <i>[Signature]</i>		9-28-63	
MPL. ENG.			
MAT. & PROCESS			
FINISH			
ACRD			
APP. <i>[Signature]</i>		9-30-63	
APP.			
TREATMENT		9-28-63	
FINISH			
SPEC.			

AR Research Manufacturing Company of Arizona PHOENIX, ARIZONA		
DRWG. TITLE <h1>SHIM, SEAL</h1>		
CURR. IDENT. NO. 99193	SIZE C	DRWG. NO. <h1>369744</h1>
SCALE FULL		WT.
SHEET 1 OF 1		

90 (2)

APB-5108-R
APB-5109-R



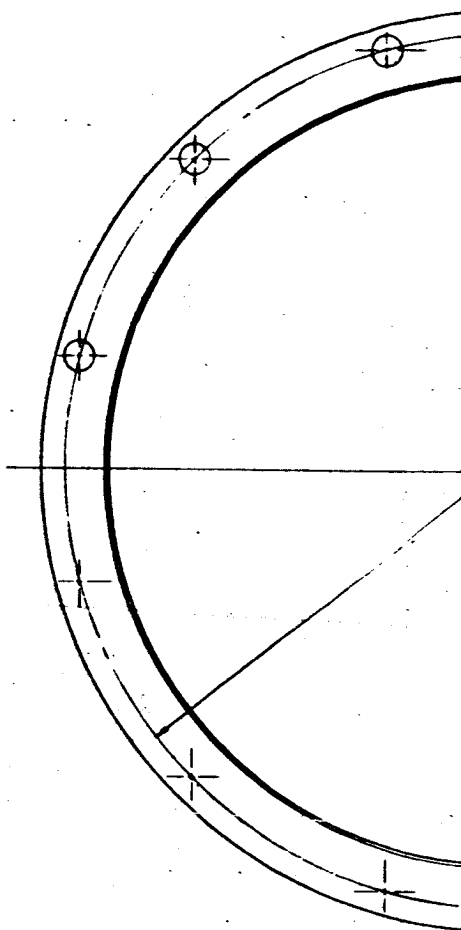
NOTES:
1. THE DIMENSIONS OF THE
PRODUCT SHOWN ON THE
DRAWING ARE THE DIMENSIONS
OF THE PARTS AND NOT THE
DIMENSIONS OF THE PARTS
AS SHOWN IN THE DRAWING.
2. THE DIMENSIONS OF THE
PARTS ARE THE DIMENSIONS
OF THE PARTS AND NOT THE
DIMENSIONS OF THE PARTS
AS SHOWN IN THE DRAWING.

APB-5108-R and APB-5109-R

910

ACE PLAT

E: FINAL THICKNESS
E ESTABLISHED
ASSEMBLY



9. MAGNETIC PARTICLE INSPECT PER MIL-I-6868

REVISIONS			
SYM.	DESCRIPTION	DATE	APPROVED
1	SEE ENGINEERING ORDER	10-11-63	E. J. [Signature]

SYMBOL		DESCRIPTION	CAGE IDENT	MATERIAL	SPECIFICATION	UNIT WE.
STEEL 2512						
LIST OF MATERIAL						
SIGNATURES		DATES		AIRBORNE Manufacturing Company of America FORMERLY AIRBORNE		
BY: [Signature]		2-20-63		DRWG. TITLE SHIM- SEALING SPACER		
CHK: [Signature]		10-3-63				
DATE: [Signature]		10-8-63				
BY: [Signature]		10-5-63				
CHK: [Signature]		10-9-63				
BY: [Signature]		10-9-63		CAGE IDENT NO. 99193		
BY: [Signature]		10-9-63		Dwg. No. 369745		
BY: [Signature]		10-9-63		SCALE FULL WT.		
BY: [Signature]		10-9-63		SHEET 1 OF 1		

9104

369745

APC-5708-R
APC-5709-R

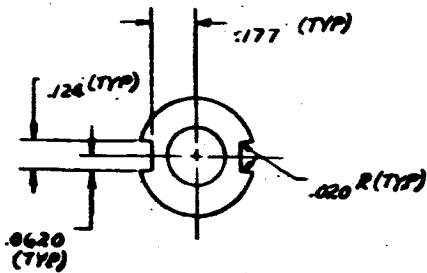


SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE INTEGRITY AND RELIABILITY OF THIS SELECTED CRITICAL ITEM. PROCUREMENT OF THIS ITEM FROM THE GARRETT CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPL 1.312.

- UNLESS OTHERWISE SPECIFIED.

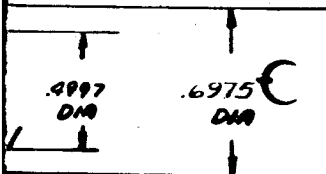
920

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED
A2	SEE ENGINEERING ORDER	11/2/63	<i>[Signature]</i>



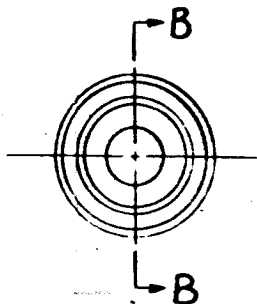
SECTION D-D

CENTER OPTIONAL



CHAM 50° 08

NO. HERE PER
M, CLASS II



QTY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
						STEEL		

← ASSYS		LIST OF MATERIAL	
SIGNATURES		DATES	
DFT. <i>[Signature]</i>		10-1-63	
CHK. <i>[Signature]</i>		10-3-63	
MFG. ENG.			
MAT. & PROCESS <i>[Signature]</i>		10-9-63	
STRESS <i>[Signature]</i>		10-6-63	
AERO			
APP. <i>[Signature]</i>		10-9-63	
APP.			
DESIGN AUTHORITY APP. <i>[Signature]</i>		10-9-63	
OTHER ACTIVITY APP.			

EAT TREATMENT		PROCESS	
NAME		NAME	
SPEC.		SPEC.	

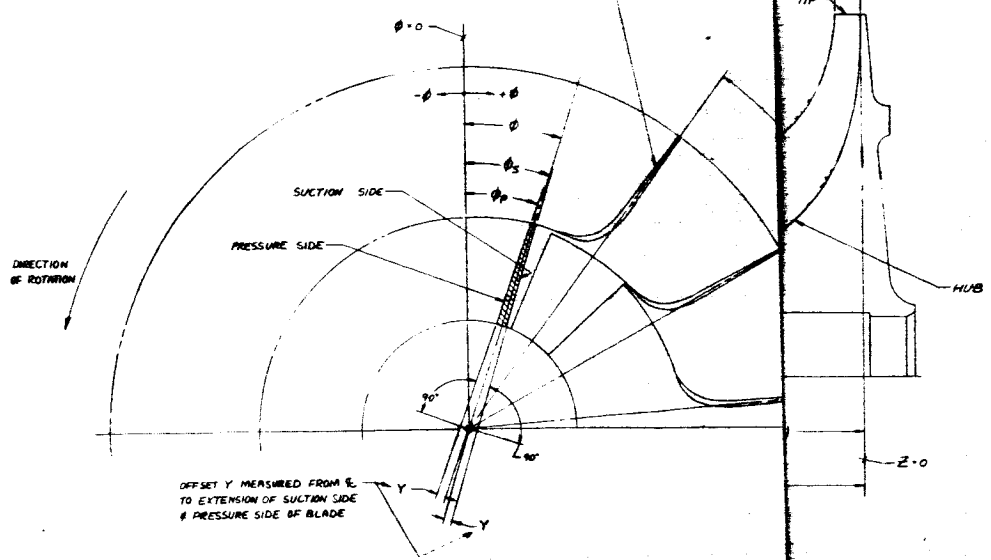
CODE IDENT NO.	SIZE	DRAW. NO.
99193	C	369746
SCALE TWICE	WT	SHEET 1 OF 1

↑
369746
A

92(2)

APC-1000-1

15 BLADES EQUALLY SPACED



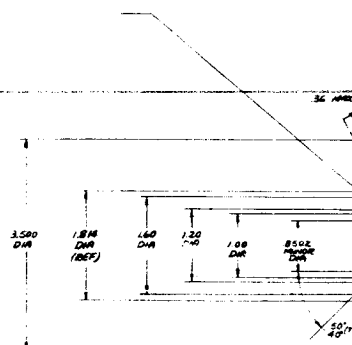
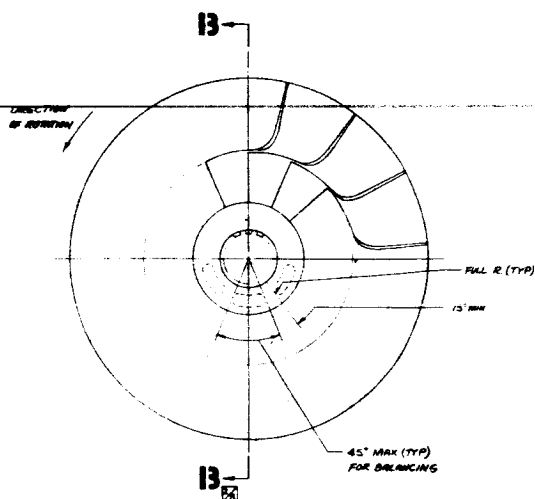
93①

93②

ITEM	DESCRIPTION
1	TYPE - PLAT ROOT SIDE RT
2	ADJUSTMENT OFFSET .004" DIAMETER FOR
3	CIRCULAR SHAPE WITH 100% GAGE
4	TYPICAL TOLERANCES UNLESS OTHERWISE SPECIFIED
5	GEOMETRIC TOLERANCES UNLESS OTHERWISE SPECIFIED
6	TRUE REVOLUTE FORM DIA

PLANE M

11A



16

15

9.3 (3)

16 FLASH COPPER PLATE SPLINE

13

12. POSITIONAL AND GEOMETRIC TOLERANCE SY PER MIL-STD-8.

11. FINISH SPLINE TOOTH PROFILES ✓

10. FINISH ALL OVER ✓ EXCEPT BLADE & BL

9. A SURFACE AND C DIA ESTABLISH E

20. THE COMPLETE SERIAL NO SHALL BE TRANSFERRED FROM FORGING AND RETAINED ON PART THROUGH-OUT LIFE OF THE PART

19. SHOT BEEN ENTIRE AREA ENCLOSED THUS (---) PER AMS 2430 USING .021- DIA DIA STEEL SHOT TO AN INTENSITY OF .006-.008 ALHARD 92

18 TYPE I PENETRANT INSPECT PER MIL-1-6866 USING EL-2 PENETRANT

17. BLADE SHAPE TO FAIR SMOOTHLY BETWEEN POINTS DIMENSIONED IN TABLES.

DATE

TIME

AP3-5168-R
AP3-5169-R

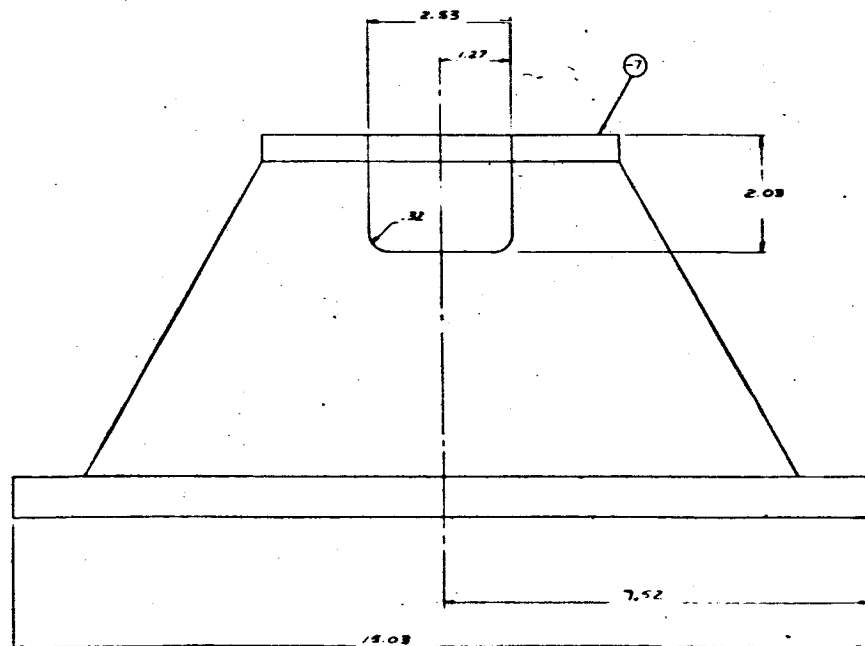
D

C

B

A

PART NO. / ASSY HERE PER AC-14, CLASS II

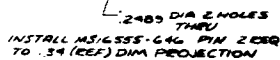


(TYP)

CRITICAL DIMS	
1	2.83
2	1.27
3	7.52
4	1.03
5	2.03
6	.32

94 ①

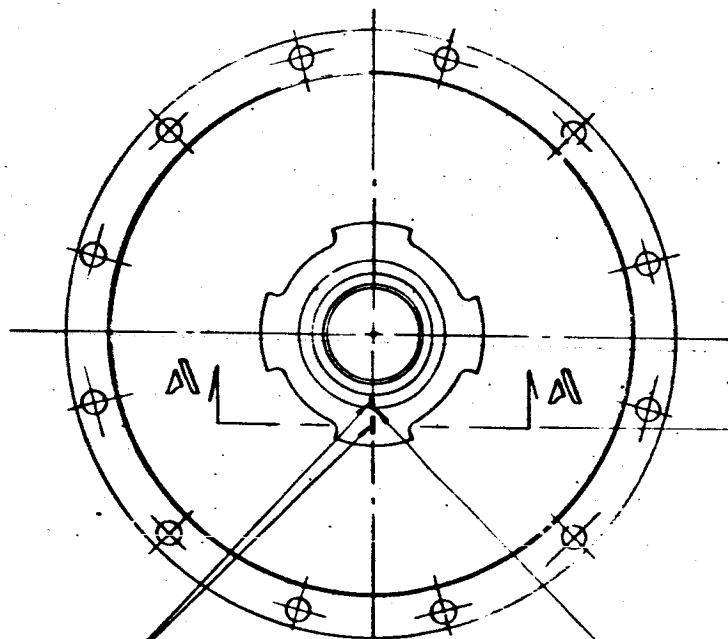
1001



1.534 TO BE FREE OF WELD SPATTER

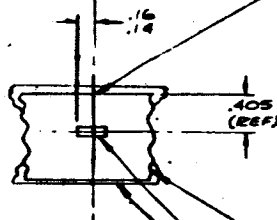
[illegible]

APB-5708-R
APB-5709-R



MARKS TO BE IN LINE
WITHIN $\pm .020$

MARK SEAL HERE PER MC 14, CLASS II
PRIOR TO ASSY. LOCATE AS SHOWN WITH
RESPECT TO SLOT IN SEAL.



SECTION A-A

SEAL (REF)

SLOT (REF)

CARRIER (REF)

CRITICAL ITEM
SATISFACTORY PERFORMANCE OF THE
END PRODUCT DEPENDS ON THE
TIMELY AND RELIABLE OF THE
SUBJECT CRITICAL ITEM. PROCEED
WITH THE ITEM FROM THE
QUALITY CORPORATION IS SECURE
SHOWN IN COMPLIANCE WITH ASME
1.12.1

90040 - 0100 000 100 000 000 000

950

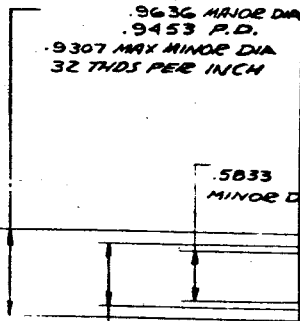
INTERNAL SPLINE DATA <i>RCW 2-11</i>		EXTERNAL
ASA B6.5-15-1968 STANDARD INVOLUTE	15 TEETH	ASA B6.15-1960 STANDARD
DIAMETRAL PITCH	24 / 48	DIAMETRAL PITCH
PRESSURE ANGLE	30°	PRESSURE ANGLE
FLAT FOOT - SIDE FIT		TYPE - FLAT FOOT
MAJOR DIAMETER	.6790	MINOR DIAMETER
MEASUREMENT BETWEEN .0400 DIAMETER PINS	.5594 MAX	MEASUREMENT OVER .0400
CIRCULAR SPACE WIDTH WITH GAGE	.0654 MIN EFFECTIVE	CIRCULAR TOOTH THICKNESS
TOOTH PARALLELISM ERROR ACROSS FACE WIDTH	MAX REF	TOOTH PARALLELISM ERROR
ECCENTRICITY OF PITCH DIAMETER WITH AXIS OF B DIAMETER	PR MAX	ECCENTRICITY OF PITCH DIAMETER WITH AXIS OF B DIAMETER
		TRUE INVOLUTE

MPS-5104-R

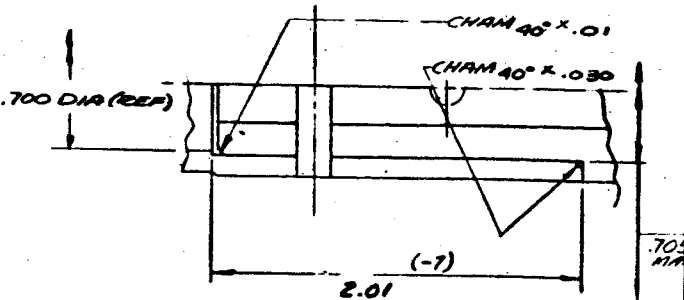
1870 DIA THRU 2 WALLS. INSTALL MS16555-
FINISH END SIMULTANEOUSLY WITH $\pm .005$ DIA (



FLASH COPPER PLATE ENTIRE INSIDE OF CAVITY,
& END OF SHAFT - SPLINE DIMENSIONS PRIOR TO PLATING

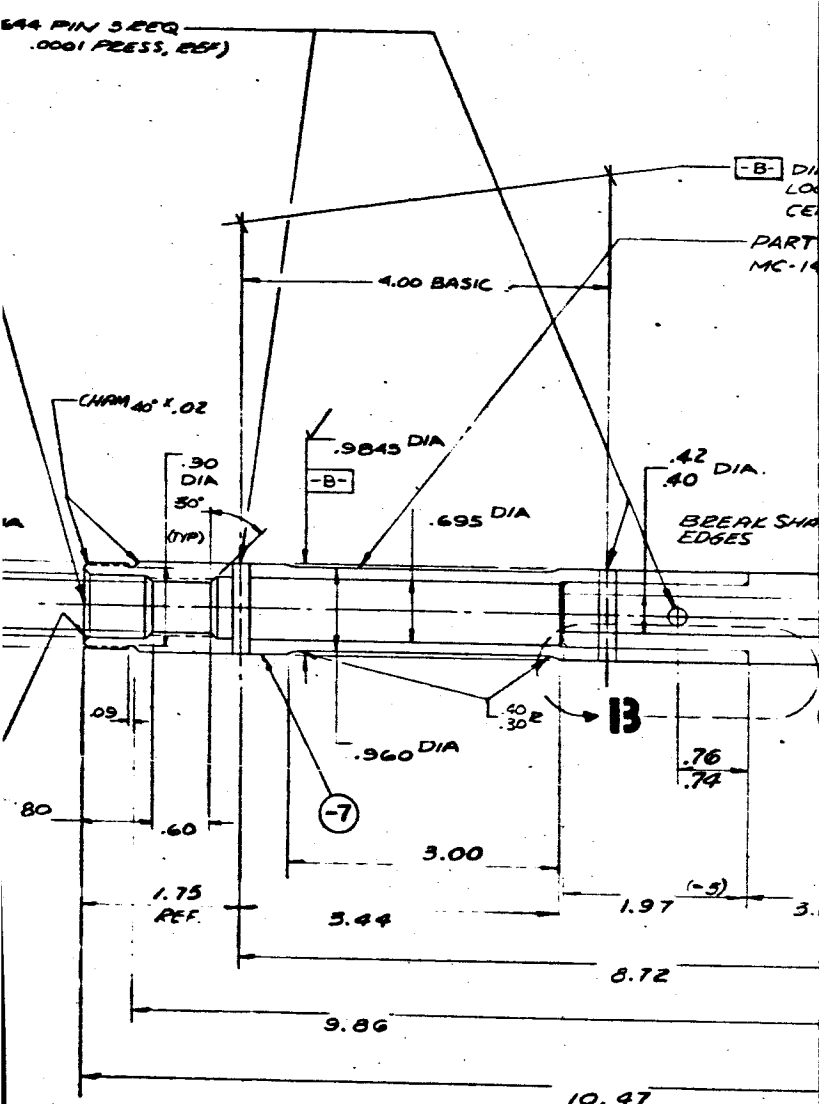


.6985 DIA
CSK 59° x .72 DIA



VIEW B
SCALE 2/1

SPLINE DATA		ECU-100-1 11-7-63
BAND ENVELOPE	18 TEETH	
	20 / 40	
	30°	
NOT SIDE FIT	CLASS 1	
	.826	
360 DIAMETER PINS	1.0415	MAX DEF MIN
PINS WITH GAGE	.0770	MAX EFFECTIVE
R ACROSS FACE WIDTH		MAX DEF
CH DIAMETER WITH		PER MAX
FORM DIA.	.8462 MAX	



2 DIA. (-7 SHAFT)

TEMP. 60.0°F (-7)

18. POSITIONAL & GEOM. TOL. SYM.
PER MIL-STD-8

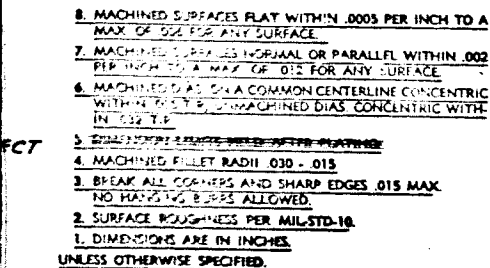
12 WA. (-5 SHAFT) $\triangle 3/9$.

12. AFTER FINAL MACHINING, PENETRANT INSP.
PER MIL-I-6866, TYPE I

11. SIMILAR TO 369726

10

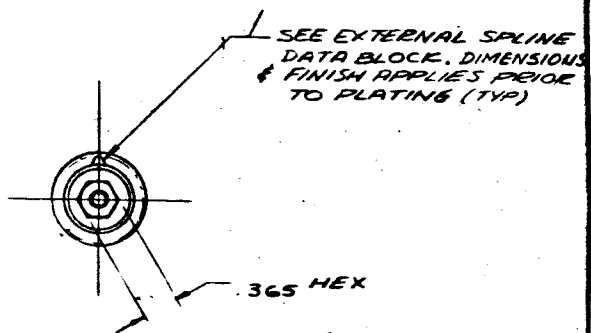
NO. HERE PER
CL.II



		-7	
		-5	
QTY. REQ.	ITEM NO.	PART NO.	SYM.
<div style="text-align: center;">← ASSYS</div>			
			INSTR.
			DWG.
			CHG.
			MFG. ENG.
			MAINT. & R.
1 369751 369730			SYNOPS.
REQD.	NEXT ASSY.	USED ON	AERO
			APP.
HEAT TREATMENT		PROCESS	APP.
FINISHES		GRADE	DESIGN A
		NOTED	NA
SPEC.		SPEC.	OTHER A
		NOTED	

3

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED



369754

CRITICAL ITEM
 SATISFACTORY PERFORMANCE OF THE
 END PRODUCT DEPENDS ON THE IM-
 PORTANCE AND RELIABILITY OF THE
 SELECTED CRITICAL PART. PROCE-
 DURE OF THIS PART FROM THE
 GARRETT CORPORATION IS BEING
 MONITORED IN COMPLIANCE WITH AFR
 1.313.

SHAFT		STEEL 4340		
SHAFT		STEEL 4340		
DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.

SIGNATURES		DATES		APPROVED:	
		12-10-63 11-7-63		APPROVED:	
11-2-63 11-14-63		DATE: 11-15-63 SHAFT ASSEMBLY, IMPELLER			
11-11-63		CODE IDENT NO. 99193	DISC D	DISC NO. 369754	
SCALE: FULL		WT.		SHEET 1 OF 1	

9604

SY-2261-A

1.986
BASIC
TYP22°30'
REF.823
BASIC
TYP

343 DIA THRU
300T-90E OTHER SIDE
48 DIA = DIA NOTED
B HOLES EQUALLY SPACED

D

1.986
BASIC
TYP.823
BASIC
TYP

4.99 DIA

4.300 DIA B.C.
REF

VIEW N-N

89° (TYP)

44°

2.08 DIA
(REF).50 (REF)
(TYP).75 DIA (REF)
(TYP)

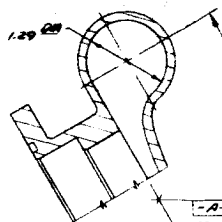
SECTION M-M

5.16 R
(REF)

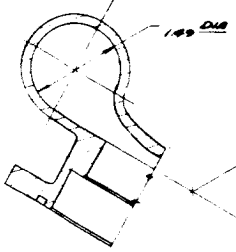
1.69 DIA

-A-
(REF)

180°

4.94 R
(REF)-A-
(REF)

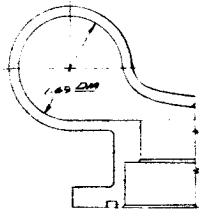
120°

5.05 R
(REF)

1.49 DIA

-A-
(REF)

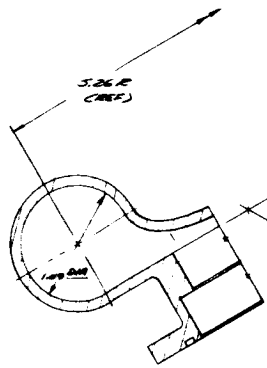
150°

5.16 R
(REF)

1.69 DIA

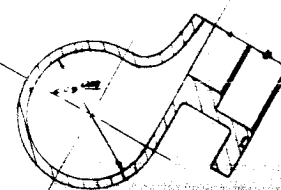
-A-
(REF)

180°

6.280 DIA B.C.
BASIC5.06 R
(REF)

1.69 DIA

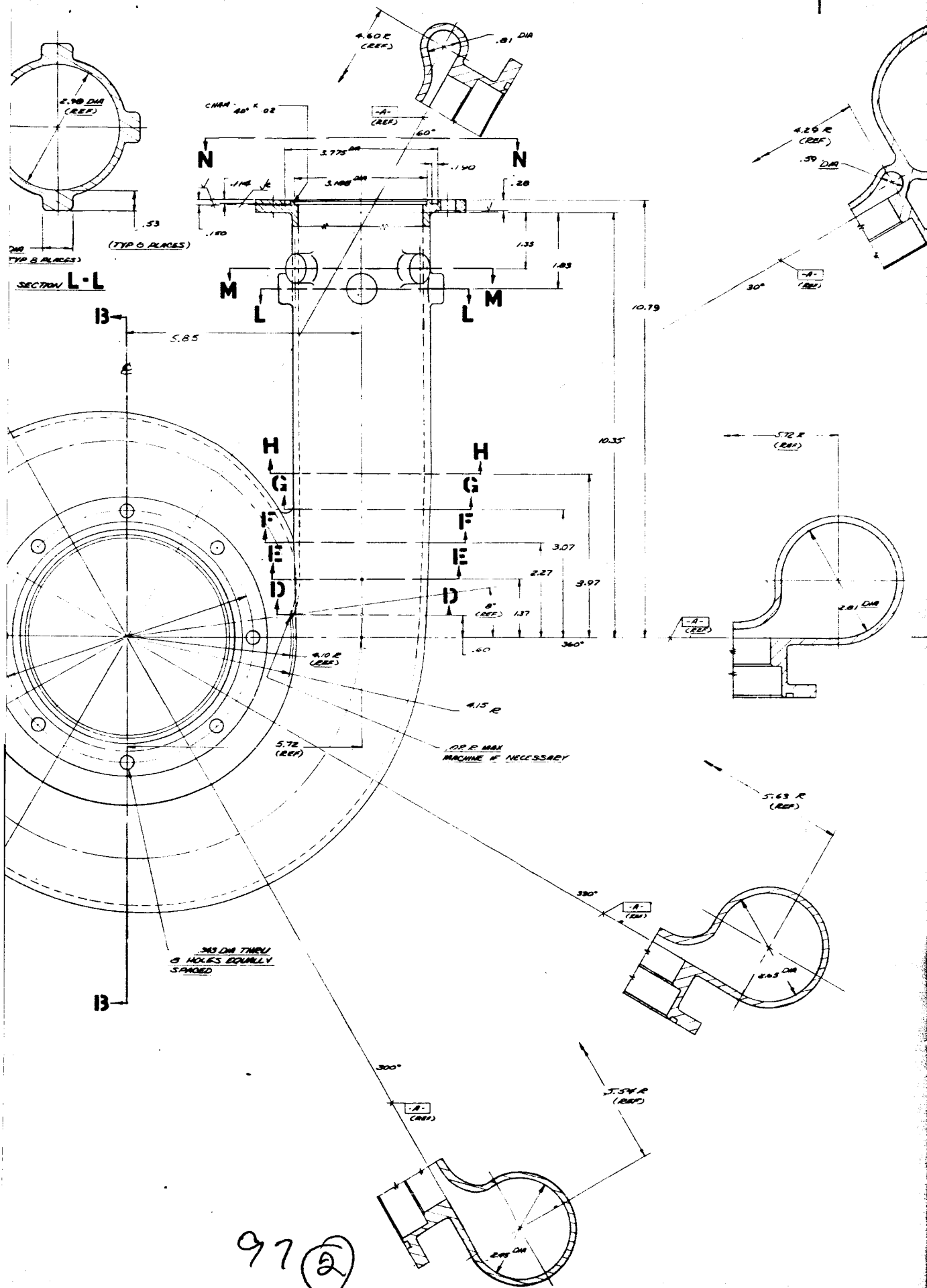
20°

-A-
(REF)5.36 R
(REF)

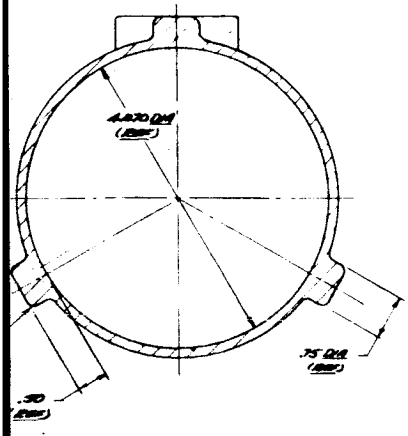
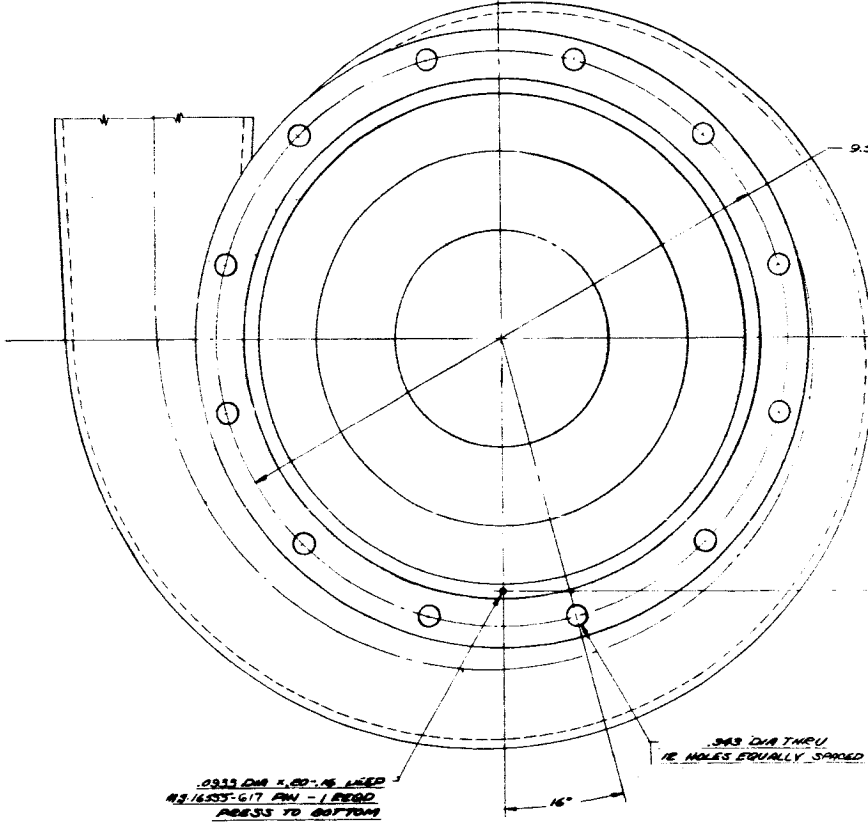
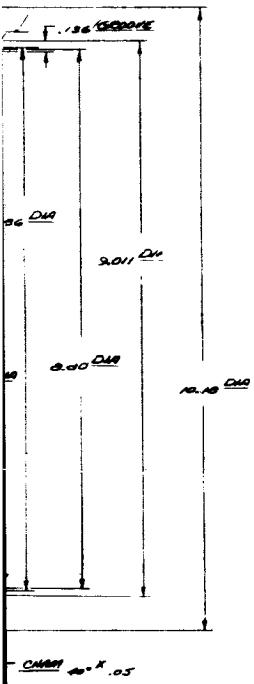
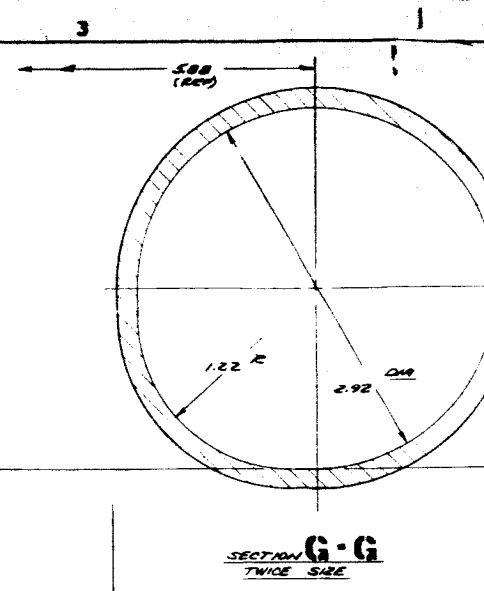
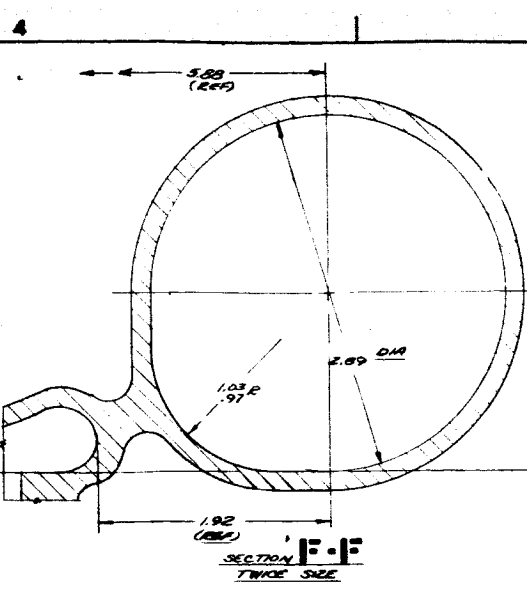
240°

-A-
(REF)

970

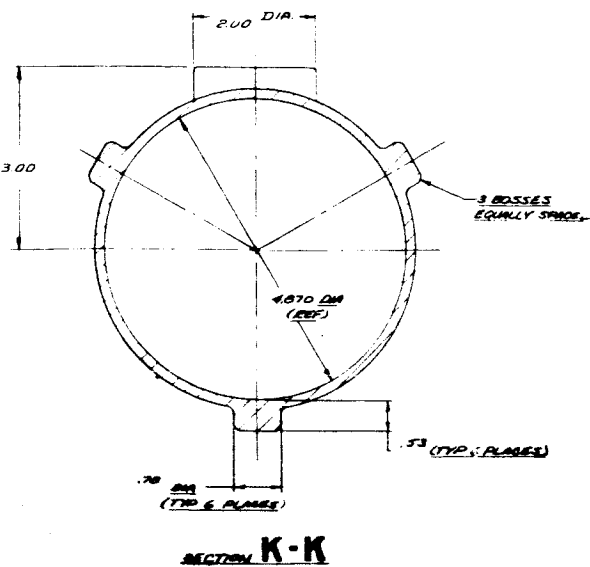
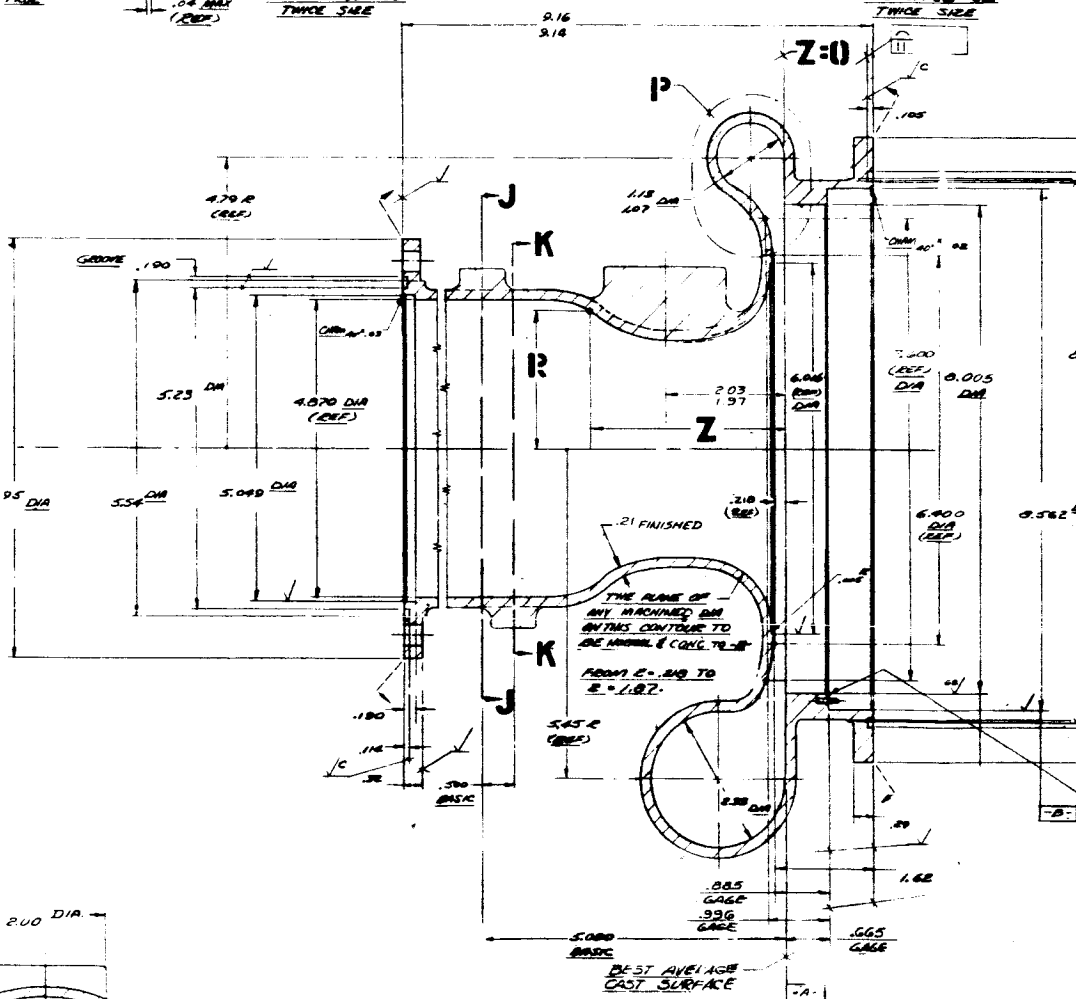
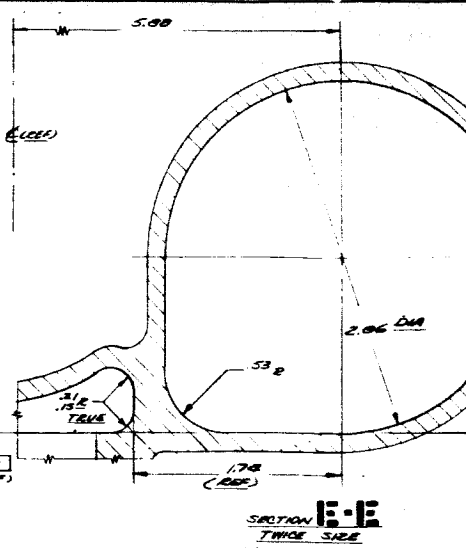


97 (2)



- 25.
- 26. POSITIONAL & GEOMETRIC
REF. DIM. STD. 1.
- 27.
- 28. CAST PART NUMBER, FOR
PART NUMBER ON AN
SURFACE MAP AREA
- 29.
- 30.
- 31. ALL RADII & CONTOURS

574



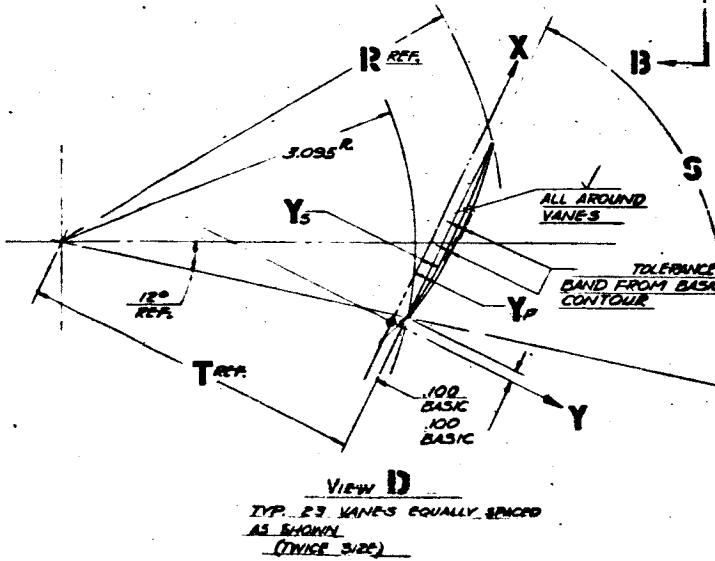
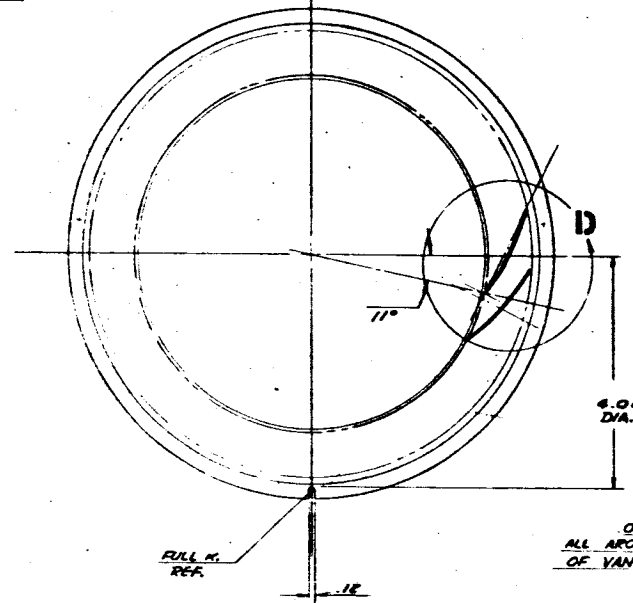
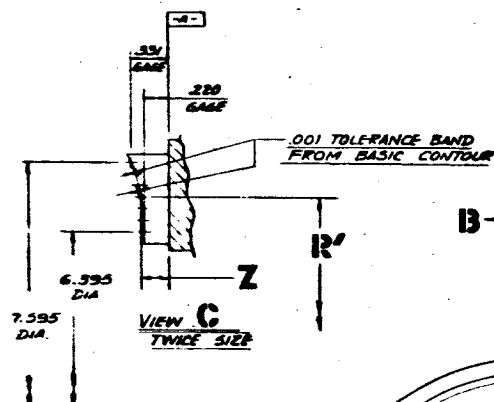
NOTES: TOLERANCE RING: FROM BASIC CONTOUR AS FOLLOWS:
FROM $E = 1.07$ TO END TOLERANCE
FROM $E = 1.000$ TO $E = 1.07$ - TOLERANCE
FROM $E = 1.000$ TO $E = 1.000$ - TOLERANCE
FOR E GREATER THAN 1.000 - TOLERANCE = $.000$ -
MACHINED PORTION OF CONTOUR MUST BLEND SMOOTHLY
WITH AS CAST SURFACE.

3 BASSES
SPARLY SPREAD

97③

ART NUMBER	R	PAID REF.	S	ANGLE	T	REF.
369758-10	3.943		72°45'		2.966	
369758-20	3.675		75°45'		3.006	
369758-30	3.807		78°45'		3.041	

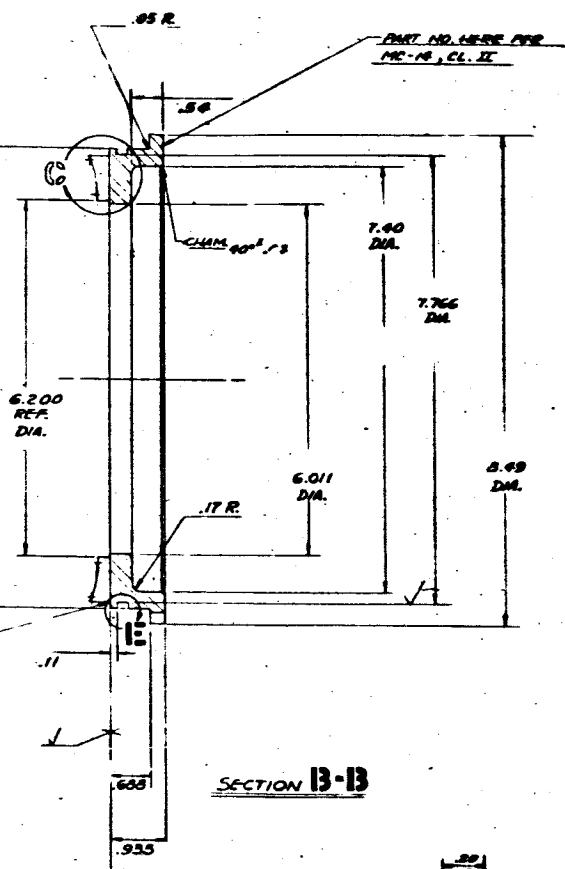
APS-5109-R



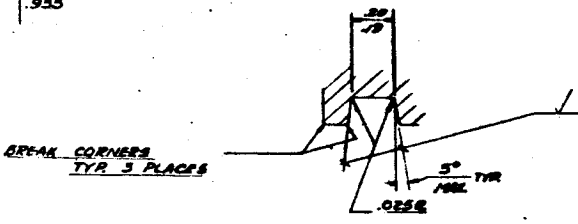
VIEW D
TYP. 23 VANES EQUALLY SPACED
AS SHOWN
(TWICE SIZE)

98

X 369758



SECTION B-B



VIEW E
4 x 5/16

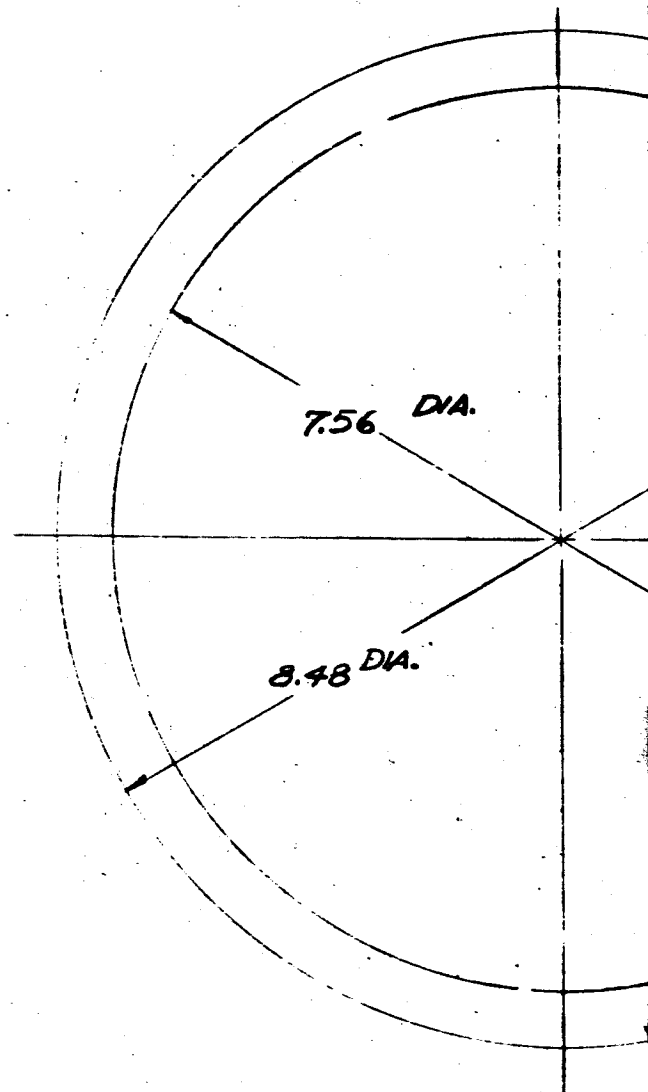
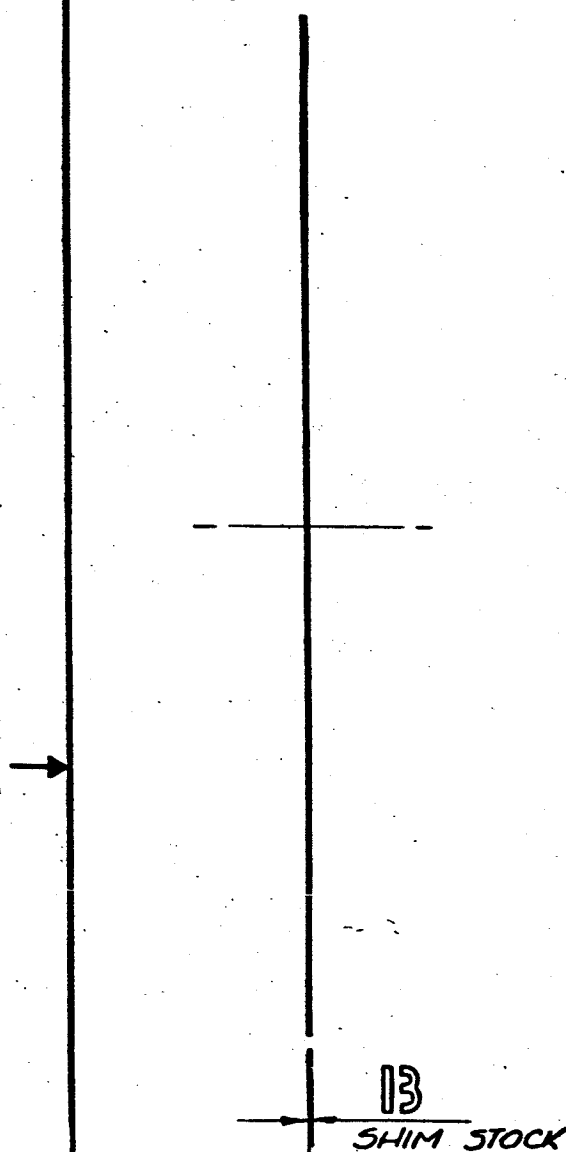
11. PENETRANT INSPECTION
PER MIL-I-6066
12. POSITIONAL & GEOM. TOL
2X PER MIL-STD-8

1. MACHINED SURFACES FLAT WITHIN .001" DIA.
2. SURFACES 1" TO 2" DIA. .001" DIA.
3. SURFACES 2" TO 4" DIA. .001" DIA.
4. SURFACES 4" TO 6" DIA. .001" DIA.
5. SURFACES 6" TO 10" DIA. .001" DIA.
6. SURFACES 10" TO 16" DIA. .001" DIA.
7. SURFACES 16" TO 25" DIA. .001" DIA.
8. SURFACES 25" TO 40" DIA. .001" DIA.
9. SURFACES 40" TO 60" DIA. .001" DIA.
10. SURFACES 60" TO 100" DIA. .001" DIA.

REV		DATE	DESCRIPTION	BY	CHKD	DATE	DESCRIPTION	BY	CHKD
1									
<p>98(2)</p> <p>DIFFUSER, COMPRESSOR</p> <p>99193 369756</p>									

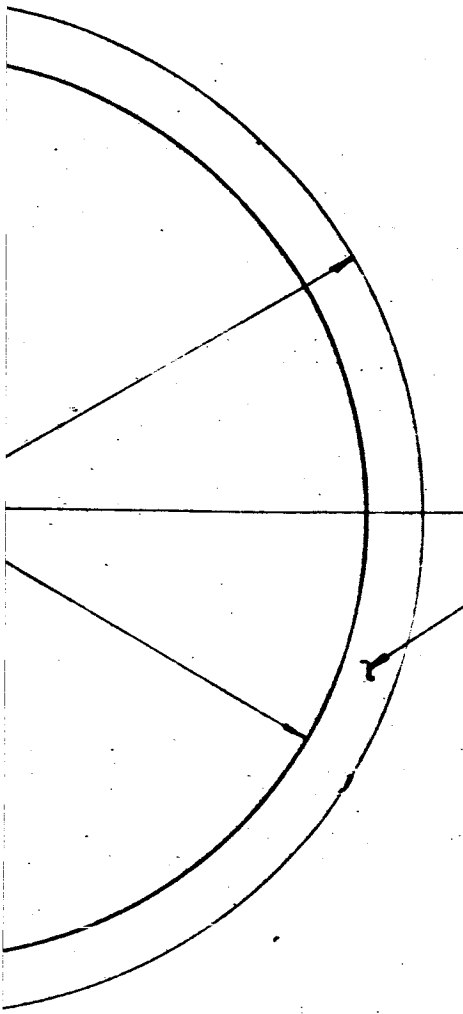
PART NO.	DIM.
369759-3	.002
369759-6	.004
369759-7	.016

APS-5109-R



10. PART TO BE FREE OF WRINKLES
 9. SHEARED EDGES PERMISSIBLE
 8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
 - ~~5. DIMENSIONS LIMITED TO 0.015 AFTER FINISHING~~
 4. MACHINED FILLET RADII .030 - .015
 - ~~3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.~~
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

990



PART NO. HERE PER
MC-14 , CLASS II

REVISIONS

SYM	DESCRIPTION	DATE	APPROVED

99(2)

QTY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
						CRES	MIL-S-6721	
← ASSYS				LIST OF MATERIAL				
SIGNATURES		DATES		AIR Research Manufacturing Company of Arizona PHOENIX, ARIZONA				
DFT. <i>[Signature]</i>		10-21-63		DRAW. TITLE <u>SHIM , HOUSING, DIFFUSER</u>				
CHK. <i>[Signature]</i>		10-25-63						
MFG. ENG.								
MAT. & PROCESS								
STRSS								
ASRD.				CODE IDENT NO. SIZE DRAW. NO. 99193 C X 369759				
APP. <i>[Signature]</i>		10-25-63						
APP.				SCALE <i>FULL</i> WT. - SHEET 1 OF 1				
DESIGN ACTIVITY APP. <i>[Signature]</i>		10-25-63						
OTHER ACTIVITY APP.								
HEAT TREATMENT		PROCESS						
HARDNESS		NAME						
PFC		SPEC.						

↑
X 369759

Z**R**

APS-5109-R

PART NO. HERE
MC-14, CL. IIA OR**Z**-0**R**.169 DIA THRU
C/BORE .38-.35 DIA
DEPTH. NOTED

.195

1.814
DIA.
REF..74
DIA.1.240
DIA

14°

.12 R.

.68
BASIC

.93

1.03

1.198
(REF)

10. PENETRANT INSPECT PER MIL-I-6866

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR
 5. DIMENSION LIMITS HELD AFTER PLATING
 4. MACHINED FILLET RADII .030 - .015
 3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

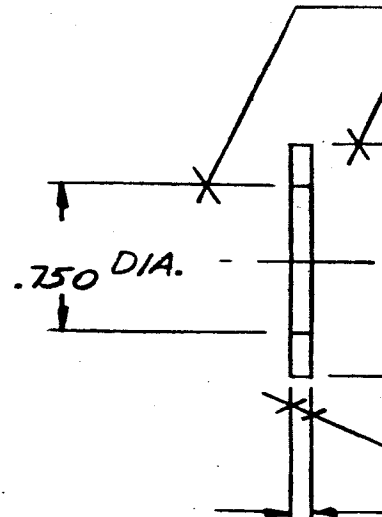
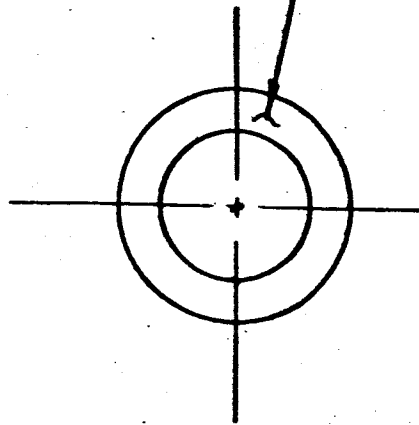
CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE IDENTITY AND RELIABILITY OF THIS SELECTED CRITICAL ITEM. PROCUREMENT OF THIS ITEM FROM THE SAFFERTY CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPL 1.212.

FIG-00 - 0000 - 0000 100 INSTRUCTIONS

100①

PART NO. HERE PER
MC-14, CL. II



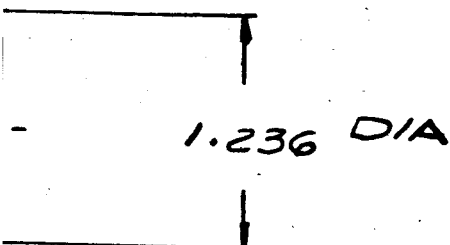
10. OPTIONAL MAT'L:
CRES 302 - MIL-S-7720

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE
 7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE
 6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR
 5. DIMENSION LIMITS HELD AFTER PLATING.
 4. MACHINED FILLET RADII .030 - .015.
 3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
 2. SURFACE ROUGHNESS PER MIL-STD-10.
 1. DIMENSIONS ARE IN INCHES.
- UNLESS OTHERWISE SPECIFIED.

QTY. REQD.		ITEM NO.	PART NO.
		← ASSYS	
1		369731	369730
REQD.	NEXT ASSY.		USED ON
HEAT TREATMENT		PROCESS	
HARDNESS		NAME	
SPEC.		SPEC.	

		REV
	SYM	DESCRIPTION

CONCENTRIC



PARALLEL

115

SYM	DESCRIPTION	CODE IDENT

LIST OF MATERIAL

SIGNATURES		DATES	AiResearch Manufacturing Co PHOENIX, ARIZO	
DET.	<i>[Signature]</i>	10-25-63		
CHK.	<i>[Signature]</i>	11-7-63	DWG. TITLE WASH	
MFG. ENG.				
MAT. & PROCESS	<i>[Signature]</i>	11-7-63		
STRESS	<i>[Signature]</i>	11-7-63		
AERO				
APP.	<i>[Signature]</i>	11-11-63	2W 11-14-63 CODE IDENT NO. 99193	
APP.				
DESIGN ACTIVITY APP.	<i>[Signature]</i>	11-11-63		
OTHER ACTIVITY APP.			SCALE FULL WT.	

VISIONS

D

DATE

APPROVED

35

DWG. NO.

369771

CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE
END PRODUCT DEPENDS ON THE IN-
TEGRITY AND RELIABILITY OF THIS
SELECTED CRITICAL ITEM. PROCURE-
MENT OF THIS ITEM FROM THE
GARRETT CORPORATION IS RECOM-
MENDED IN COMPLIANCE WITH ASPL
1.313.

CBES 303

MIL-S-7720

MATERIAL

SPECIFICATION

UNIT
WT.

Company of Arizona

A Division of

THE GARRETT CORPORATION

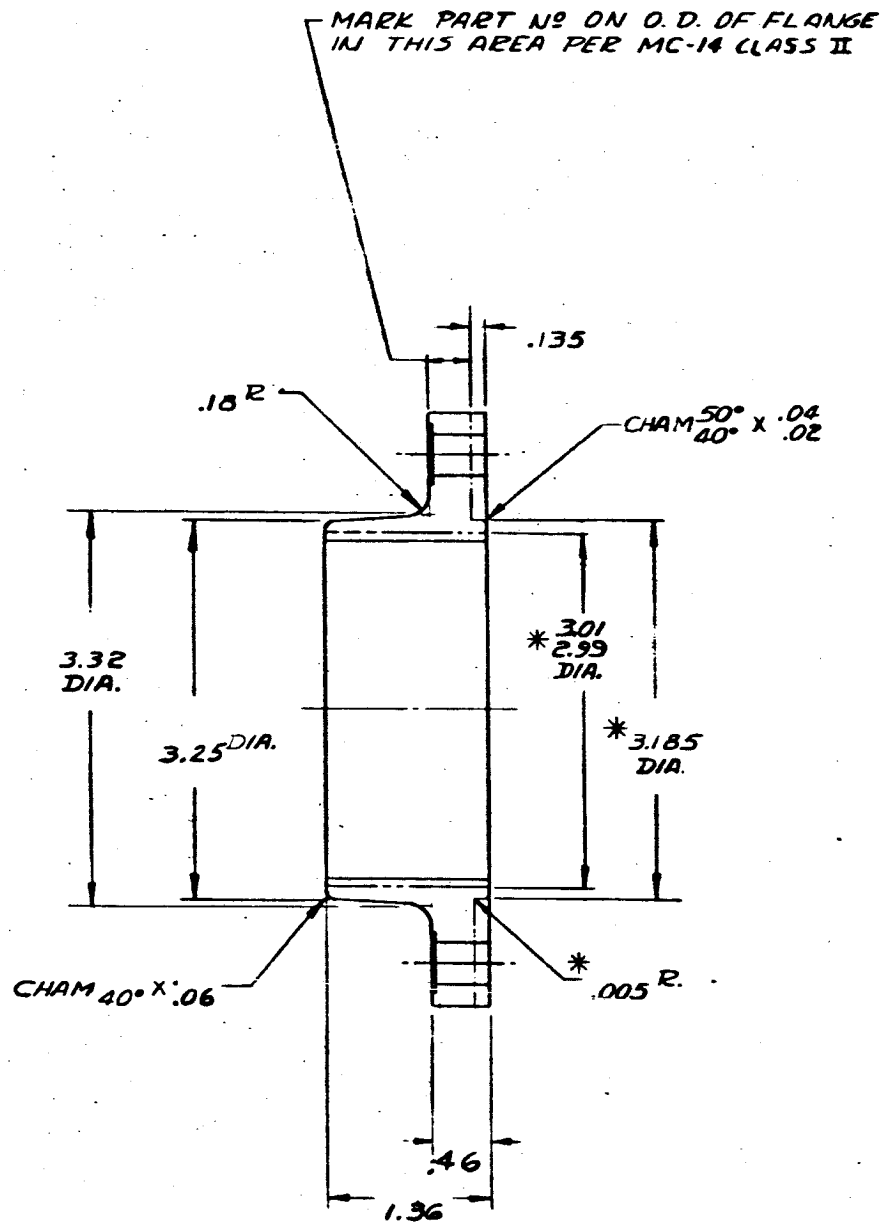
VER, FLAT

10103

DWG. NO.

369771

SHEET 1 OF 1



13. AFTER WELDING IN PLACE,
CUSTOMER TO STRESS RELIEVE
PRIOR TO FINAL MACHINING BY
HEATING TO 750-700°F. FOLLOWED
BY AIR COOL.

12. OPTIONAL MATERIAL PER MIL-S 6721

11. DIMENSIONS MARKED WITH ASTERISK
(*) TO BE MACHINED BY CUSTOMER

10. POSITIONAL & GEOMETRIC TOLERANCE
SYMBOLS PER MIL-STD-8

9. FINISH ALL OVER ✓

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.

7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.

6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC
WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITH-
IN .032 TIR.

5. DIMENSION LIMITS HELD AFTER PLATING.

4. MACHINED FILLET RADII .030 - .015

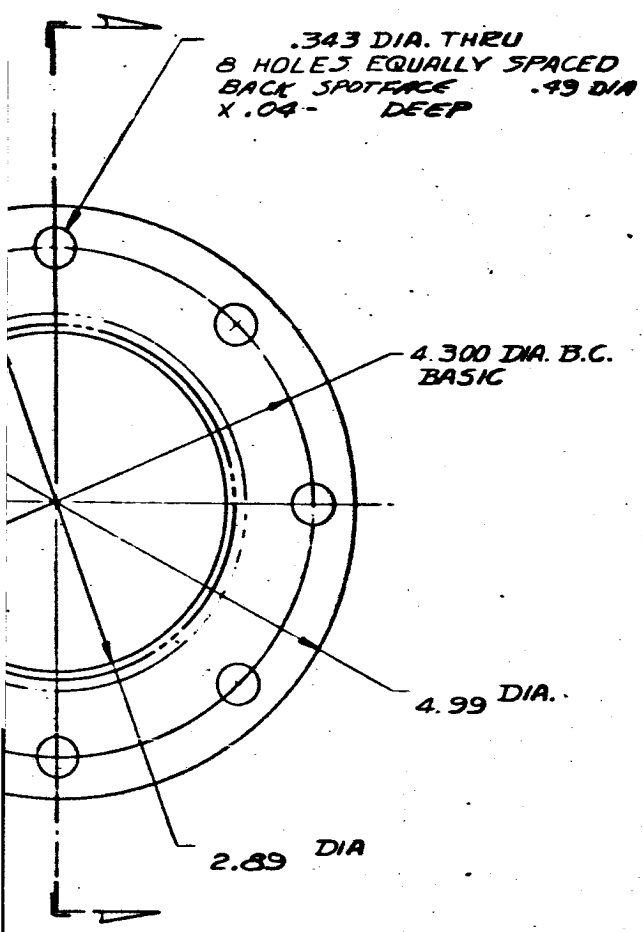
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.

2. SURFACE ROUGHNESS PER MIL-STD-10.

1. DIMENSIONS ARE IN INCHES.

UNLESS OTHERWISE SPECIFIED.

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED



CRITICAL ITEM
SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE INTEGRITY AND RELIABILITY OF THIS SELECTED CRITICAL FINAL PROCUREMENT OF THIS ITEM FROM THE GARRETT CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPL 1.313.

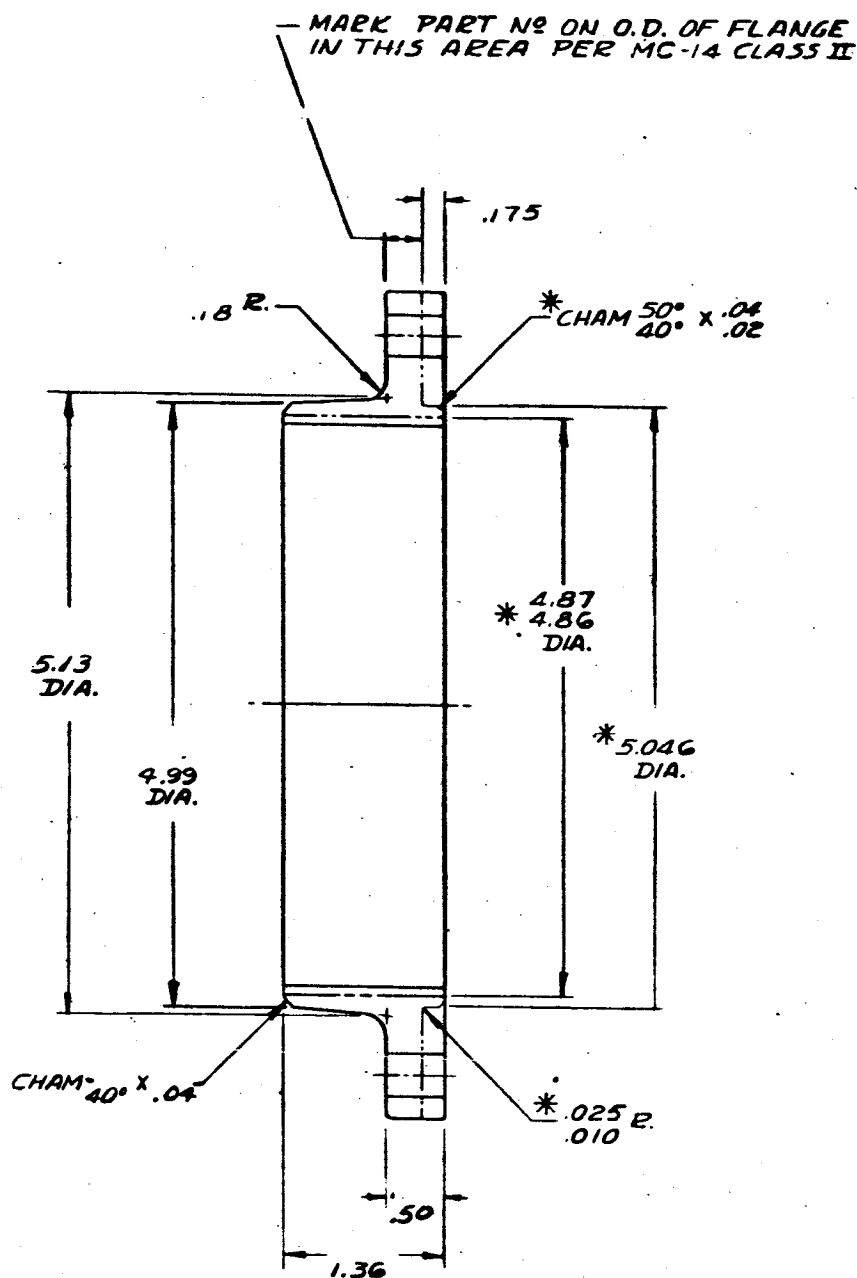
QTY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
------------	----------	----------	-----	-------------	------------	----------	---------------	----------

← ASSYS		LIST OF MATERIAL	
DFT. <i>N. Walker</i> 11-8-63 CHK. <i>Garford</i> 11-9-63 MFG. ENG. <i>Garford</i> MAT. & PROCESS <i>Garford</i> 12-2-63 STRESS AERO APP. <i>Garford</i> 11-12-63		ARResearch Manufacturing Company of Arizona PHOENIX, ARIZONA DWSL TITLE FLANGE, COMPRESSOR OUTLET	
EQD. NEXT ASSY. USED ON TREATMENT PROCESS ADDRESS NOTED SPEC. NOTED	NAME DESIGN ACTIVITY <i>Walker</i> 12-3-63 OTHER ACTIVITY APP.	CODE IDENT NO. 99193 SCALE <i>FULL</i>	SIZE C DWSL NO. 369778 WT. SHEET 1 OF 1

↑
LAST
369778
↑

102 (2)

PS-5109-R



13. AFTER WELDING IN PLACE,
CUSTOMER TO STRESS RELIEVE
PIECE TO FINAL MACHINING BY
HEATING TO 750-100°F. FOLLOWED
BY AIR COOL.

12. OPTIONAL MATERIAL PER QQ-S-763
11. DIMENSIONS MARKED WITH ASTERISK
(*) TO BE MACHINED BY CUSTOMER
10. POSITIONAL & GEOMETRIC TOLERANCE
SYMBOLS PER MIL-STD-8
9. FINISH ALL OVER

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.

7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.

6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC
WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITH-
IN .032 TIR.

5. DIMENSION LIMITS HELD AFTER PLATING.

4. MACHINED FILLET RADII .030 - .015

3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.

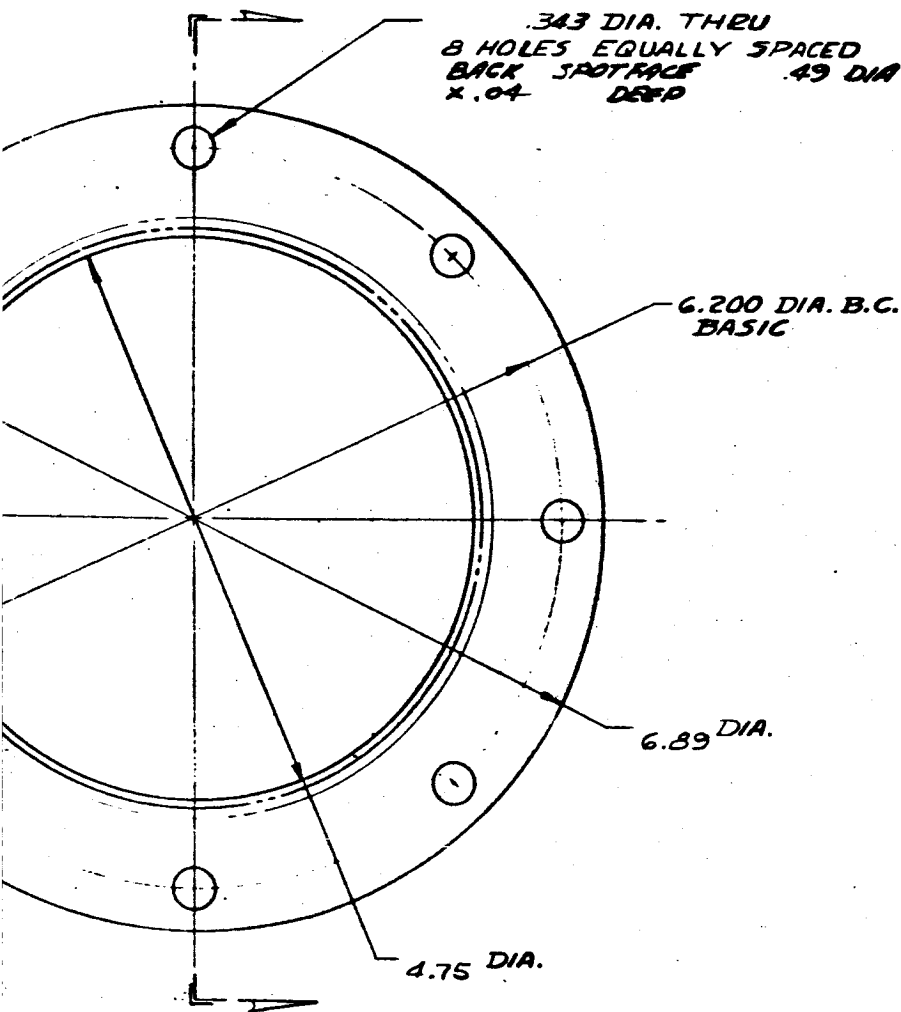
2. SURFACE ROUGHNESS PER MIL-STD-10.

1. DIMENSIONS ARE IN INCHES

UNLESS OTHERWISE SPECIFIED.

1030

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED



CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE
ENGINE PRODUCT DEPENDS ON THE IN-
TEGRITY AND RELIABILITY OF THIS
SELECTED CRITICAL ITEM. PROCURE-
MENT OF THIS ITEM FROM THE
GARRETT CORPORATION IS RECOM-
MENDED IN COMPLIANCE WITH ASPL
1.313.

TY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
						CRS	MIL 3-6721 (231 OR 347)	

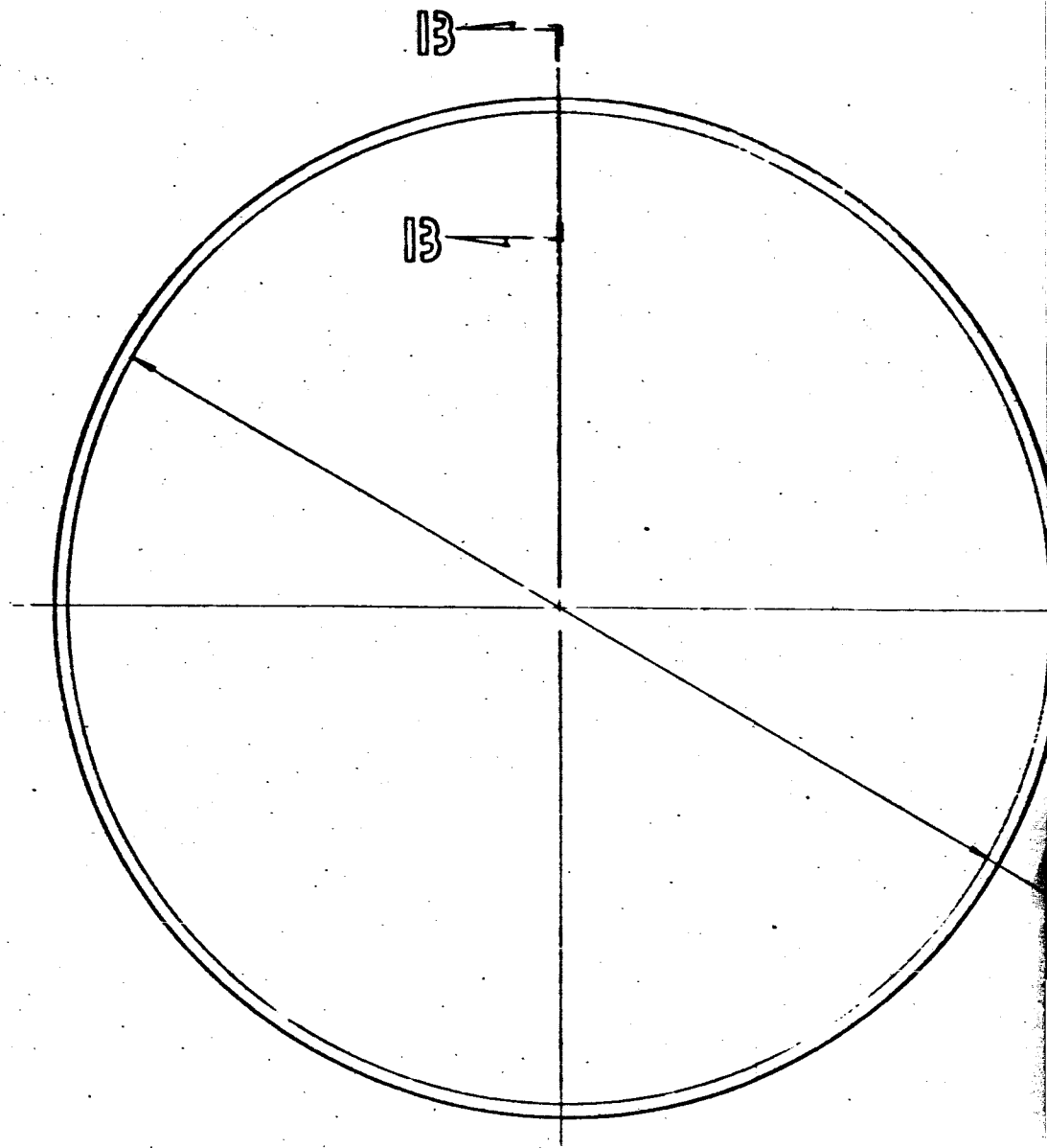
← ASSYS		LIST OF MATERIAL	
QD. NEXT ASSY. USED ON 369730		DWT. TITLE FLANGE, COMPRESSOR INLET	
AT TREATMENT NOTED		DWT. NO. 99193	
PROCESS NOTED		SIZE C	
NAME NOTED		DWT. NO. 369779	
SPEC NOTED		SCALE FULL WT.	
SIGNATURES DPT. <i>W. Weber</i> CHK. <i>Garrett</i> MFG. ENG. MAT. & PROCESS <i>Garrett</i> STRESS AIRD APP. <i>Garrett</i> DESIGN ACTIVITY APP. <i>W. Weber</i> OTHER ACTIVITY APP.		DATES 11-8-63 11-9-63 12-2-63 11-17-63 12-3-63	

AD Research Manufacturing Company of Arizona PHOENIX, ARIZONA		A GARRETT CORPORATION	
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103 (2)

369779

APS-5108-R
APS-5109-R



13. PARTS PROCURED BY VENDOR PART NO SHALL BE PROCURED IN ACCORDANCE WITH THIS AIRSEARCH SOURCE CONTROL DRAWING.
12. PROCUREMENT SOURCE(S) PER ASL 369813
11. ALL DESIGN & PART NO CHANGES MUST RECEIVE PRIOR AIRSEARCH APPROVAL.
10. IDENTIFY ALL PACKAGING WITH AIRSEARCH PART NO.
9. DATE OF MANUFACTURE MUST BE MARKED ON ALL SPARE PARTS PACKAGES.
8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A MAX. OF .006 FOR ANY SURFACE.
7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002 PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITHIN .032 TIR.
5. DIMENSION LIMITS HELD AFTER PLATING.
4. MACHINED FILLET RADII .030 - .015
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX. NO HANGING BURRS ALLOWED.
2. SURFACE ROUGHNESS PER MIL-STD-10.
1. DIMENSIONS ARE IN INCHES.

UNLESS OTHERWISE SPECIFIED.

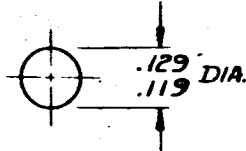
1040

REVISIONS

DESCRIPTION

DATE

APPROVED



SECTION 13 - 13
SCALE: 4/1

ALL DIAMETERS OF CROSS
SECTION MUST EQUAL
.129 - .119 DIA.

8.78 DIA.
8.12

LAST
=

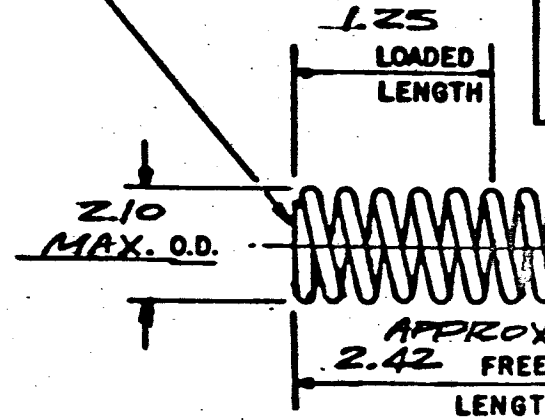
369813

TY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
						VITON "A"	MIL-E-25097 TYPE I, CLASS I	
← ASSYS								
LIST OF MATERIAL								
SIGNATURES				DATES				
DFT. <i>W. Walker</i>				11-13-63				
CHK. <i>W. Walker</i>				11-13-63				
MFG ENG								
MAT. & PRO. <i>R. Fuchs</i>				11-14-63				
STRESS								
AIRD								
APP. <i>W. Walker</i>				11-14-63				
APP								
DESIGN ACTIVITY				<i>W. Walker</i> 11-14-63				
OTHER ACTIVITY APP.								
DWS TITLE				SEAL, "O" RING				
CODE IDENT NO.				SIZE		DWS NO.		
99193				C		369813		
SCALE <i>FULL</i> UNNOTED				WT.		SHEET 1 OF 1		

104 (2)

APS-5100-10
 APS-5109-TR

SEE SPRING SPECIFICATIONS FOR
 TYPE OF ENDS REQUIRED

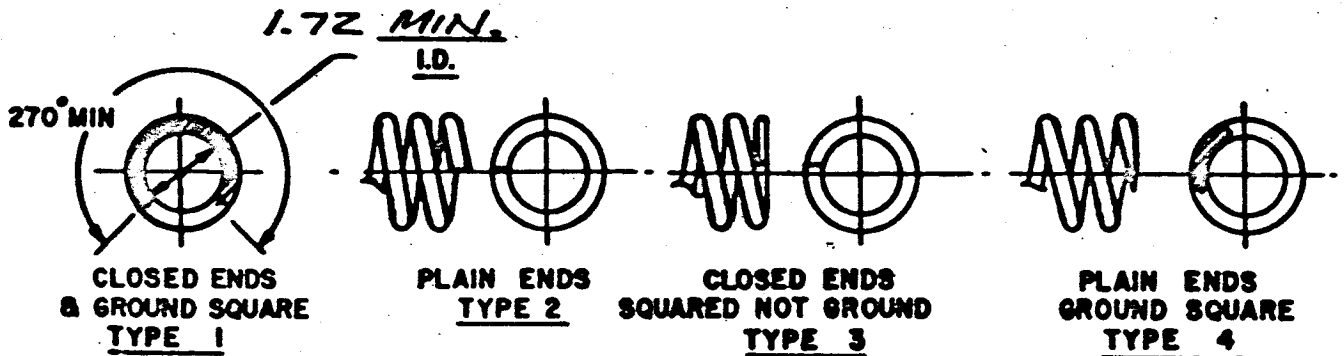


1. SPRING SHALL NOT TAKE A PERMANENT SET WHEN
 COMPRESSED TO SOLID HEIGHT.
NOTES: UNLESS OTHERWISE SPECIFIED.

			<u>NOTED</u>	
R.H.	L.H.	NO.	SIZE	
DASH NO.		REQ.		
CAL. WT.			Alpha	
WT.			71-24-	
FINISH				
HEAT TREAT				
<u>NOTED</u>				
ISSUE		DATE		

105 ①

ZONE	SYM.	REVISION	DATE	DEPTH	APP.



SPRING SPECIFICATIONS

1. .162 DIAMETER OF WIRE
 2. 35.0 LB. LB. AT LOADED LENGTH
 - (REF.) 3. 30.0 LB. LB. PER INCH (SPRING RATE) BETWEEN LENGTHS OF INCHES AND INCHES
 4. 6 TOTAL NUMBER OF COILS
 5. 4 ACTIVE COILS
 6. MAXIMUM SOLID HEIGHT
 7. LEFT HAND COILED RIGHT HAND COILED
 8. I TYPE OF ENDS
 9. SHAFT DIAMETER OVER WHICH SPRING WILL FUNCTION
 10. HOLE DIAMETER IN WHICH SPRING WILL FUNCTION
- THE ABOVE REQUIREMENTS AS CHECKED OR NOTED WILL PROVIDE DATA TO PRODUCE THIS SPRING. BLANK SPACES WILL INDICATE NO REQUIREMENTS.

225 ST. WIRE		TYPE		QQ-VI-423					
"SPRING TEMPER"		"B"		COMP. FS, (302)					
DESCRIPTION		COND.		SPECIFICATION		COND.		ZONE	
MATERIAL (COMP.)				MATERIAL SPECIFICATION					
1-25-56		1-25-56		1-25-56		1-25-56		1-25-56	
CHECKER		APP.		APP.		APP.		APP.	
UNLESS OTHERWISE SPECIFIED: 1. DIMENSION TOLERANCES ARE: DECIMAL = $\pm .010$ ANGLES = $\pm 0^\circ 30'$ 2. REMOVE BURRS AND BREAK SHARP EDGES. 3. DIMENSION LIMITS HELD AFTER PLATING. 4. R.M.S. $\sqrt{\text{SURFACE ROUGHNESS PER NAS NO. 30}}$									
AIRESEARCH MANUFACTURING COMPANY						PHOENIX, ARIZONA		SCALE	
SPRING — COMPRESSION						AREA SQ. FT.		DWS. SIZE	
						1.30		B	
								111917	
								DWS. NO.	

105 (2)

BEARING DESCRIPTION BALL, ANGULAR CONTACT			GRADE AIR RESEARCH S	
INNER RING	OUTER RING	ASSEMBLED BEARING CHARACTERISTICS		
MATERIAL: SAE 52100 STEEL	MATERIAL: SAE 52100 STEEL	TOTAL DIAMETRAL CLEARANCE OF (DIAMETRAL AREA) .0019 UNDER 2 LBS GAGE LOAD		
BORE: .9841 TAPER/FT:	OD: 1.8504 TAPER/FT:	CONTACT ANGLE REF: 20°		
WIDTH: .4624	FLANGE OD:	MAXIMUM TORQUE: (LBS IN. IN. LBS) AT RPM		
RACE DEPTH REF: 20 MIN %BALL DIA.	WIDTH: .4624	UNDER: (GRAND, LBS) (THRUST, RADIAL) LOAD		
RACE CURVATURE REF: 52-53 %BALL DIA.	FLANGE WIDTH:	STABILIZED FOR OPERATION AT: SEE NOTES		
SEPARATOR PILOT LAND TO GROOVE RUNOUT:	RACE DEPTH REF: 16 MIN %BALL DIA.	(INNER, OUTER) RING FACE OFFSET: UNDER: LBS THRUST LOAD.		
SEPARATOR	RACE CURVATURE REF: 54-55 %BALL DIA.	SPECIAL FEATURES		
FORGED SILICON IRON BRONZE MATERIALS: PER AMS 4616	SEPARATOR PILOT LAND TO GROOVE RUNOUT:	1. PARTS MUST NOT CHANGE IN DIMENSION IN		
CONSTRUCTION: MACHINED	ROLLING ELEMENTS	2. FACE "A" TO BE FLUSH WITH 30 LBS THRUST		
ASSEMBLY: ONE PIECE	MATERIAL: SAE 52100/51100 STEEL	LOAD APPLIED IN DIRECTION		
PILOTING SURFACE: OUTER RING	COMPLEMENT PER ROW: 13-1/4	3. SEPARATOR PACKET SURFACE		
PILOT CLEARANCE: .008	CLOSURES	CIRCUIMFERENCE TYPALLY FORMS		
OPERATIONAL LUBRICANT	NUMBER: NONE	ACT TO BE AT BEARING P.O.		
OIL: SYNTHETIC AIRCRAFT GAS	TYPE: (INNER, BEARING)	4. SEPARATOR TO BE SILVER PLATED		
NAME: TURBINE LUBRICATING	MATERIAL:	.0006-.0020 THICK PER AMS 2412		
MILITARY SPEC NO. MIL-L-7608	CONSTRUCTION:			
BEARING PRELUBRICATION: DIPS DRAIN				
PACKAGING PER AIR SPEC CP-14	PRODUCTION BULK PACK	COMMERCIAL SPARES PACK	MILITARY SPARES PACK	
PRESERVATIVE:	MIL-L-6085	MIL-L-6085	MIL-R-187	
AIR RESEARCH PART NUMBER:	BASIC	BASIC	BASIC-4	
VENDOR NAME & LOCATION	VENDOR PART AND SPECIFICATION NUMBER			
PER SUPPLY CODE NO.	PROCUREMENT			
	PER ASL			

050

358313

DWG. NO.

LAST

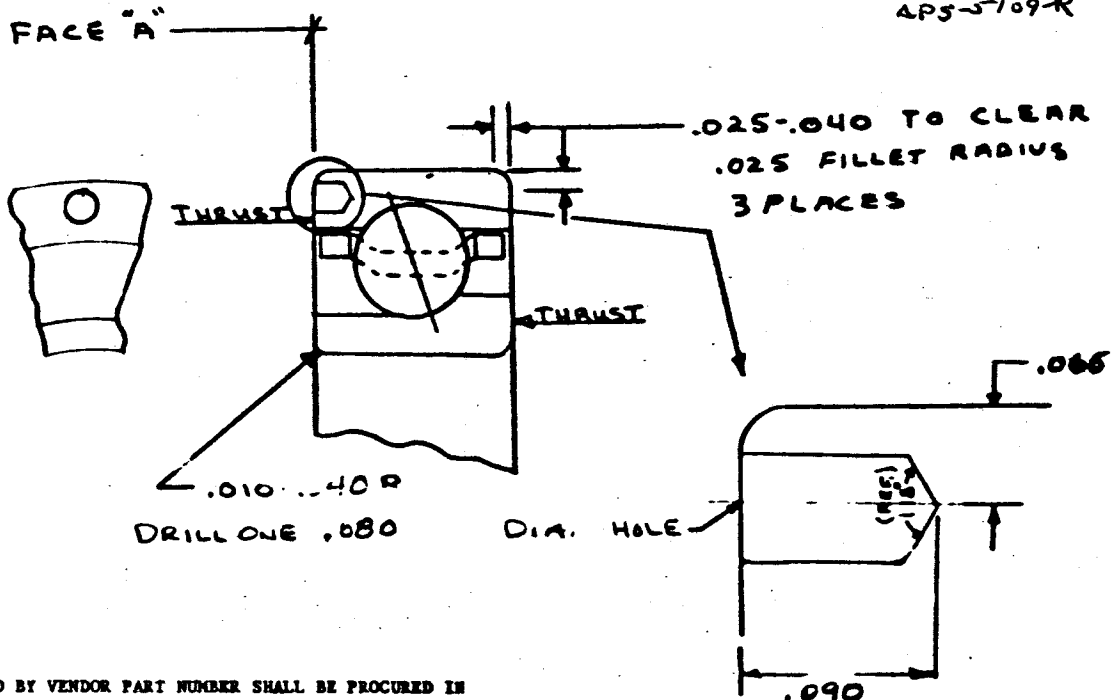
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Form 1552B

REVISIONS

SYM	DESCRIPTION	DATE	APPROVAL

AP3-5706-R
AP3-5709-R



7. PARTS PROCURED BY VENDOR PART NUMBER SHALL BE PROCURED IN ACCORDANCE WITH THIS AIRESEARCH SOURCE CONTROL DRAWING.
6. IDENTIFY PACKAGING WITH AIRESEARCH PART NUMBER.
5. ALL DESIGN AND PART NUMBER CHANGES MUST RECEIVE PRIOR AIRESEARCH APPROVAL.
4. ONLY THE ITEMS LISTED ON THIS DRAWING AND IDENTIFIED BY VENDORS' NAMES, ADDRESSES AND PART NUMBERS HAVE BEEN TESTED AND APPROVED FOR USE IN THE END UNIT. A SUBSTITUTE ITEM SHALL NOT BE USED WITHOUT PRIOR TESTING AND APPROVAL BY AIRESEARCH.
3. MILITARY SPARES PACK BEARINGS ARE INTENDED TO FILL MILITARY SPARES ORDERS. BEFORE INSTALLATION WASH OUT THE PRESERVATIVE AND REPLACE WITH OPERATING LUBRICANT. AFTER THIS OPERATION THE -4 IDENTIFICATION IS DROPPED AND THE BEARINGS BECOME INTERCHANGEABLE WITH THE PRODUCTION BULK AND COMMERCIAL SPARES PACK BEARINGS. THEY SHOULD NOT BE USED IN FACTORY INSTALLATIONS BECAUSE OF THEIR RELATIVELY HIGH COST.
2. FOR ECONOMY, PRODUCTION BULK PACK BEARINGS ARE PREFERRED FOR ALL FACTORY INSTALLATIONS. COMMERCIAL SPARES PACK BEARINGS ARE INTENDED TO FILL COMMERCIAL SPARES ORDERS.
1. PRODUCTION BULK AND COMMERCIAL SPARES PACK BEARINGS ARE INTERCHANGEABLE.

CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE END PRODUCT DEPENDS ON THE INTEGRITY AND RELIABILITY OF THIS SELECTED CRITICAL ITEM. PROCUREMENT OF THIS ITEM FROM THE GARRETT CORPORATION IS RECOMMENDED IN COMPLIANCE WITH ASPR 1.313.

SOURCE CONTROL DRAWING

SIGNATURES	DATES
DRF	
Johnson	8-15-63
CHK	
Carlson	8-15-63
APP	
Plummer	8-16-63
APP	
Wahl	8-17-62
APP	
APP	

NAME

BEARING BALL, SINGLE ROW ANGULAR CONTACT

HW 8-21-63

SCALE NONE

WGT -

NEXT ASSY

USED ON

Airesearch Manufacturing Company of Arizona

PHOENIX, ARIZONA

Division of THE GARRETT CORPORATION

DWG.

B

SIZE

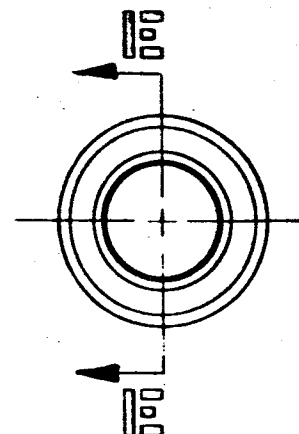
DWG NO.

358313

Code: 99193

106 (2)

APS-5708-11
APS-5709-R



21 OPERATING CONDITIONS -

PRESSURE, AIR SIDE, 0 PSIG.
OIL SIDE, - 2 PSIG.
TEMPERATURE, AIR SIDE, 100°F.
OIL SIDE 225°F.
LEAKAGE, AIR TO OIL, .01 LB/MIN
OIL TO AIR, 1 CC/HR.
OIL MIL-L-7808
LIFE REQUIRED 500 HOURS.

20. SEAL CASE TO BE CRES.

19. POSITIONAL & GEOMETRIC TOLERANCE
SYMBOLS PER MIL-STD-B.

CRITICAL ITEM

SATISFACTORY PERFORMANCE OF THE END PRODUCT
DEPENDS ON THE INTEGRITY AND RELIABILITY OF
THIS SELECTED CRITICAL ITEM. PROCUREMENT
OF THIS ITEM FROM THE GARRETT CORPORATION IS
RECOMMENDED IN COMPLIANCE WITH ASB 1.313.

1-5007

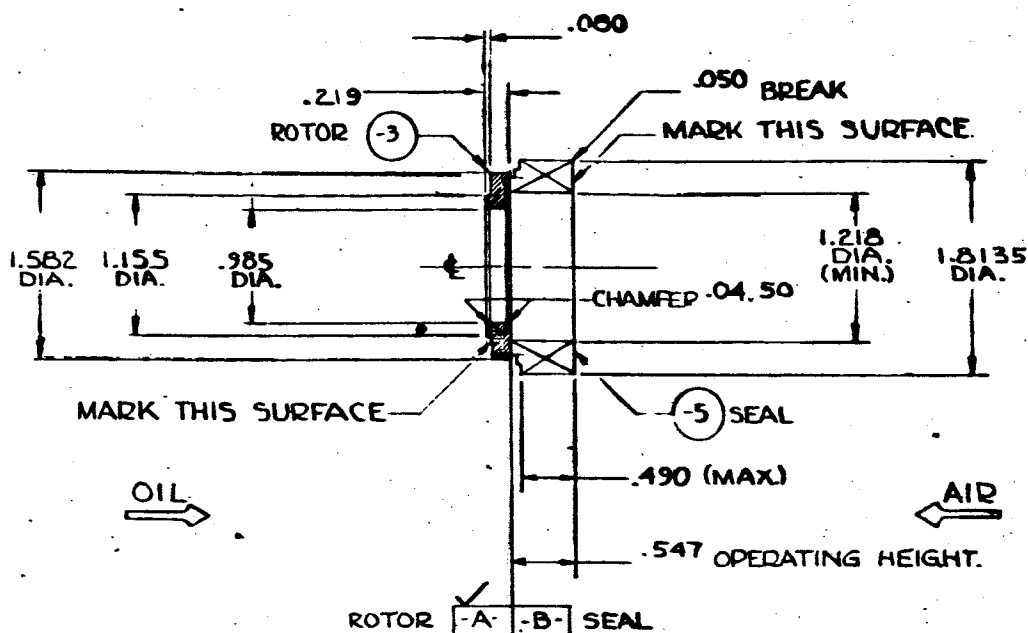
FORM 40 - 6700 - 11 6 100 MERCURY

15. PROCUREMENT SOURCE(S) PER ASL
MARKED PER
14. PART TO REMC-14 CLASS II WITH AIRSEARCH
NUMBER 352919.
13. ALL DESIGN AND PART NUMBER CHANGES REQUIRE
PRIOR AIRSEARCH APPROVAL.
12. ONLY ITEMS LISTED ON THE APPLICABLE ASL AND
IDENTIFIED BY VENDOR'S NAMES, ADDRESSES, AND
PART NUMBERS HAVE BEEN TESTED AND APPROVED FOR
USE IN THE END UNIT. A SUBSTITUTE ITEM SHALL
NOT BE USED WITHOUT PRIOR TESTING AND APPROVAL
BY AIRSEARCH.
11. IDENTIFY PACKAGING WITH AIRSEARCH NUMBER.
10. PARTS PROCURED BY VENDOR PART NUMBER SHALL BE
PROCURED IN ACCORDANCE WITH THIS AIRSEARCH
SOURCE CONTROL DRAWING.
9. DETAILS OF DESIGN AND CONSTRUCTION OTHER THAN
SHOWN SHALL BE AT OPTION OF VENDOR.
8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.
7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC
WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITH-
IN .032 TIR.
5. DIMENSION LIMITS HELD AFTER PLATING.
4. MACHINED FILLET RADII .030 - .015
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.
2. SURFACE ROUGHNESS PER MIL-STD-10.
1. DIMENSIONS ARE IN INCHES.
UNLESS OTHERWISE SPECIFIED.

1070

1

REVISIONS			
SYM	DESCRIPTION	DATE	APPROVED



SECTION

358319

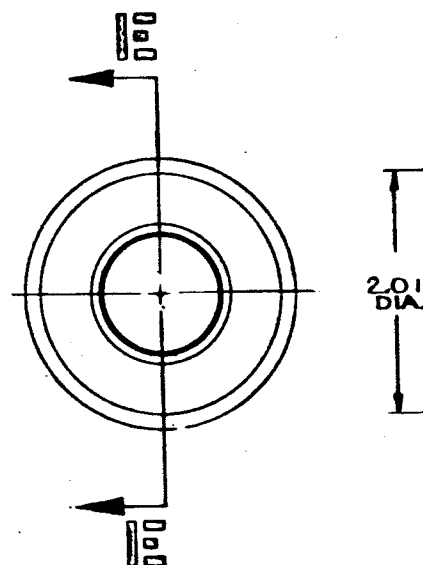
QTY. REQD.	ITEM NO.	PART NO.	SYM	DESCRIPTION	CODE IDENT	MATERIAL	SPECIFICATION	UNIT WT.
1	369741	369740	-3			CRES	60-5-V63 440 F	
<div style="display: flex; justify-content: space-between;"> <div> <p>← ASSYS</p> <p>HEAT TREATMENT</p> <p>HARDNESS</p> </div> <div> <p>LIST OF MATERIAL</p> <p>SIGNATURES</p> <p>DATES</p> <p>DESIGN AUTHORITY APP.</p> <p>OTHER ACTIVITY APP.</p> </div> <div> <p>ADDITIONAL MANUFACTURING COMPANY OF ACTIONS</p> <p>PROCESS, FINISHES</p> <p>APPROVED</p> <p>SEAL, AIR-OIL</p> <p>99193</p> <p>SCALE FULL</p> </div> <div> <p>SEAL, AIR-OIL</p> <p>99193</p> <p>WT.</p> <p>SHEET 1 OF 1</p> </div> </div>								

107 ②

PART NO	SEAL	ROTOR
358320	-5	-3

AP3-5108-R
AP3-5109-R

OPERATING CONDITIONS	SCHEDULE V
SPEED, RPM	38,500
PRESSURE, PSIG	
GAS SIDE	-1.5
OIL SIDE	-2
TEMPERATURE, °F	
GAS SIDE	800
OIL SIDE	225
LEAKAGE (MAX)	
GAS TO OIL, LBS/MIN	.01
OIL TO GAS CC/HR.	ZERO
LIFE REQ'D, HRS.	500



21. CHROME PLATE SURFACE-A PER
QQ-C-320 CLASS 2, .002-.004 THICK.
20. SURFACE-B- LAPPED FLAT

19

18. POSITIONAL & GEOMETRIC TOLERANCE
SYMBOLS PER MIL-STD-8.

17

16. (C) Designate Critical Characteristics
(M) Designate Major Characteristics

CRITICAL ITEM

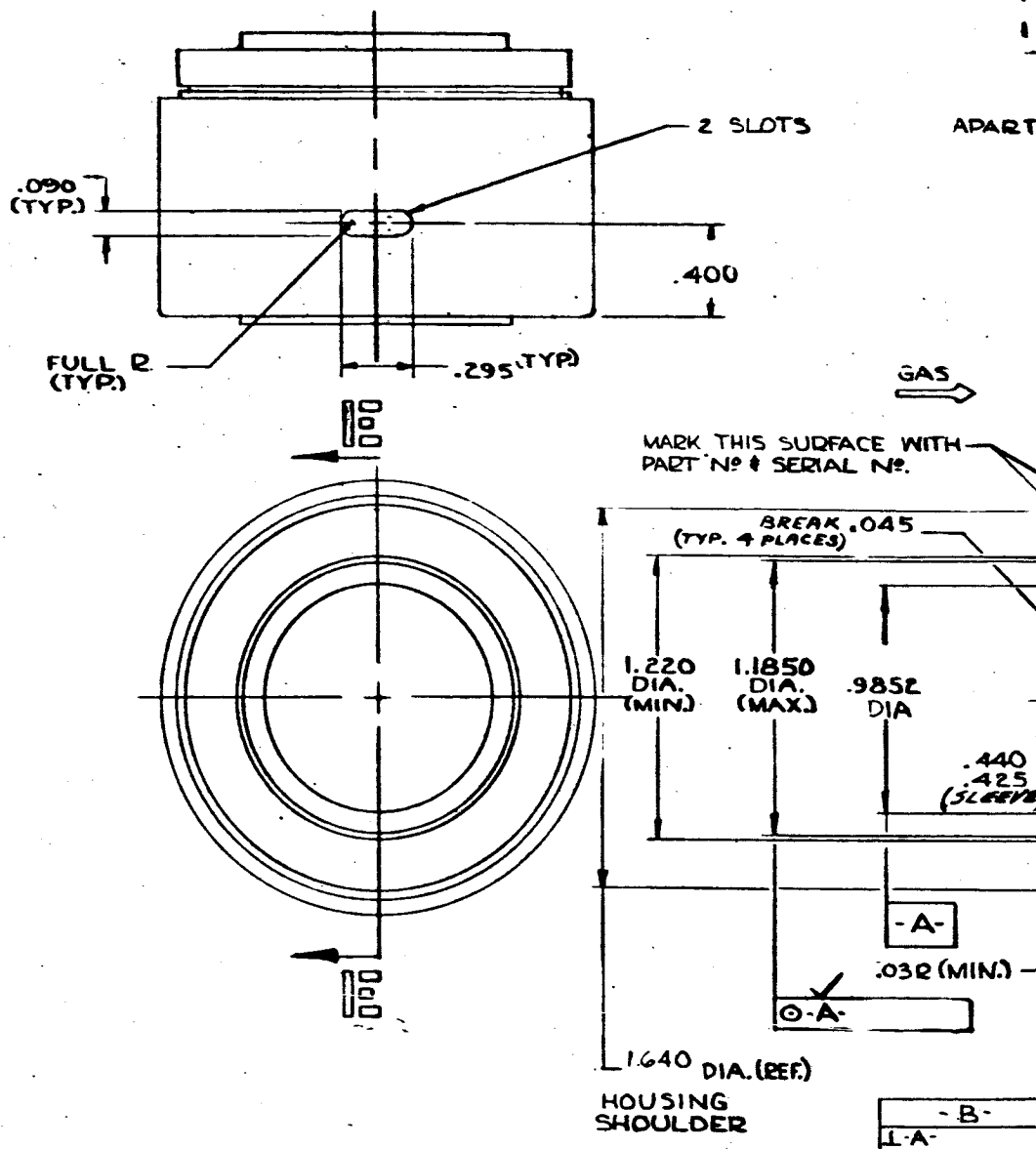
SATISFACTORY PERFORMANCE OF THE END PRODUCT
DEPENDS ON THE INTEGRITY AND RELIABILITY OF
THIS SELECTED CRITICAL ITEM. PROCUREMENT
OF THIS ITEM FROM THE GARRETT CORPORATION IS
RECOMMENDED IN COMPLIANCE WITH ASPE 1.312.

9-0000

15. PROCUREMENT SOURCE(S) PER ASL
MARKED PER
14. PART TO BE MC-14 CLASS II WITH AIRSEARCH
NUMBER 358320-5 & 5.0R-7
13. ALL DESIGN AND PART NUMBER CHANGES REQUIRE
PRIOR AIRSEARCH APPROVAL.
12. ONLY ITEMS LISTED ON THE APPLICABLE ASL AND
IDENTIFIED BY VENDOR'S NAMES, ADDRESSES, AND
PART NUMBERS HAVE BEEN TESTED AND APPROVED FOR
USE IN THE END UNIT. A SUBSTITUTE ITEM SHALL
NOT BE USED WITHOUT PRIOR TESTING AND APPROVAL
BY AIRSEARCH.
11. IDENTIFY PACKAGING WITH AIRSEARCH NUMBER.
10. PARTS PROCURED BY VENDOR PART NUMBER SHALL BE
PROCURED IN ACCORDANCE WITH THIS AIRSEARCH
SOURCE CONTROL DRAWING.
9. DETAILS OF DESIGN AND CONSTRUCTION OTHER THAN
SHOWN SHALL BE AT OPTION OF VENDOR.
8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.
7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.
6. MACHINED DIAS ON A COMMON CENTERLINE CONCENTRIC
WITHIN .002 TIR, UNMACHINED DIAS CONCENTRIC WITHIN
.002 TIR.
5. DIMENSION LIMITS FIELD AFTER PLATING.
4. MACHINED FILLET RADII .030 - .015
3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.
2. SURFACE ROUGHNESS PER MIL-STD-10.
1. DIMENSIONS ARE IN INCHES
UNLESS OTHERWISE SPECIFIED.

109 (2)

APS-5108-R
APS-5109-R



16. PARTS TO BE MADE, INSPECTED, STOCKED,
& ISSUED AS MATCHED SETS ONLY.

15. SERIAL NUMBER CONTROL MUST BE MAINTAINED ON THIS PART
THROUGHOUT MANUFACTURING AND ASSEMBLY CYCLES AND
AFTER FINISH MACHINING. SERIAL NUMBERS SHALL BE APPLIED
BY THE METHOD AND AT THE LOCATION SPECIFIED.

14. ~~PROCESSED~~ SOURCE (S) PER ASL
MARKED PER

13. PART TO BE MC-M CLASS II WITH AIRSEARCH
NUMBER 350321 & SERIAL NO.

12. ALL DESIGN AND PART NUMBER CHANGES REQUIRE
PRIOR AIRSEARCH APPROVAL.

11. ONLY ITEMS LISTED ON THE APPLICABLE ASL AND
IDENTIFIED BY VENDOR'S NAMES, ADDRESSES, AND
PART NUMBERS HAVE BEEN TESTED AND APPROVED FOR
USE IN THE END UNIT. A SUBSTITUTE ITEM SHALL
NOT BE USED WITHOUT PRIOR TESTING AND APPROVAL
BY AIRSEARCH.

10. IDENTIFY PACKAGING WITH AIRSEARCH NUMBER.

9. PARTS PROCURED BY VENDOR PART NUMBER SHALL BE
PROCESSED IN ACCORDANCE WITH THIS AIRSEARCH
SOURCE CONTROL DRAWING.

8. MACHINED SURFACES FLAT WITHIN .0005 PER INCH TO A
MAX. OF .006 FOR ANY SURFACE.

7. MACHINED SURFACES NORMAL OR PARALLEL WITHIN .002
PER INCH TO A MAX. OF .012 FOR ANY SURFACE.

6. MACHINED DIAS. ON A COMMON CENTERLINE CONCENTRIC
WITHIN .005 TIR, UNMACHINED DIAS. CONCENTRIC WITH-
IN .032 TIR.

5. DIMENSION LIMITS HELD AFTER PLATING.

4. MACHINED FILLET RADII .030 - .015

3. BREAK ALL CORNERS AND SHARP EDGES .015 MAX.
NO HANGING BURRS ALLOWED.

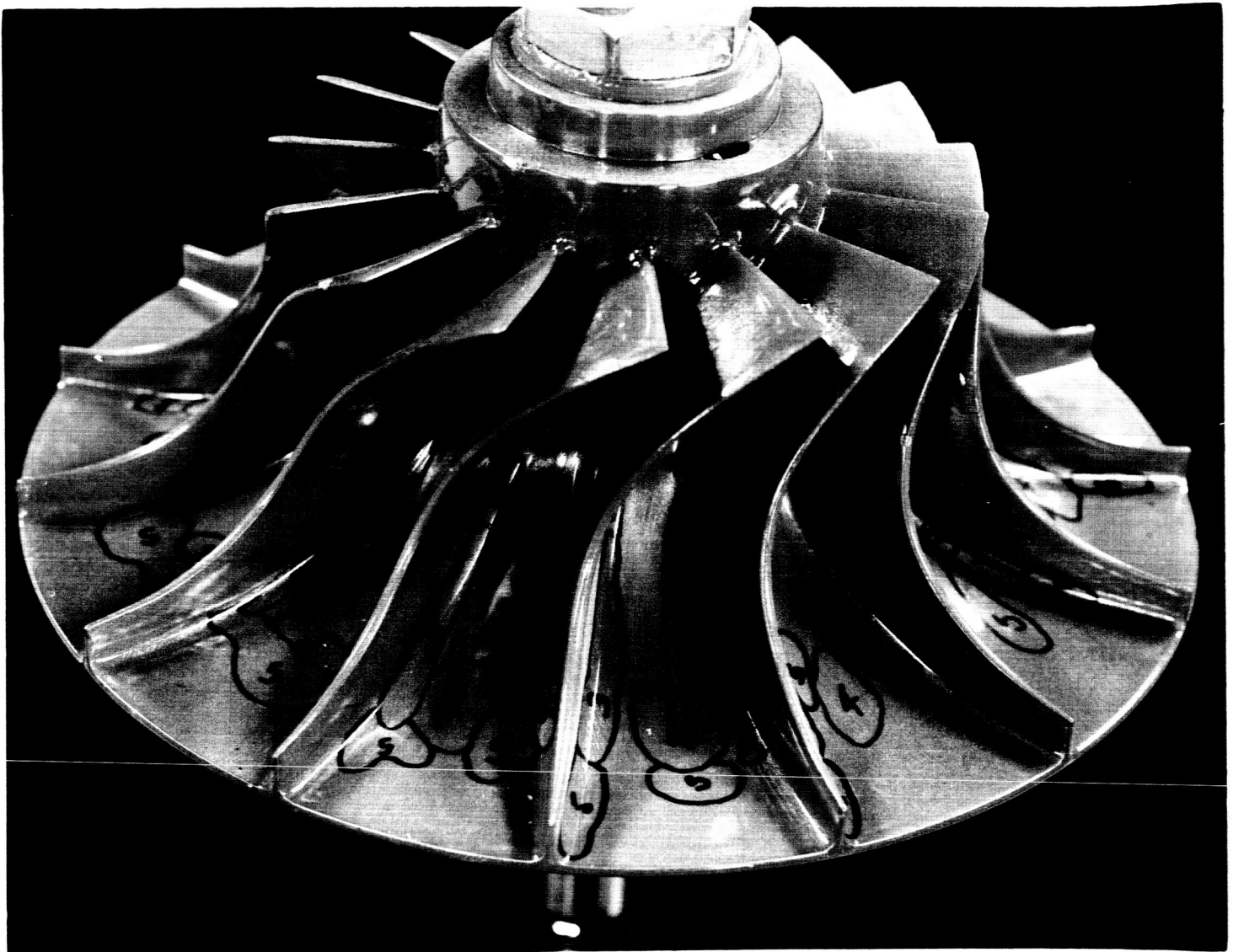
2. SURFACE ROUGHNESS PER MIL-STD-10.
DIMENSIONS ARE IN INCHES.

1. UNLESS OTHERWISE SPECIFIED.

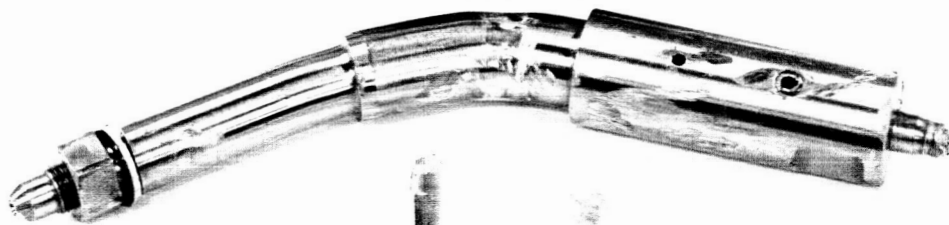
109 (2)



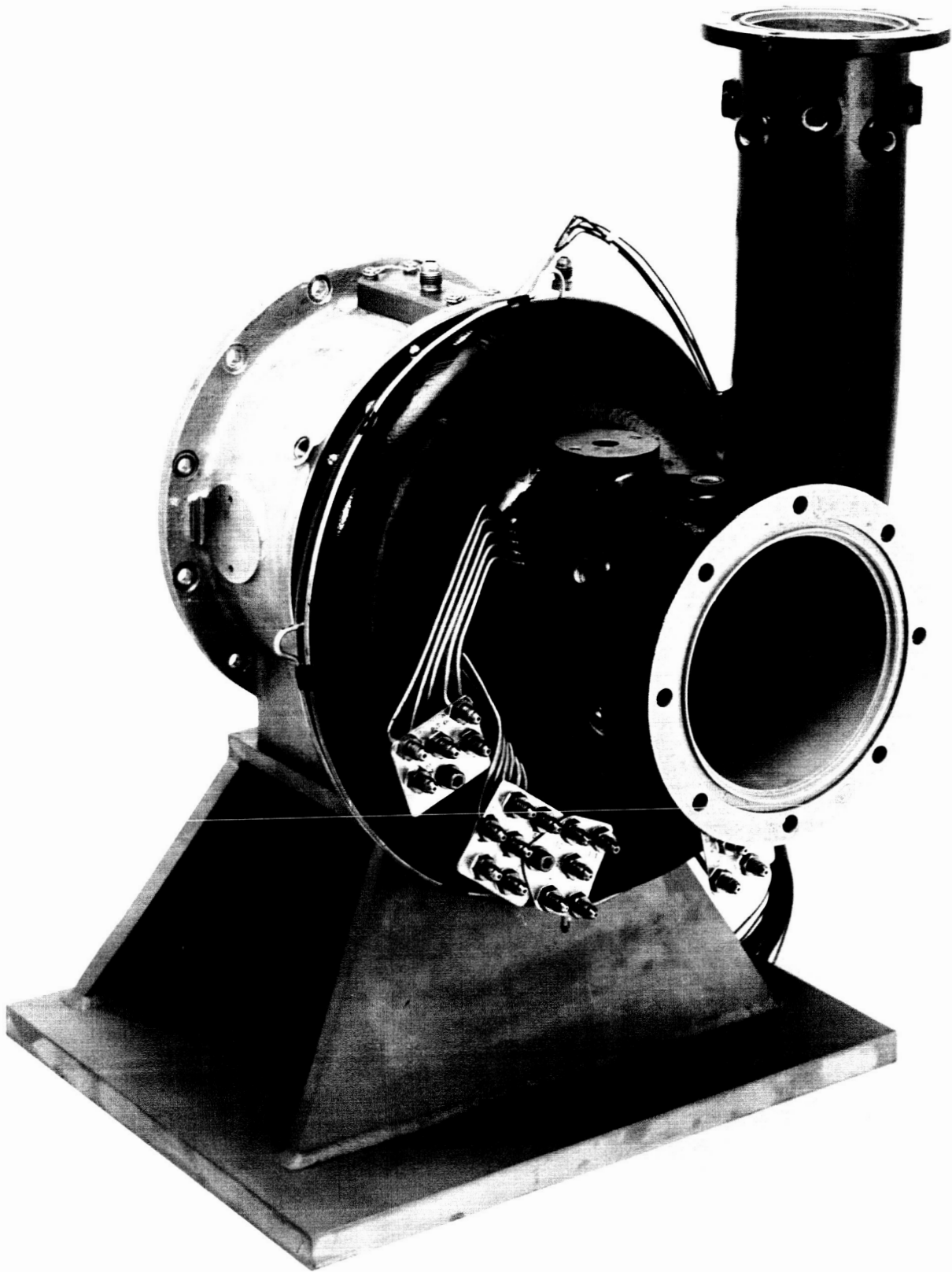
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



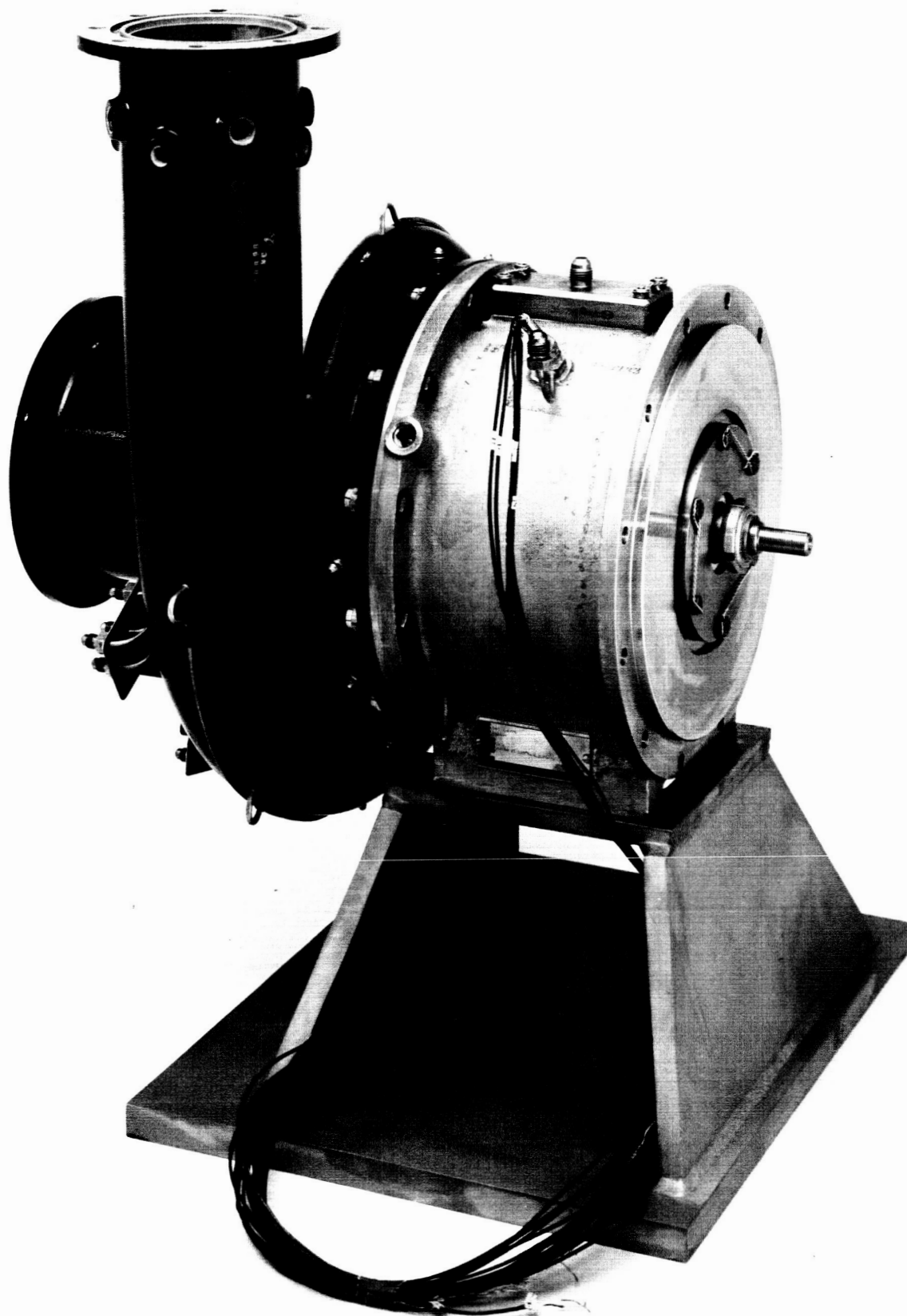
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



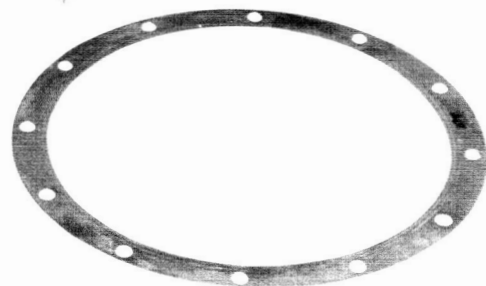
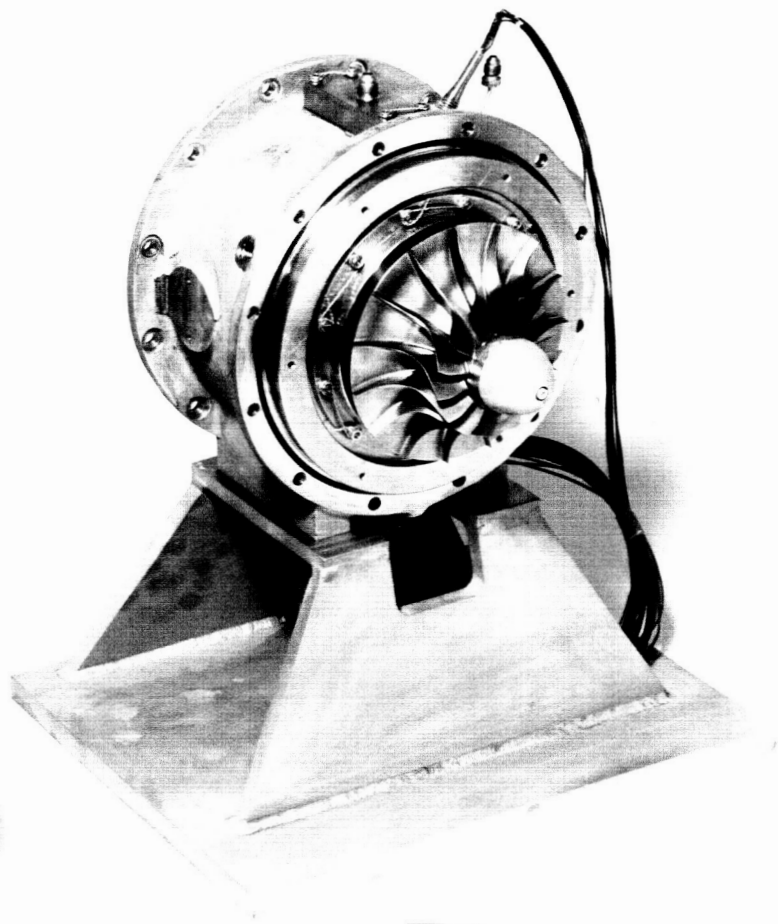
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



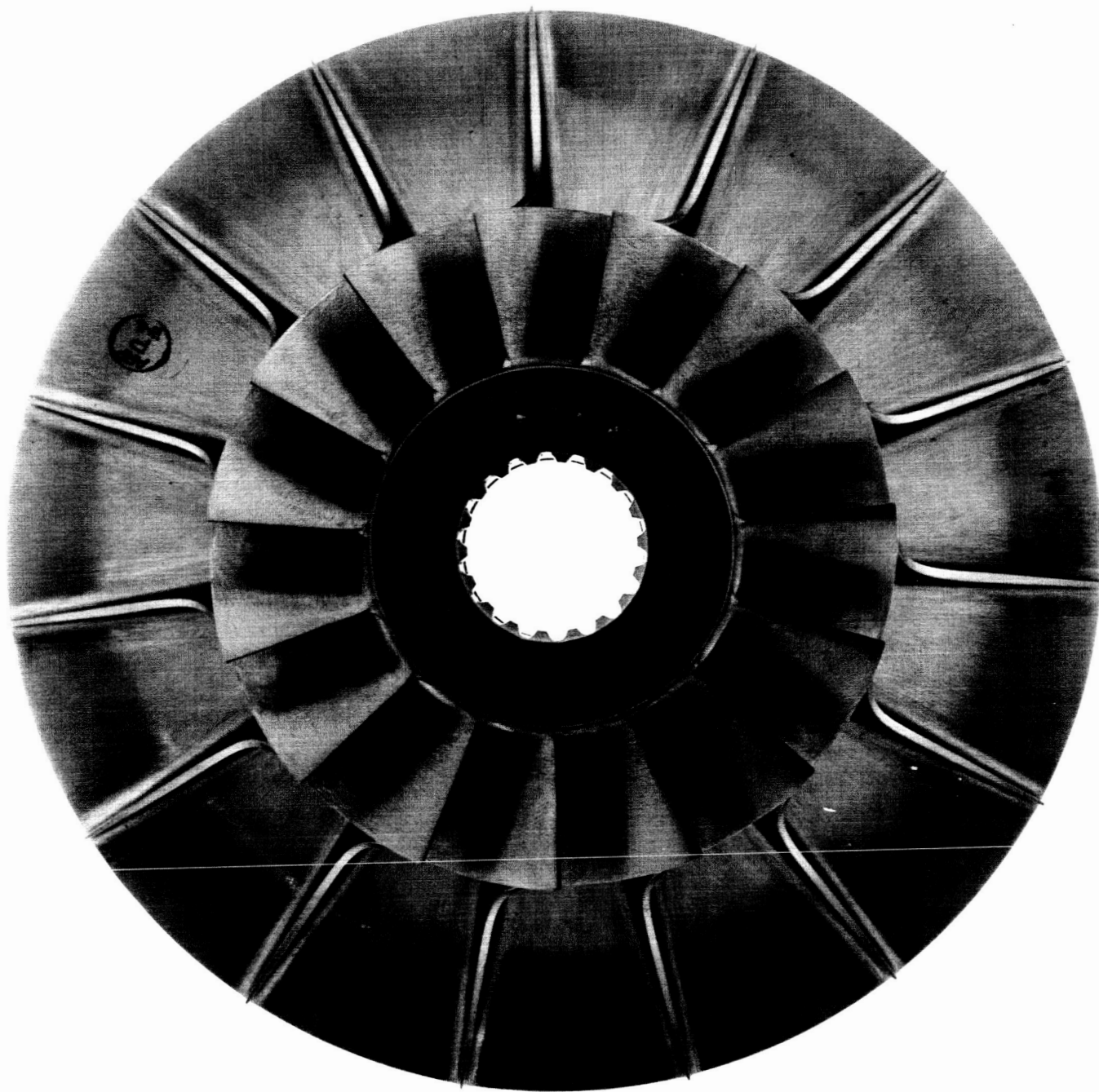
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



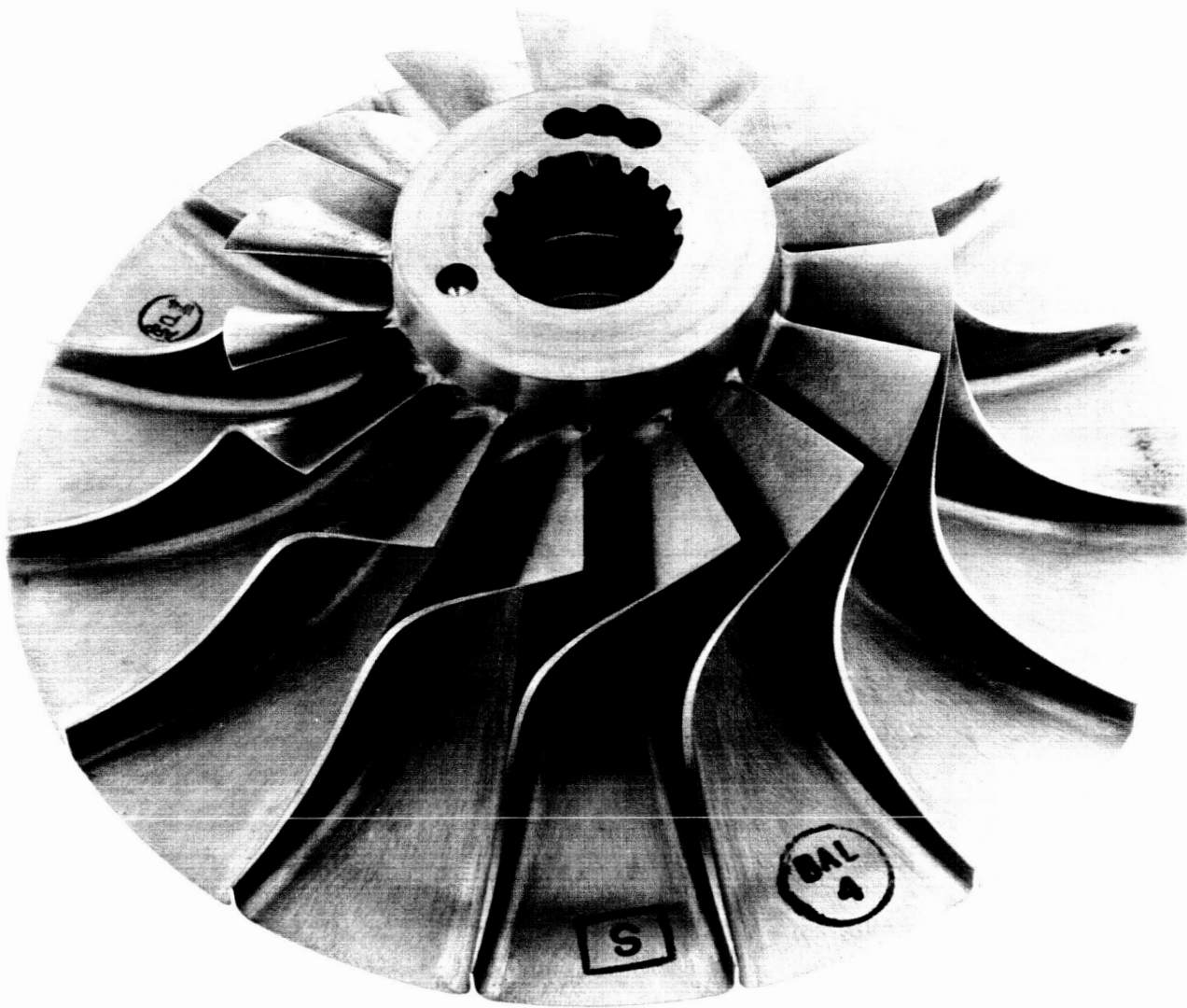
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



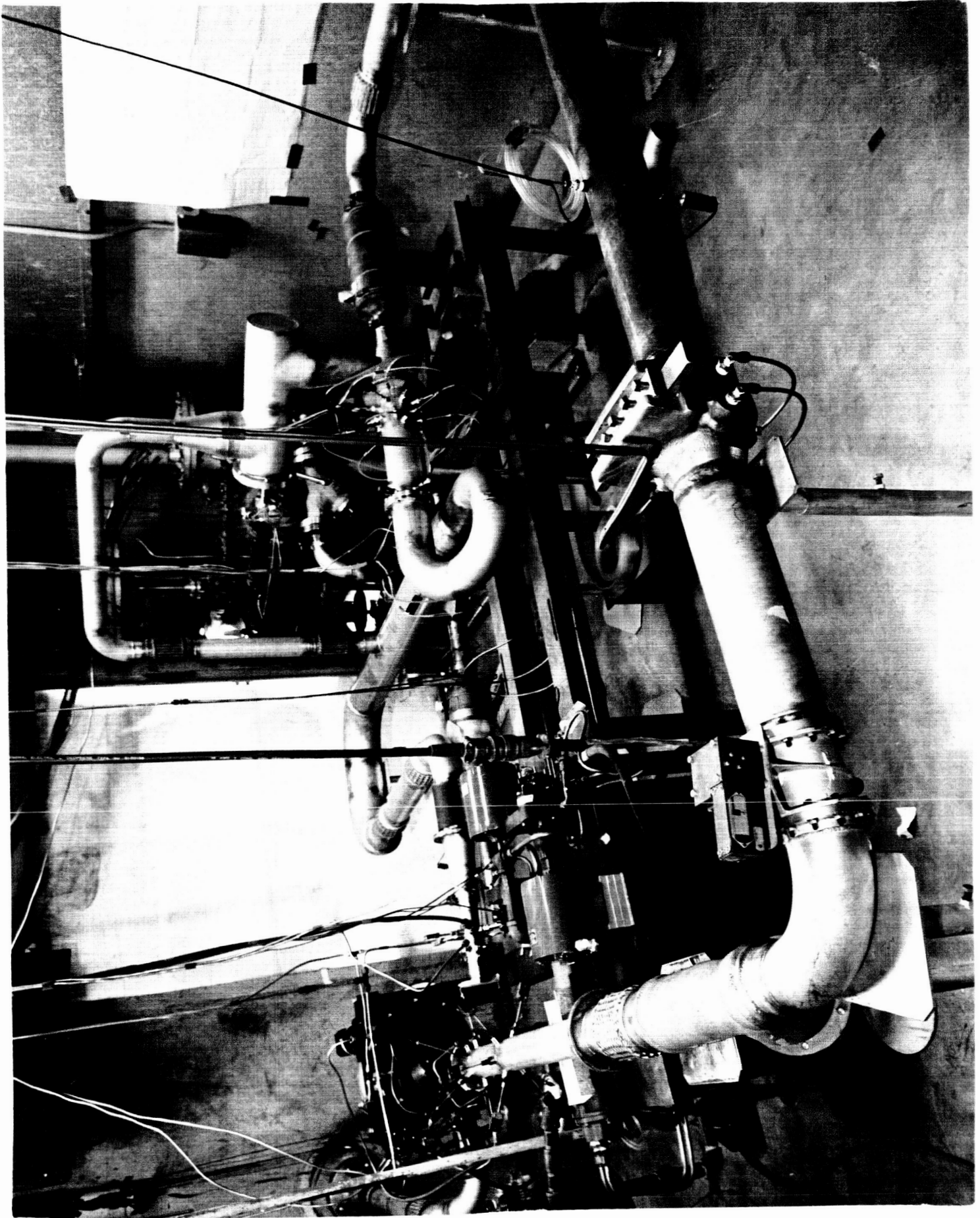
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



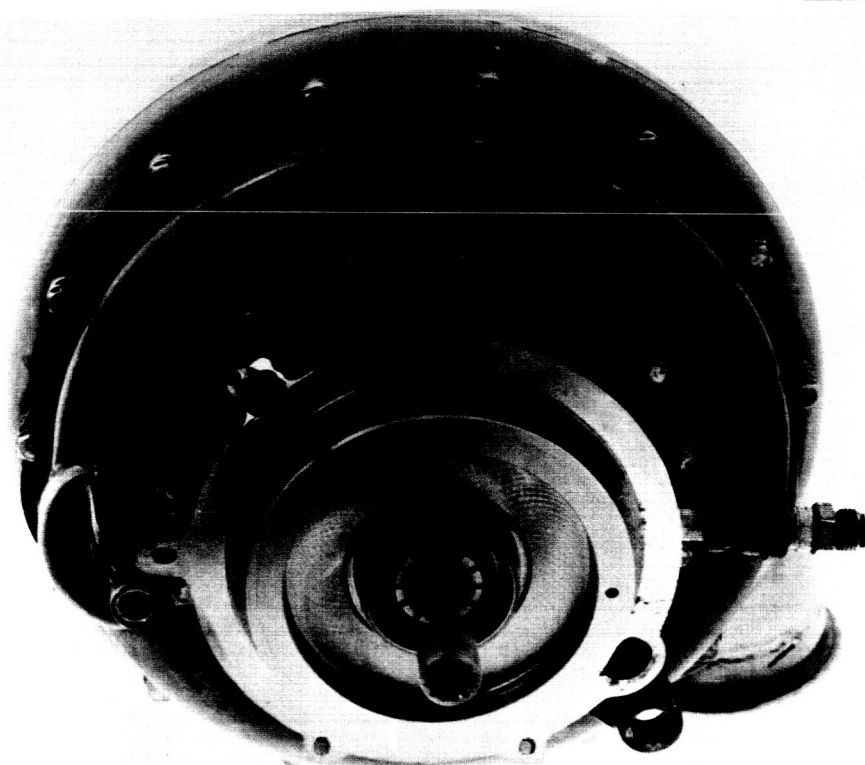
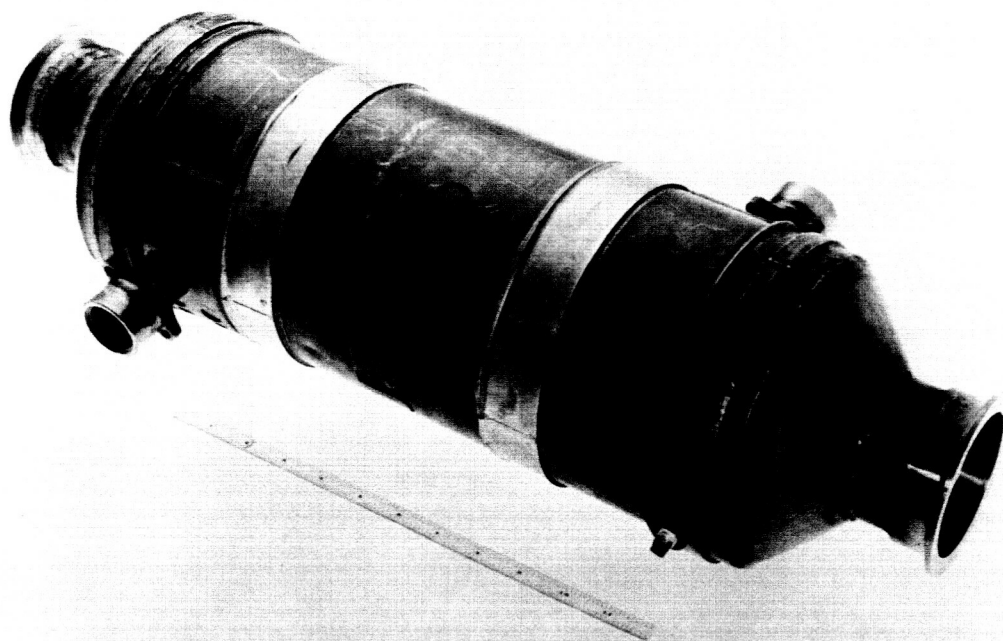
AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



AIRESEARCH MANUFACTURING COMPANY
A DIVISION OF THE GARRETT CORPORATION
PHOENIX, ARIZONA



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PHOENIX, ARIZONA



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PHOENIX, ARIZONA